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半導體材料與製程產業研發碩士專班
碩士論文

底膠材料熱機械性質對低介電覆晶球狀陣列封裝體之
熱變形行為研究

The Effect of Underfill Materials' Thermo-mechanical
Properties on The Thermal Deformation Behavior of
Low-K Flip-Chip Ball Grid Array Packages



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摘要

隨著半導體製造技術的發展，積體電路公司引進低介電材料與銅製程用以減輕電阻電容延遲效應。但是，低介電材料較之傳統的二氧化矽介電質有著較差的材料機械性質與附著性。這將使得低介電材料層發生破裂的風險大為增加。此外，由於環保意識的抬頭，無鉛焊料已逐漸取代傳統的錫鉛合金。然而，無鉛焊料需要更高的迴流溫度 (Reflow temperature) 以及容易與銅墊產生界金屬化合物 (inter-metallic compounds, IMCs)。這些因素都將降低無鉛焊料覆晶封裝製程產品的可靠度。因此，針對具有低介電材料的覆晶球狀陣列封裝體，如何調配其底膠的機械性質以增加其可靠度是目前研發工作的主要課題。

在本研究中，吾人利用高解析雲紋干涉儀去量測並比較二種不同底膠材料的熱機械變形。該儀器解析度可達 26nm，足以充分觀測底膠與錫球間的熱變形行為。根據量測的結果，具有較高彈性模數的底膠材料會造成較大的晶片翹曲與較低的錫球剪應變。此外，我們亦藉由熱循環測試 (Thermal Cycling Test, TCT) 評估了包含不同底膠與焊料合金的六種試片之可靠度。測試的結果指出了高鉛焊料 (Sn95Pb) 及無鉛焊料 (Sn0.7Cu) 較傳統的錫鉛共晶焊料 (Sn37Pb) 可靠度來的差。因此需要機械性質較強的底膠材料來提供錫球足夠的保護。

另外，我們利用 ANSYS™ 商用軟體建立了一個簡化的三維有限元素模型。此模型預測的晶片翹曲度與雲紋干涉儀所量測的數據差異小於 5%。而且此模型針

對上述六種試片所預測的應力趨勢亦與熱循環測試實驗有著極佳的相關性。

最後，我們以有限元素分析法來找出應用於低介電覆晶封裝體的最佳底膠機械性質。底膠的熱膨脹係數、彈性模數與玻態轉換溫度是影響封裝體可靠度的三大主要因素。根據模擬的結果，較低的底膠熱膨脹係數能同時降低錫球與低介電層產生裂縫的風險。倘若底膠具有較高的彈性模數，雖然可以提供錫球較佳的保護，但是將會導致低介電層的應力升高。由於上述二個底膠的材料參數在環境溫度超過了玻態轉換溫度後會有急遽的變化，因此玻態轉換溫度在底膠的材料性質中亦扮演一個關鍵的角色。綜上所述，針對低介電覆晶封裝體我們建議使用具有適中的彈性模數、低熱膨脹係數以及高玻態轉換溫度的底膠材料。



The Effect of Underfill Materials' Thermo-mechanical Properties on The Thermal Deformation Behavior of Low-K Flip-Chip Ball Grid Array Packages

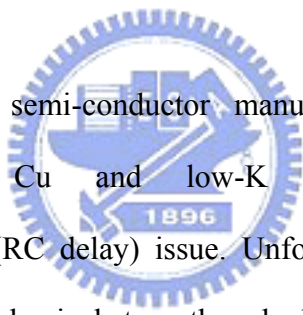
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Abstract



As the advancement of semi-conductor manufacturing technology, the IC manufacturers introduced Cu and low-K dielectric to relieve the resistance-capacitance delay (RC delay) issue. Unfortunately, the low-K dielectric material possessed weaker mechanical strength and adhesion than the traditional SiO₂ dielectric, leading to increased delamination potential for low-K layer. In addition, due to the rise of environmental awareness, the conventional eutectic solder alloy Sn37Pb was gradually replaced by lead-free alloys. However, lead-free alloys required higher reflow temperature and the alloys would form Cu-Sn inter-metallic compounds with Cu pads readily, which may degrade the reliability of lead-free packages. Therefore, how to formulate the thermal and mechanical properties of underfill materials to meet the reliability requirements for low-K flip-chip ball grid array (FC-BGA) packages is one of the critical tasks in current research and development of flip-chip technology.

In this study, a high resolution Moiré interferometry (resolution up to 26 nm) was

employed to measure and compare the thermo-mechanical deformation of two types of underfill materials. It could provide sufficient sensitivity to observe the thermal deformation behavior in bump/underfill layer. Based on the measurement results, the underfill material with higher elastic modulus induced larger die warpage and lower bump shear strain. In addition, we also evaluated the reliability of six FC-BGA package samples involving different underfill and solder alloys by thermal cycling tests (TCT). The TCT results indicated the package samples with high lead (Sn95Pb) and lead-free bump (Sn0.7Cu) had worse reliability than conventional Sn37Pb bump. Thus, the underfill with more rigid mechanical properties is required in order to protect solder bumps.

Furthermore, a simplified three-dimensional finite element model by ANSYSTM was also established. The die warpage difference between Moiré interferometry measurement and simulation was less than 5%. The stress simulation results by the finite element model also correlated well with aforementioned TCT results.

Finally, finite element analysis (FEA) was employed to find out the optimal mechanical properties of underfill material for low-K FC-BGA packages. The coefficient of thermal expansion (CTE), elastic modulus (E) and glass transition temperature (T_g) of underfill were the three major material properties which directly affected the reliability of FC-BGA packages. Based on FEA results, the underfill with lower CTE would decrease both stress on bumps and low-K layer. For the underfill with higher elastic modulus, it would enhance the bump protection, but induced higher stress in low-K layer. Since CTE and elastic modulus of underfill material would change drastically while environmental temperature exceeded its T_g temperature, T_g temperature was a critical materials property of underfill material. In summary, the underfill material with moderate elastic modulus, low CTE and high T_g temperature was recommended for low-K FC-BGA packages.

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Chapter 1 Introduction

In recent years, more and more electronic products appeared in our daily life. High-end electronic products such as microprocessors and graphics chips demanded powerful computing speed and low cost. Hence, the integrated circuits (IC) manufacturers continued scaling the device to enhance the performance and reduce cost. However, device scaling would increase RC delay due to increasing resistance and capacitance. In general, IC manufacturers employed Cu lines and low dielectric constant (low-K) material to relieve the RC delay issue. Unfortunately, the fragile low-K material exhibited a poor adhesion with silicon interface and weaker mechanical properties than traditional SiO₂ [1-2], which increased the risk of delamination between low-K and silicon interface.

Meanwhile, as transistor numbers increased, the packaging assembly needed more pin numbers, shorter connection distance, and more efficient cooling capacity. The flip-chip ball grid array (FC-BGA) packaging technology met aforementioned requirements and has been widely used in high-end electronics package in recent years [3]. However, the large mismatch of coefficient of thermal expansion (CTE) between silicon chip (2.6 ppm/°C) and bismaleimide triazine (BT) substrate (14 ppm/°C) may induce large die warpage and thermal stress during reflow process and thermal fatigue test, resulting in bump crack or low-K delamination. As a result, an underfill layer was introduced between bumps to mitigate the thermal induced stresses and enhance the reliability in the die/package interaction [4-6].

In the solder bumping, tin-lead solder bump is used in traditional bonding process. According to different requirements and processes, tin-lead solder bump could be divided into eutectic alloy, 63Sn37Pb which has a melting point of 183 °C, and the

high-lead alloy, 95Pb5Sn whose melting point is about 312 °C. However, lead compounds could infiltrate into the environment with rain water, resulting in the pollution of drinking water and crippling children's brain development if the electronic products were ineffectively recycled or willfully abandoned. On February 13, 2003, RoHS (Restriction of Use of Hazardous Substances) legislated by European Union required that leaded solder products were restricted to be sold in all Member States after July 1, 2006. In response to EU's RoHS, manufacturers had invested tremendously in the development of lead-free solder alloys, whose melting points can be summarized in Table 1.1 [7]. The melting points of common lead-free alloys are 210-227 °C, higher than the traditional 63Sn37Pb eutectic alloy (183 °C). The higher reflow temperature may induce thermal reliability issues due to larger ΔT incurred in the reflow and bumping steps. Therefore, how to adjust the thermal and mechanical properties of underfill material to protect fragile low-K layer and solder bumps has become a critical challenge for packaging industry.

This thesis is organized into five chapters as described briefly below:

- (1) Chapter 1 gives a brief introduction on this thesis.
- (2) Chapter 2 describes the literature review of key packaging technologies and motivations of this study.
- (3) Chapter 3 illustrates the theorems of Moiré interferometry and finite element analysis, and describes the procedures of sample preparation.
- (4) Chapter 4 covers the experimental and simulation results, and discussion.
- (5) Chapter 5 summarizes the key findings and contributions of this thesis work.

Table 1.1 The melting points for common lead-free solder alloys

Alloy	99.3Sn0.7Cu	96.5Sn3.5Ag	Sn3.5Ag0.9Cu	SnAgCuBi
Melting point (°C)	227	221	217	210-216

Chapter 2 Literature review

2.1 Evolution of IC industry

Due to the requirements of multi-function and high speed computing, more transistors were accommodated in a single chip. As the integrated density increased and device size scaled, 2-3 metal-layers interconnects design in $> 0.25 \mu\text{m}$ node no longer met the requirements of wiring signals and current carrying. Below the 65 nm node, 9-11 metal layers were adapted in the IC design, as illustrated typically by Fig. 2.1 [8].

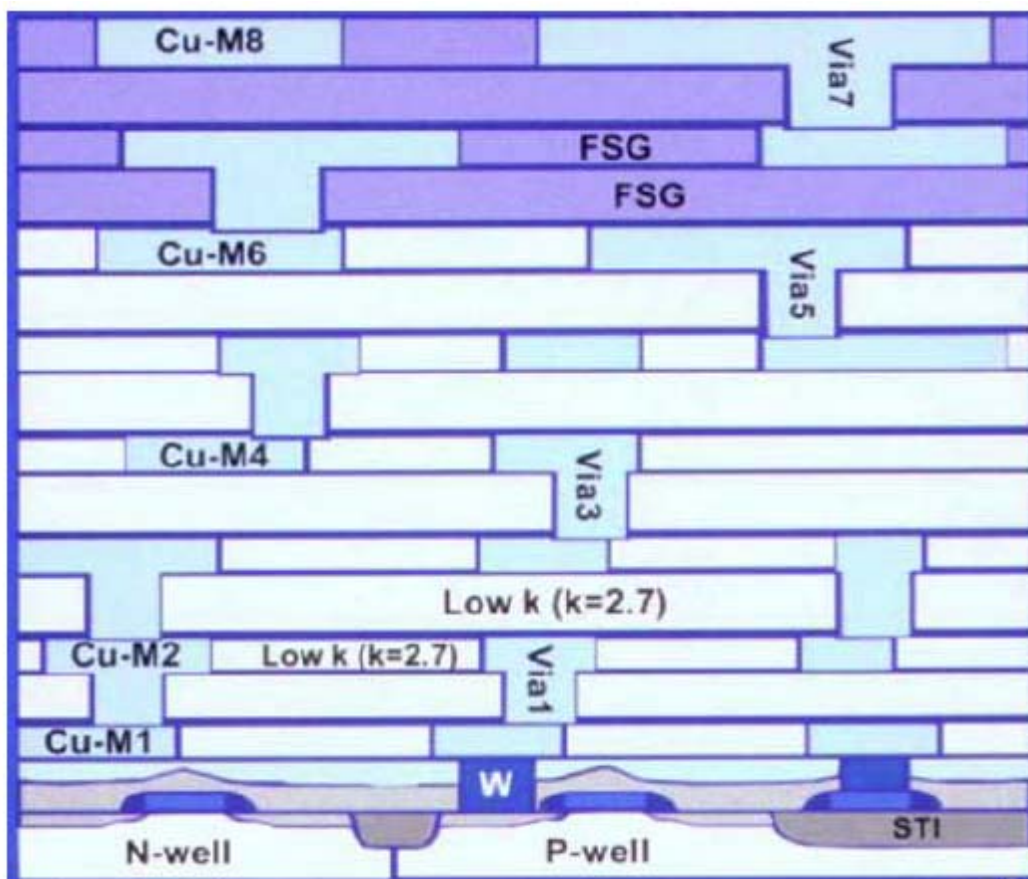


Figure 2.1 The structure of multi-metal layers [8]

Moreover, due to the great progress of semiconductor manufacture technology and the demand of cost down, the manufacturers in recent years continued to shrink IC device following the Moore's Law which stated the number of transistors on a chip would be doubled about every 24 months [9].

Table 2.1 revealed the IC manufacture technology roadmap of near-term years according to ITRS Roadmap [9]. However, the continued shrink of the metal line widths also brought some side-effects. Since the increase of total length of metal circuit by multi-layers design and the reduction of metal line widths which made the total resistance value increased substantially. In addition, the capacitance also increased due to the smaller pitch of metal lines. Therefore, the propagation performance of the chip was limited by interconnect delay for technology beyond 0.25 μm node instead of gate delay, as shown in Fig. 2.2 [10]. Thus, the IC manufacturers employed Cu to replace Al as the metal wiring, and used low-K dielectric material to relieve the increase of the RC delay.

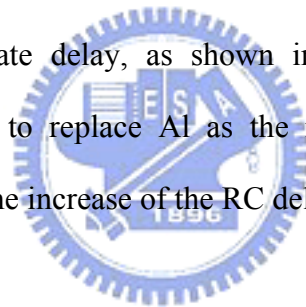


Table 2.1 The roadmap of IC manufacture technology (unit: nm) [9]

Year	2007	2008	2009	2010	2011	2012	2013	2014	2015
DRAM 1/2 M1 pitch	65	57	50	45	40	36	32	28	25
MPU/ASIC 1/2 M1 pitch	68	59	52	45	40	36	32	28	25
Flash Poly Si	54	45	40	36	32	28	25	23	20
MPU Printed Gate length	42	38	34	30	27	24	21	19	17
MPU Physical Gate length	25	23	20	18	16	14	13	11	10

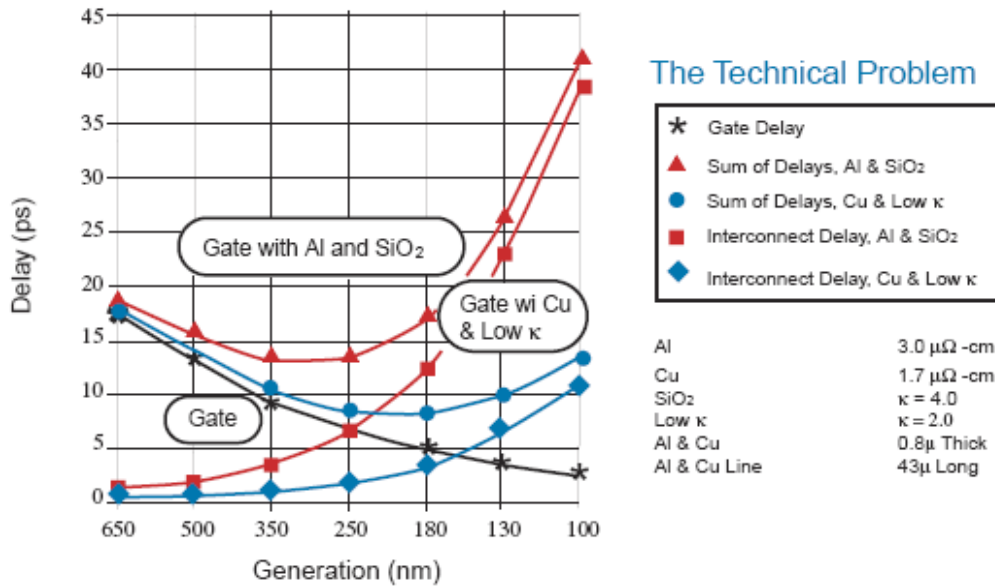


Figure 2.2 Gate delay, interconnect delay and total RC delay versus technology node [10]

Figure 2.3 showed that the trend of effective dielectric constant in ITRS 2007 roadmap [11]. However, the next generation ultra low-K (ULK) dielectric materials ($K < 2.5$) have to introduce porous materials. Unfortunately, the nature material characteristic of porous materials would further degrade material strength and reduce interface adhesion.

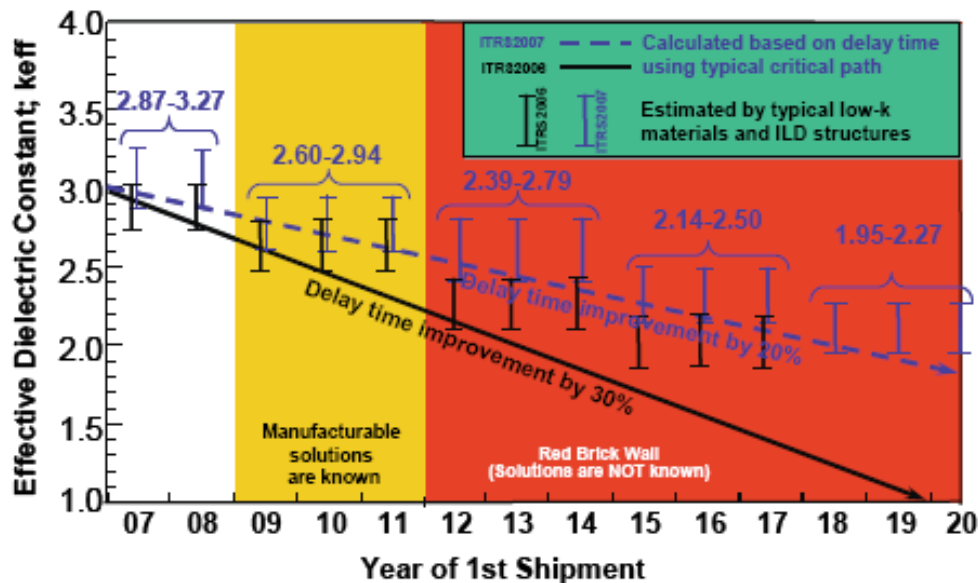


Figure 2.3 K_{eff} roadmap from ITRS 2007 [11]

2.2 Introduction of electronic packaging

With the development of several generations in electronic industry, the process of IC devices scaled down to the nanometer scale. These sophisticated, brittle IC devices could not sustain any surrounding collided and contamination. Therefore, the electronic packaging technologies were applied to protect IC devices and transmit the signals and current between transistors and mother board. The electronic packaging technology has four major purposes which were described below [12]:

1. to support and protect the IC devices
2. to transmit the signals and supply the power
3. to dissipate the heat
4. to avoid the delay of signal transmission

In fact, a complete electronic product needed to go through several package processes. Usually, the electronic packaging could be divided into five different levels as illustrated in Fig. 2.4 [13].

- (1) The zero level packaging:

It involved the IC design and fabrication of chips.

- (2) The first level packaging:

This step would stick the chip into a packaging module. The circuit connection and sealing was completed at this step. This packaging was also called the chip level packaging.

- (3) The second level packaging:

In this packaging, the first level packaging devices were connected to the circuit boards.

- (4) The third level packaging:

This packaging connected several circuit boards into a motherboard.

(5) The fourth level packaging:

In this level packaging, several motherboards were combined to form a complete electronic product.

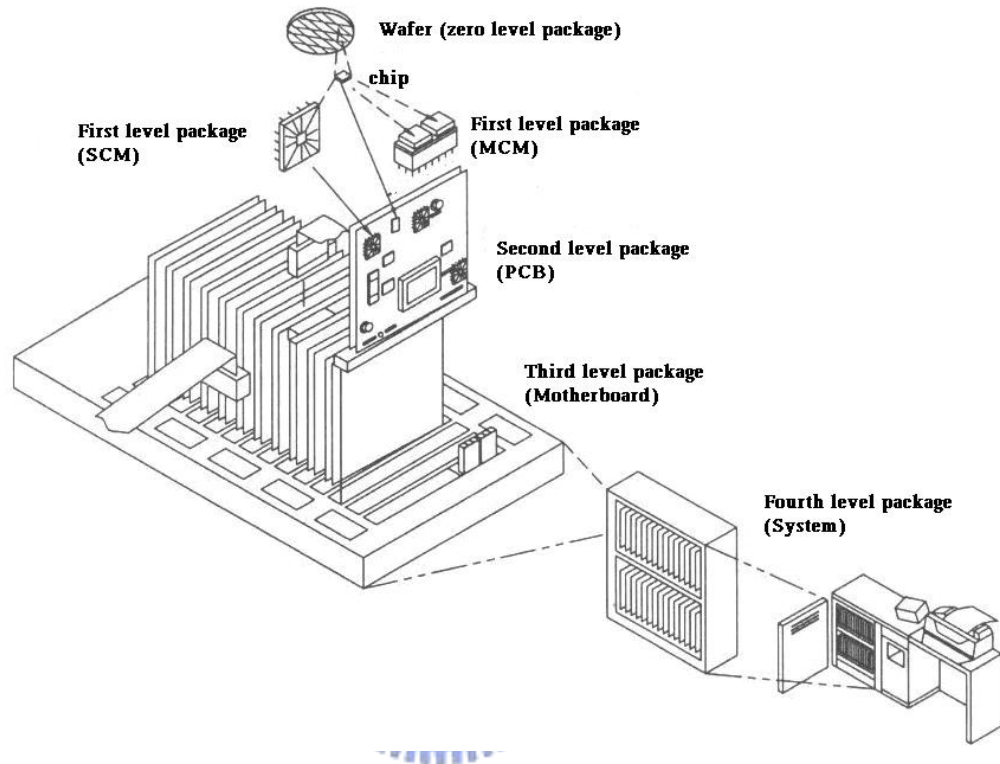


Figure 2.4 Five different levels of electronic packaging [13]

Different electronic products possessed various shapes and functionalities which resulted in the different demands for packaging technologies and materials.

According to the utilized materials, electronic packaging could be divided into ceramic packaging which had better heat dissipation efficiency and reliability, and plastic packaging which had the advantages of low-cost and a thinner volume.

In addition, electronic packaging could also be classified into Pin-Through-Hole (PTH) packaging and Surface Mount Technology (SMT) packaging by IC devices and circuit boards connection methods. Table 2.2 summarized most of packaging types [14].

Table2.2 IC packaging types

Connect type	Pin style	Pin appearance	Packaging Type
PTH	Single inline	Vertical to side area	SIP
		Zig-zag	ZIP
	Dual inline	Parallel to side area	DIP/SK-DIP
	Bottom	Needle	ZIP
SMT	Single inline	L type	SVP
	Dual inline	L type	SOP/TSOP/SSOP
		I type	SOP
		J type	SOJ
	Quad inline	L type	QFP/LQFP/TQGP
		I type	QFI
		J type	QFJ
		Electrode bump	QFN
	Bottom	Pin type	PGA
		Ball type	BGA

2.3 Introduction of flip-chip technology

As the advancement of IC industry, the shapes of electric products were requested to be lighter, thinner, shorter, and smaller. These requirements would render more challenges in packaging technology. In general, there are three major methods to accomplish circuit connection, which were wire bonding, tape automated bonding (TAB), and flip-chip (FC) packaging technologies. Table 2.3 summarized the comparison of key features among these three package methods [3].

Table 2.3 The comparison among wire bond, TAB, and FC technologies

	Wire bond	TAB	Flip chip
Area ratio	1	1.33	0.33
Weight ratio	1	0.25	0.2
Thickness ratio	1	0.67	0.52
I/O numbers	300~500	500~700	>1000
Bond pad pitch	~50 μm	40 μm	~150 μm
Ball size	~40 μm	NA	~150 μm
Interval of bond pad	100~180 μm	80 μm	~300 μm

Unlike the wire bonding and TAB packaging, which were peripheral array bonding, the flip-chip packaging employed area array connection. This technology could provide high I/O pins density packaging for high-end electronic products. The flip-chip packaging technology also provided other advantages which were listed as below:

1. shorter signal transmission distance

2. lower propagation delay
3. lower self-inductance
4. better heat dissipation and reliability

The flip-chip packaging technology was also called C4 (Controlled Collapse Chip Connection) packaging, when it was proposed by IBM in 1960s [15]. Figure 2.5 showed the basic structure of a flip-chip assembly [16]. The solder bumps were deposited on I/O pads, and then, the chip was flipped and heated to connect with the substrate by molten solder bumps. The flip-chip packaging process could be divided into two steps; namely: flip-chip bumping and flip-chip assembly.

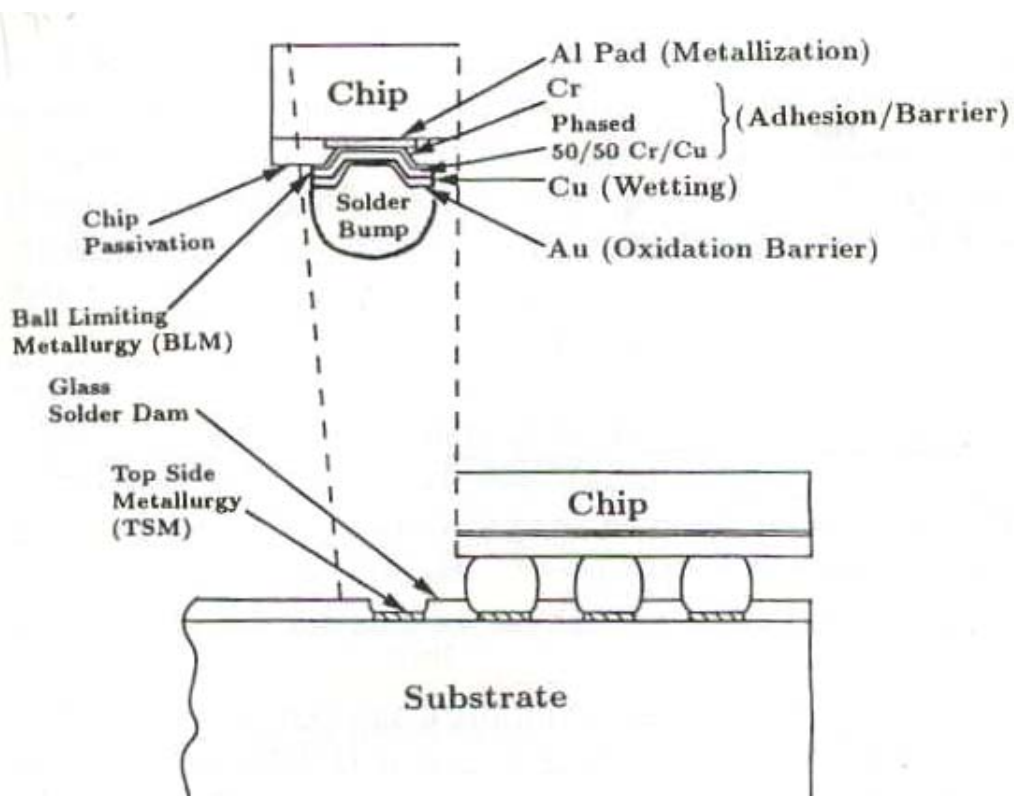


Figure 2.5 Flip-chip packaging structure [16]

2.3.1 Flip Chip Bumping

The solder bump structure included two parts, which were under bump metallurgy (UBM) and solder ball. The UBM structure usually included three metal layers. The function and materials of each layer were described as below [3]:

1. Adhesion layer:

This layer was employed to enhance the adhesion to bond pads. The main materials of this layer were Ti and Cr.

2. Wetting layer:

The wetting layer increased the adhesion between solders and the adhesion layer. The main materials in this layer were Au, Ag, Cu, and Ni.

3. Protective layer

The main objective of this layer was to protect the Cu or Ni from oxidation. The precious metals like Au, often used in this layer.

The common solder included Sn37Pb eutectic, high lead and lead-free alloys. The solder alloy could be deposited on UBM by evaporation, and electro-plating, and stencil printing. The solder alloy would be melted while the temperature above its melting point, and then formed solder balls after cooling to room temperature. Figure 2.6 illustrated the bumping manufacturing process by electro-plating method [17].

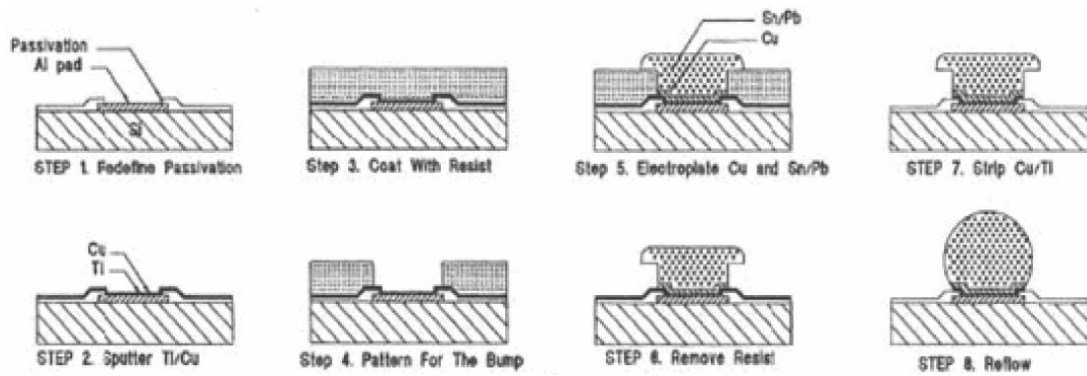


Figure 2.6 The solder bump manufacturing process by the electro plating method [17]

2.3.2 Flip-chip assembly

After the bumping process, the chip had to connect with the substrate to complete the flip-chip assembly. At first, the solder bumps were aligned with the bond pads on the substrate. Afterwards, the assembly was heated and the solder balls would be melted to connect the substrate. During this reflow process, the FC-BGA packaging showed unique self-alignment advantage to avoid bonding failure as shown in Fig. 2.7 [18]. After flux cleaning and underfill dispensing, the package would be heated at about 150 °C to cure the underfill material. The role of underfill layer was to protect solder bumps and die from crack and delamination. The major ingredients of underfill included epoxy resin, hardener, catalyst, filler and some additives [3]. The overall process of flip-chip assembly was shown in Fig. 2.8 [19].

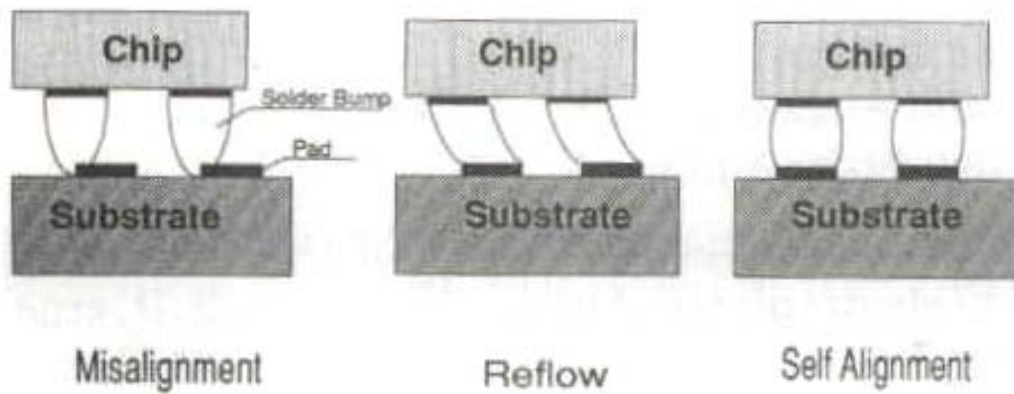


Figure 2.7 Self-alignment of FC-BGA assembly [18]

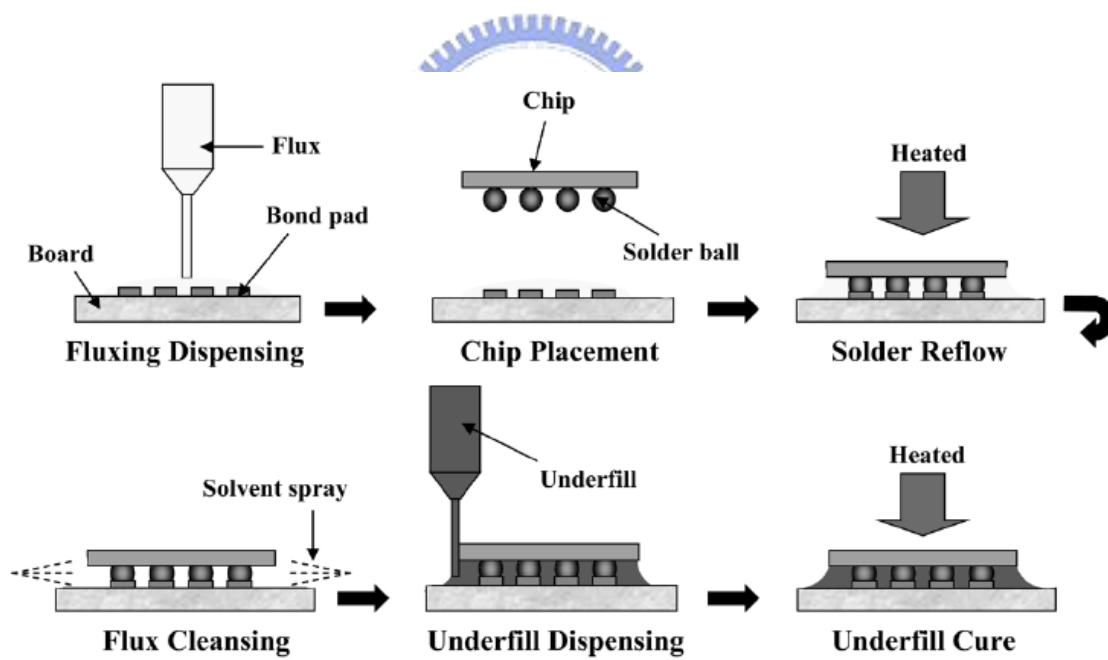


Figure 2.8 Flip-chip assembly process [19]

2.4 The challenges in flip-chip packaging technology

The reliability issues of FC-BGA assembly had been of critical concern and closely studied in recent years. Usually, several factors may cause reliability degradation after thermal processing step in flip-chip assembly and thermal cycling test as listed below [20-23]:

1. The inter-metallic compounds (IMC) formation,
2. The CTE mismatch between each material,
3. The thermo-migration behavior, and
4. The electromigration effect.

Due to the CTE mismatch between silicon chip and substrate, the FC-BGA package incurred large warpage as shown in Fig. 2.9. However, the warpage could be mitigated by optimal designs, such as smaller and thinner die, or higher E of thermal interface material (TIM) etc [24].

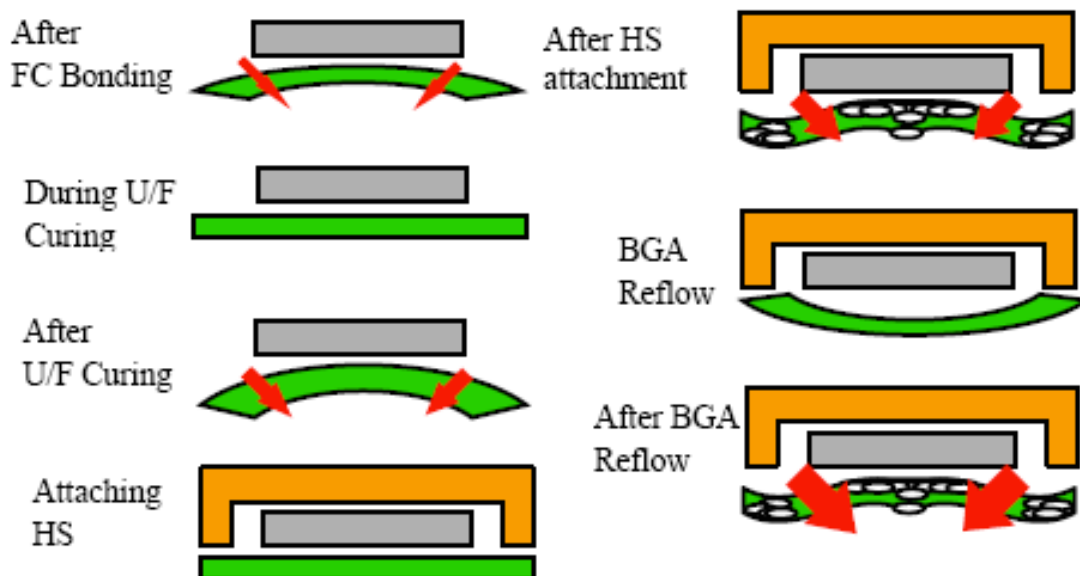


Figure 2.9 The warpage of FC-BGA assembly [24]

The silicon chip and organic substrate were connected by solder joints. Therefore, the thermal induced stresses may cause bumps cracks during thermal fatigue tests. Figure 2.10 showed the bump crack occurred at die/solder interface after TCT 100 cycles [25]. H. C. Cheng *et al.* evaluated the relationship between solder joint fatigue life and die/PCB material properties [26]. Pang *et al.* also investigated the solder bumps creep phenomenon by simulation [27]. S. K. Groothuis applied non-linear, viscous and plastic properties of solder balls to predict the fatigue life [28].

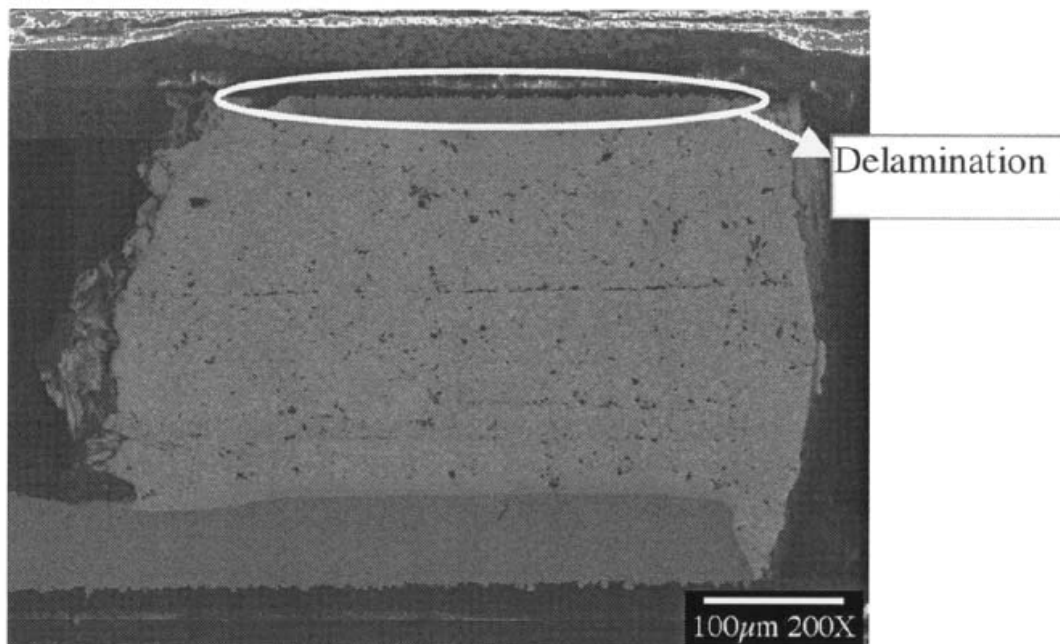


Figure 2.10 The SEM picture of the outmost bump crack after TCT 100 cycles [25]

In addition, the fragile low-K dielectric layer had high delamination risk due to its weak stiffness and poor adhesion. Figure 2.11 showed the typical delamination at die/low-K interface [29]. K. C. Chang *et al.* proposed some design guidance for increasing reliability of low-K FC-BGA assemblies [30]. L. Mercado proposed that put some tiles or slots in the interconnect layer to reduce available area for crack growth [31]. M. Rasco *et al.* evaluated the delamination risks of ultra low-K material for different passivation types [32]. L. L. Mercado *et al.* pointed out that the more Cu

layer of interconnects, the higher risk of low-K delamination [33]. This increased the difficulties of FC-BGA for high-end electronic products.

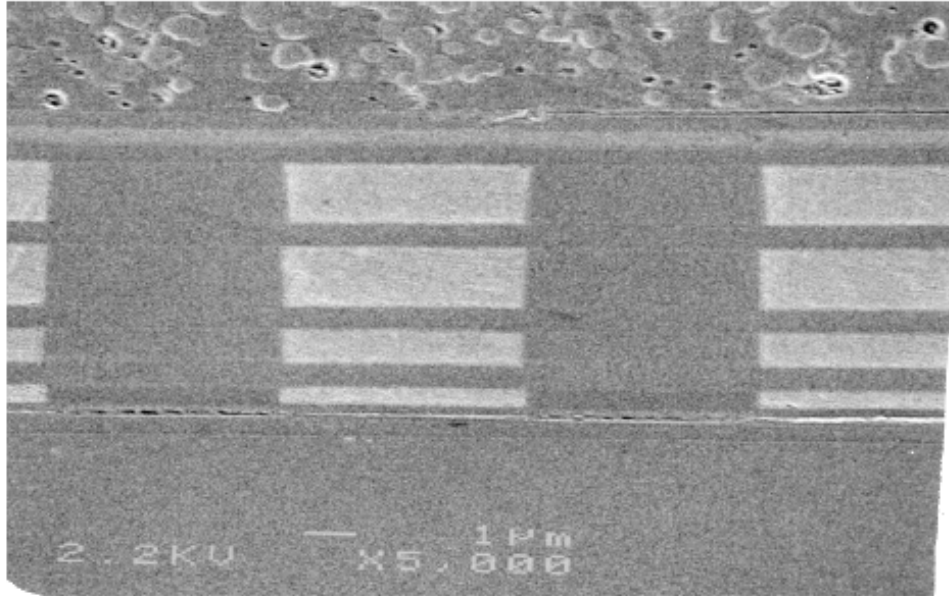


Figure 2.11 The SEM picture of low-K delamination [29]



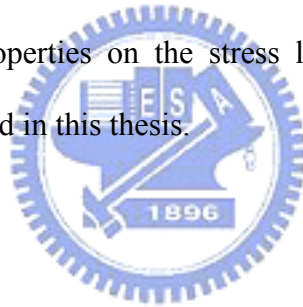
2.5 Motivation of this thesis

As devices continued scaling down to 45 nm node, copper and ultra low-K dielectric materials ($k < 2.5$) has become the mainstream in the backend interconnects for further reduction in RC delay. Since the fragile low-K layer possessed weaker mechanical properties and poor adhesion with silicon chip, the low-K delamination would occur easily than traditional SiO_2 dielectric due to the thermal induced stresses by CTE mismatch between die and BT substrate. In addition, the traditional SnPb eutectic solder was gradually replaced by lead-free solder alloy due to the rise of environmental awareness. The underfill layer played an important role to mitigate the thermal induced stresses from CTE mismatch of each component. Both solder bumps and low-K layer should be protected from cracks and delamination. Therefore, the material properties of underfill should be modified for different FC-BGA packaging applications. Some studies indicated that high modulus and high glass transition temperature were good for bump protection, but increase delamination potential for low-K layer [29, 34]. It is difficult to find an optimal underfill material to protect both bumps and low-K layer perfectly due to its opposite requirements in mechanical properties. Thus, how to balance three major material properties (E, CTE and T_g) of underfill materials to protect both bumps and low-K layer becomes a critical challenge for IC packaging industry.

Due to the advancement in simulation programs and computation performance with more powerful computer, the finite element analysis (FEA) had been a popular tool for thermo-mechanical analysis of die/package interaction. The simulation model should be verified by experimental data prior to predictive simulation. The Moiré interferometry had been used to measure the thermal deformation of FC-BGA packages [25, 26, 35]. Unlike regular Moiré interferometry, the high resolution Moiré

interferometry utilized phase shift technology to enhance the resolution to 26 nm. The thermal induced strain in the solder bumps/underfill layer could be measured more exactly by the high resolution Moiré interferometry, which in turn can be used to validate simulation model and results.

In this study, a 3D simulation model using ANSYSTM was first established to analyze the warpage and strain distribution in FC-BGA packages with Sn0.7Cu solder and two latest underfill materials, and validated by experimental results from Moiré interferometry measurement. Such model was then used to analyze the stresses in the bumps and layer-k layer of six FC-BGA package samples with various solder alloys (Sn0.7Cu, Sn95Pb and Sn37Pb) and underfill materials (various E, CTE and Tg) and to compare with reliability results from temperature cycling test (TCT). The impact of various underfill material properties on the stress level at the outmost bump and low-K layer was also addressed in this thesis.



Chapter 3 Experimental Methods

3.1 Moiré interferometry

3.1.1 Introduction

The vocabulary of Moiré is from French whose meaning is a watered silk or mohair fabric. In fact, the Moiré pattern is a common optical phenomenon in our daily life. For example, when light passed through two overlapped combs which have the approximate teeth pitch, we can observe several broad black lines which are called the Moiré fringes as shown in Fig. 3.1 However, there are some conditions need to be met if we want to obtain distinct fringe patterns for science research [36]:

- Equal widths of the bars and spaces
- The two gratings are well defined
- The intersection of the two gratings is less than 3 degree
- The pitches ratio of the two gratings is less than 1.05

Lord Rayleigh was the first man who employed Moiré fringes to measure object deformation in 1874 [37]. In 1956, J. Guild developed the geometric Moiré by optical interference and diffraction phenomenon [38]. This is the predecessor of Moiré interferometry. In order to achieve highly sensitive measurement, the high frequency grating and coherent light source need to be developed. However, the two key techniques were not mature enough yet, so that Moiré interferometry did not attract much attention in 1950s. The Moiré interferometry has been widely applied in various kinds of deformation measurement until the laser beam and high frequency grating with 1200 lines/mm had been developed in 1980s.

Figure 3.2 showed the schematic diagram of Moiré interferometry. It can provide *in-situ*, whole-field and in-plane displacement results. The resolution of regular Moiré

interferometry can reach $0.417 \mu\text{m}$ using a grating with 1200 lines/mm [36]. If we employed the phase-shift technology for Moiré interferometry, the sensitivity could be enhanced to 26 nm [39].



Figure 3.1 The geometry Moiré fringes

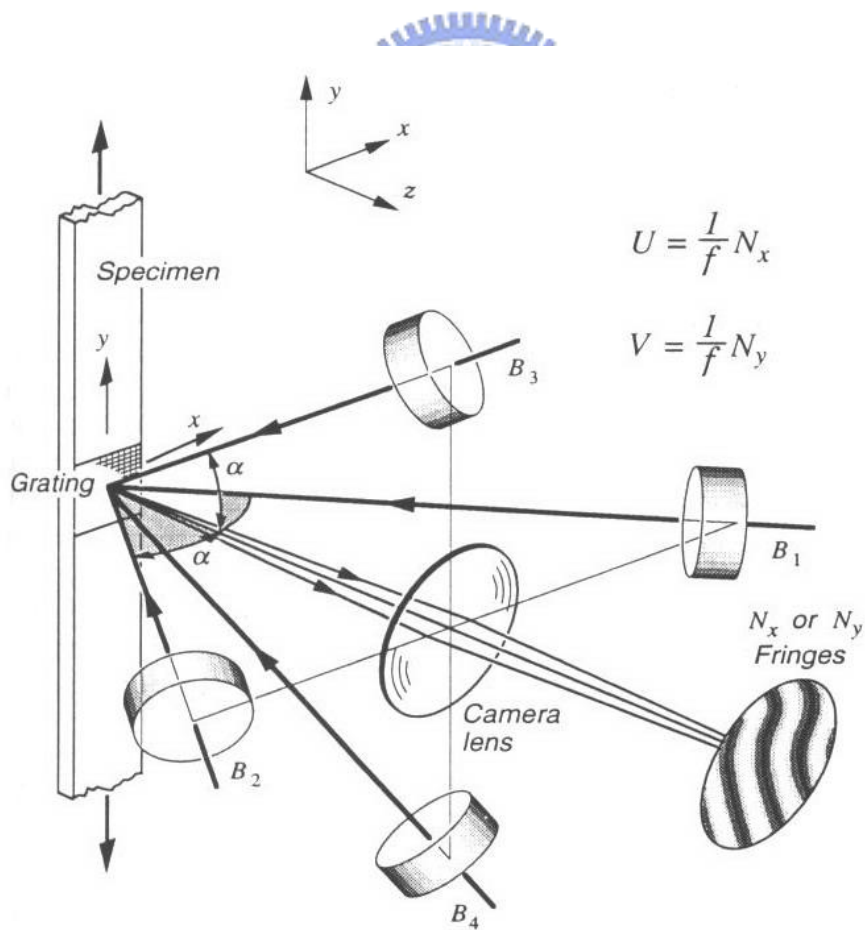


Figure 3.2 The schematic diagram of Moiré interferometry

3.1.2 Theorem of Moiré interferometry [36]

First, we considered two intersected coherent beams as shown in Fig. 3.3. The two laser beams would produce optical constructive and destructive interference in the three-dimensional intersection space. The constructive interference formed some equal spaced relatively high intensity planes. Figure 3.4 revealed the recorded interference image at the cross-section plane (BB) from Fig. 3.3 by a photographic. The adjacent dark and bright bands, namely fringes were observed. The frequency of the fringes, or the fringe gradient on the plane, F , was determined as:

$$F = \frac{1}{G} = \frac{2}{\lambda} \sin \theta \quad (3.1)$$

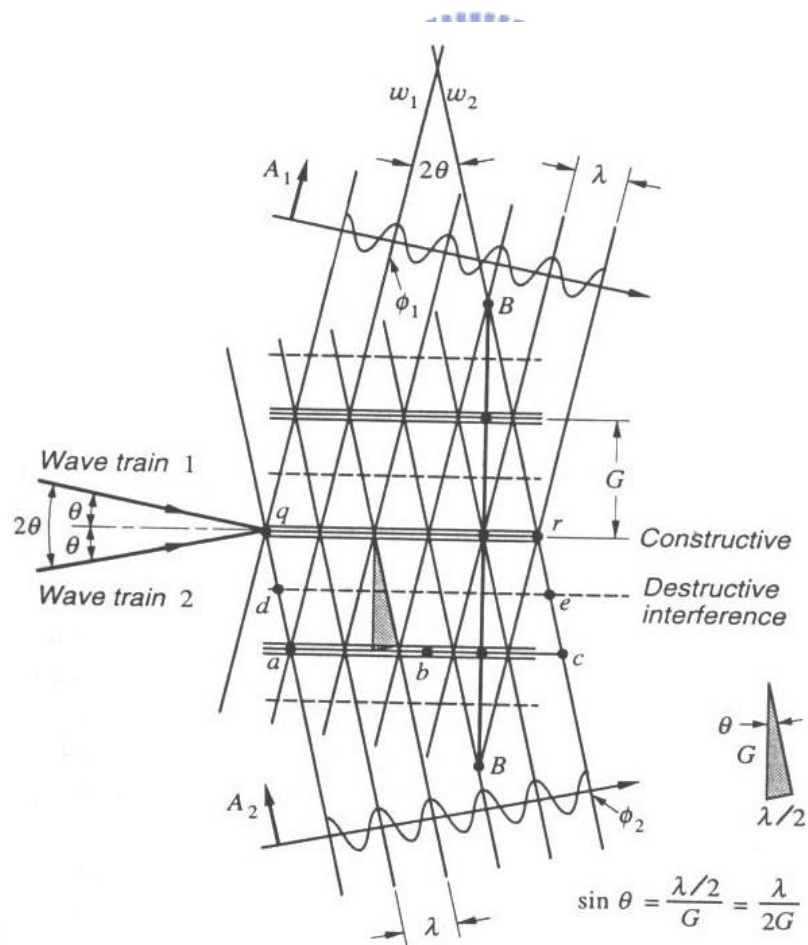


Figure 3.3 The interference phenomenon

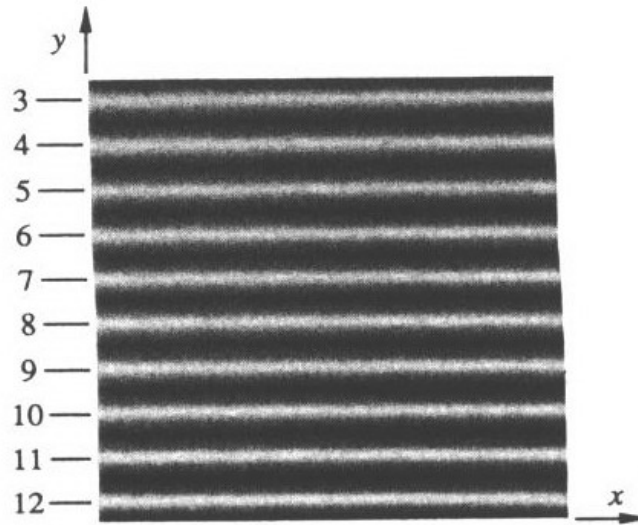
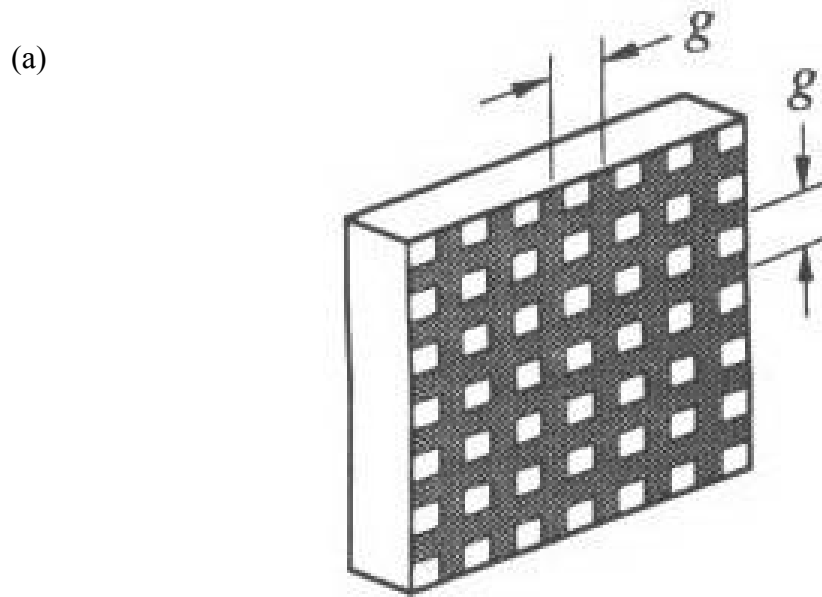


Figure 3.4 The recorded pattern in plane BB

The Moiré interferometry needs a diffraction grating to produce Moiré fringes. Figure 3.5 (a) illustrated the *cross-line* grating which had the same pitch (distance = g) in two orthogonal direction. Figure 3.5 (b) showed the scanning electronic microscope image of a diffraction grating surface with the frequency of 1200 lines/mm. The surface showed regularly spaced bars and furrows to produce optical diffractions. The frequency of a grating (f) means the number of bars per unit length. It often expressed as lines per millimeter or inch. In general, a low frequency grating ($f = 10$ to 50 lines/mm) is used for geometric Moiré, and a high frequency grating ($f = 300$ to 2400 lines/mm) is used for Moiré interferometry. The relationship between grating frequency and pitch could be expressed as:

$$f = \frac{1}{g} \quad (3.2)$$



(b)

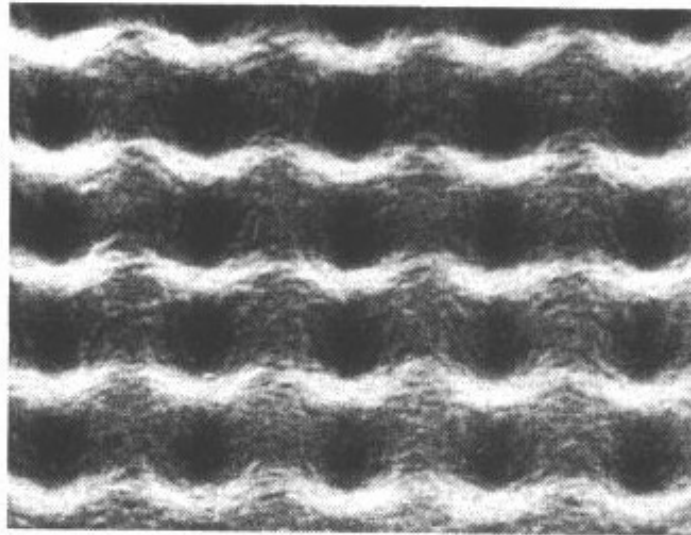


Figure 3.5 (a) A schematic diagram of a grating

(b) The SEM picture of the surface of grating

The incident laser beam will be divided into a number of diffracted beams by a reflection grating which was shown as Fig. 3.6. The diffraction angles will follow the grating equation showed as below:

$$\sin \beta_m = \sin \alpha + m\lambda f \quad (3.3)$$

where m is the diffraction order, β_m is the angle of the m^{th} diffraction angle, α is the incident angle, λ is the wavelength, and f is the grating frequency.

Figure 3.7 illustrated a special case while the zeroth diffraction angle is equal to the incident angle α , and -1th diffraction order is $-\alpha$. Thus, we will obtain the following equation by substituting $\beta_{-1} = -\alpha$ to Eq. (3.3).

$$\sin \alpha = \frac{\lambda}{2} f \quad (3.4)$$

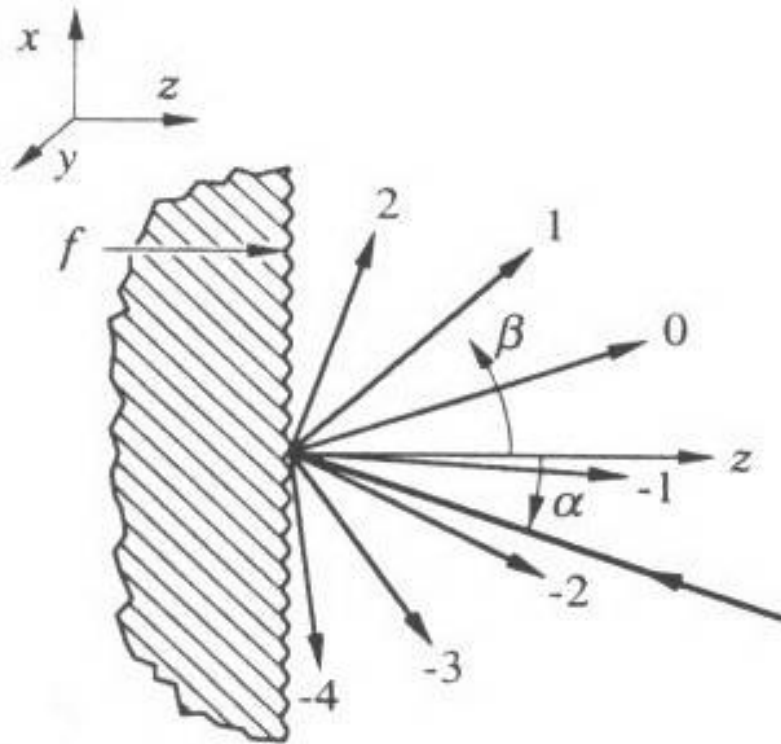


Figure 3.6 The diffraction orders of a reflection grating

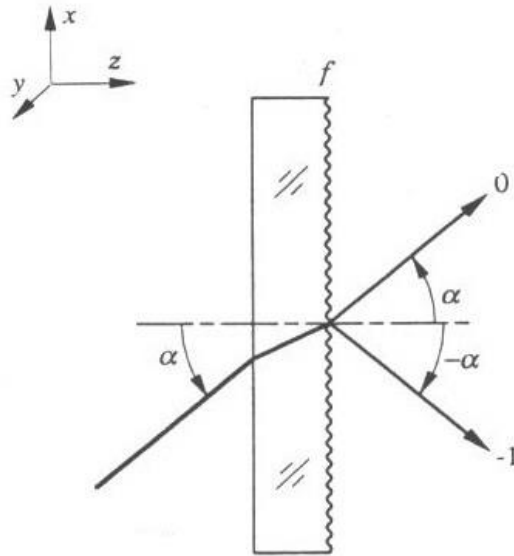


Figure 3.7 The zero and -1 diffraction order when $\beta_0 = \alpha$, and $\beta_{-1} = -\alpha$

For Moiré interferometry, we could create a “virtual” grating by constructive and destructive interferences of two coherent beams. Figure 3.8 showed the interference of two coherent beams. If we let $\theta = \alpha$, we can get $f = F$ (refer to Eq. 3.1 and 3.4).

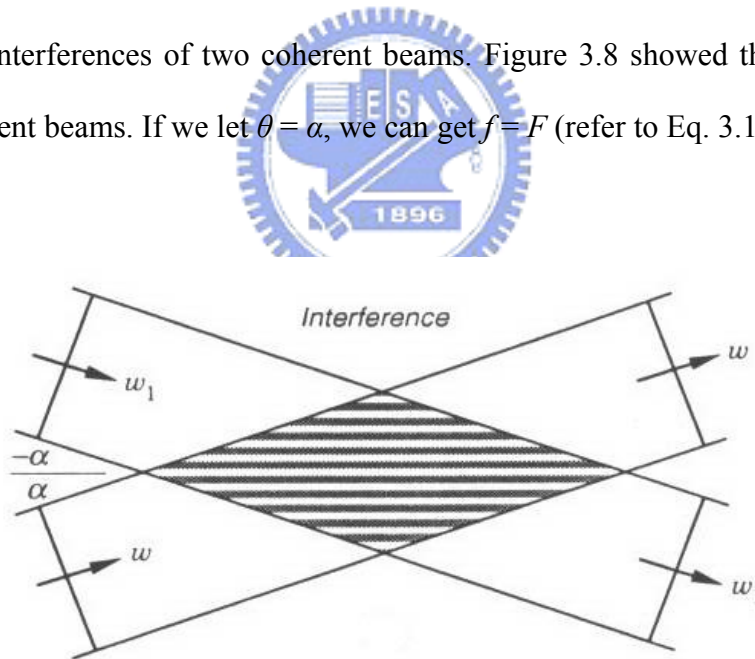


Figure 3.8 The interference of two intersected coherent beams

The Moiré interferometry used both interference and diffraction phenomenon to measure deformations of an object as shown in Fig. 3.9. The real grating had been attached to the specimen surface with the frequency of f_s . The specimen grating will

deform with the specimen. Two coherent beams produce the virtual reference grating which had the frequency of f . The Moiré fringes would be generated by the interaction of the deformed specimen and reference grating. The camera was used to record the fringe patterns. The relationship between f and f_s can be expressed as:

$$f = \beta f_s \quad (3.5)$$

where f is the frequency of reference grating, f_s is the frequency of specimen grating, and β is an integral number standing for the fringe multiplication factor. In this study, $\beta=2$.

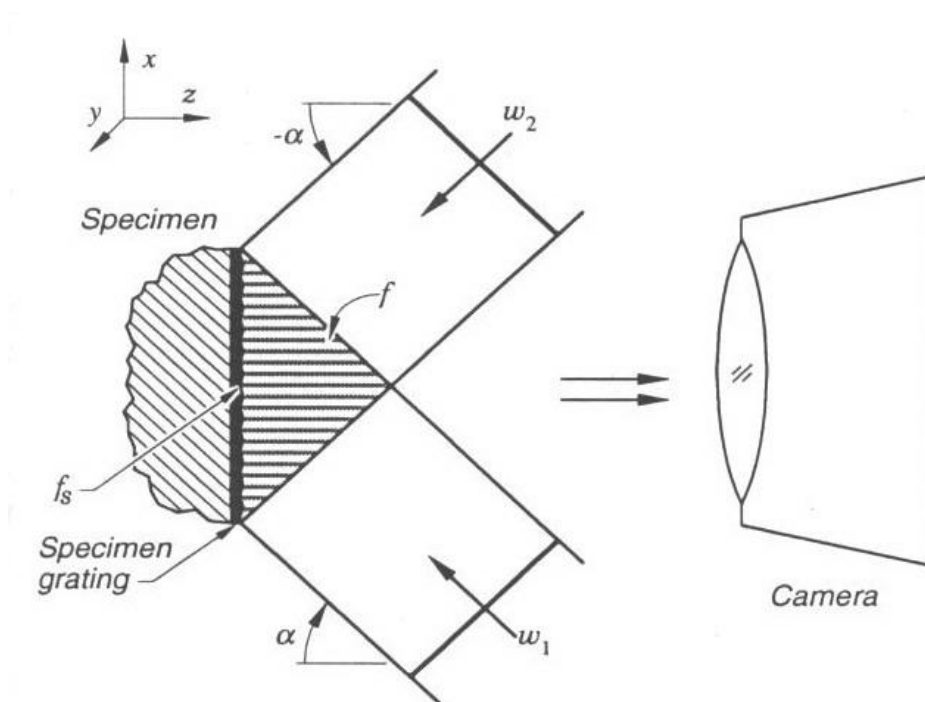


Figure 3.9 The interference and diffraction on the specimen grating

Figure 3.10 depicted the diffraction and interference in Moiré interferometry more detail. The specimen grating had been attached to an undeformed specimen. We could obtain the following relationship from Eq. (3.3):

$$\sin \beta_m = \sin \alpha + m \lambda f_s \quad (3.6)$$

If we substitute the following governing conditions for beam 1: $m = 1, f = 2f_s$, and $\sin(-\alpha) = -\frac{\lambda}{2} f$ (refer to Eq. 3.4)

We would find that:

$$\sin \beta_1 = 0$$

We also obtained the same result for beam 2 by substituting $m = -1$.

Thus, the intersected angle of the two coherent diffraction beams was equal to zero. This represented zero fringe frequency (0 lines/mm), namely null field, would be generated.

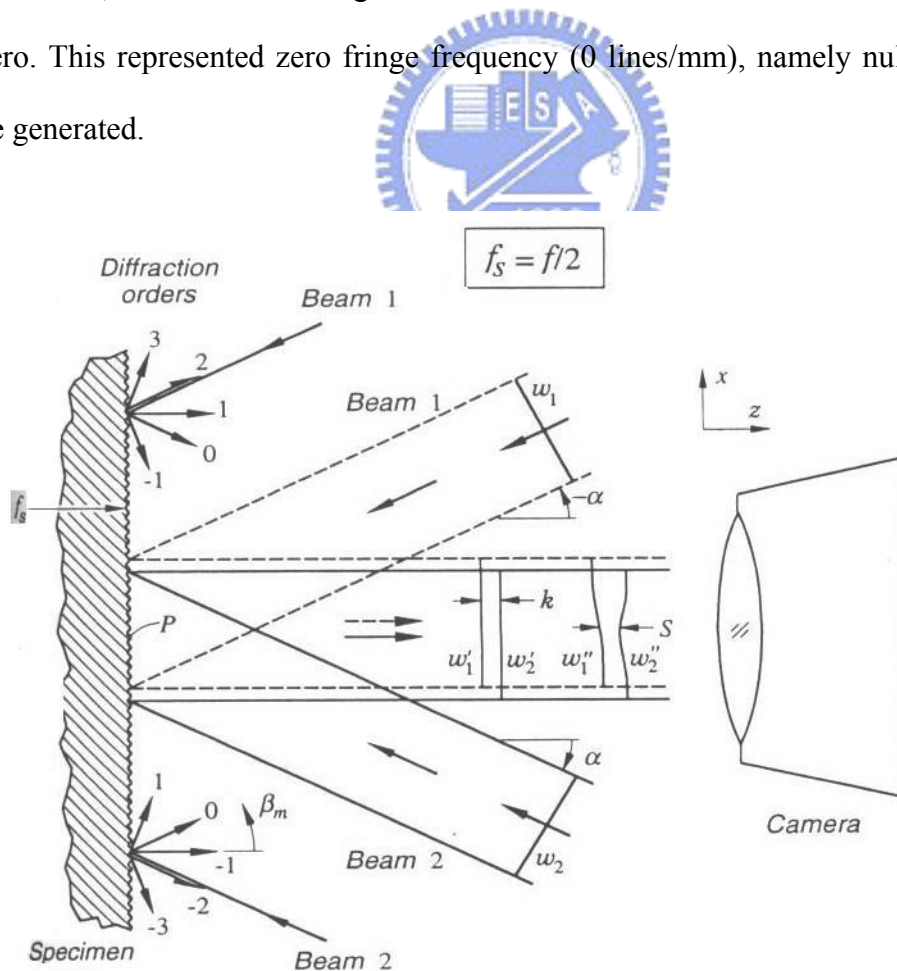


Figure 3.10 The null field of Moiré interferometry

Now, we considered a uniform normal strain (ε_x) in the X direction due to a force applied to a specimen. Thus, the new frequency of the specimen should be modified as:

$$f_s = \frac{f/2}{1 + \varepsilon_x} \quad (3.7)$$

We substituted the Eq. (3.7) to Eq. (3.6) for the 1st diffraction order beam of beam 1:

$$\sin \beta_1 = \sin(-\alpha) + \frac{\lambda f}{2(1 + \varepsilon_x)}$$

By Eq. (3.4):

$$\sin(-\alpha) = -\frac{\lambda}{2} f$$

Thus,

$$\sin \beta_1 = -\frac{\lambda f \varepsilon_x}{2(1 + \varepsilon_x)}$$

Since the value of β_1 and ε_x are very small, therefore:

$$\beta_1 = -\frac{\lambda f \varepsilon_x}{2} \quad (3.8)$$

For the -1st diffraction order beam of beam 2, we can also get the similar equation as below:

$$\beta_{-1} = \frac{\lambda f \varepsilon_x}{2}$$

The two coherent diffraction beams had an interacted angle of $2\beta_l$. We could substitute $\sin \theta = |\beta_l|$ in Eq. 3.1 due to θ and β_l were very small. Thus, we could find the following equation as:

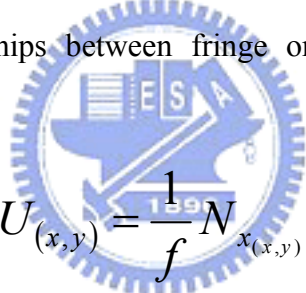
$$F_{xx} = f\varepsilon_x \quad (3.9)$$

where F_{xx} was the fringe gradient in the X direction. It could also express as $\left(\frac{\partial N_x}{\partial x}\right)$.

Since,

$$\varepsilon_x = \frac{\partial u}{\partial x} = \frac{F_{xx}}{f} = \frac{1}{f} \left(\frac{\partial N_x}{\partial x} \right)$$

Therefore, the relationships between fringe order and displacement can be determined as:



$$U_{(x,y)} = \frac{1}{f} N_{x(x,y)}$$

$$V_{(x,y)} = \frac{1}{f} N_{y(x,y)} \quad (3.10)$$

where U and V are the displacement of U field and V field, respectively N_x and N_y are the fringe orders of U field and V field, respectively.

The strains could be represented as:

$$\varepsilon_x = \frac{\partial U}{\partial x} = \frac{1}{f} \left(\frac{\partial N_x}{\partial x} \right)$$

$$\varepsilon_y = \frac{\partial V}{\partial y} = \frac{1}{f} \left(\frac{\partial N_y}{\partial y} \right)$$

$$\gamma_{xy} = \frac{\partial U}{\partial y} + \frac{\partial V}{\partial x} = \frac{1}{f} \left(\frac{\partial N_x}{\partial y} + \frac{\partial N_y}{\partial x} \right) \quad (3.11)$$

Figure 3.11 illustrated the four beams Moiré system. The incident laser beam produced two pair diffraction beams by a cross-line grating. These diffraction beams would produce a virtual reference grating with twice frequency of the actual grating. The Moiré fringe pattern would be observed while the specimen grating deform with the specimen by a CCD camera.

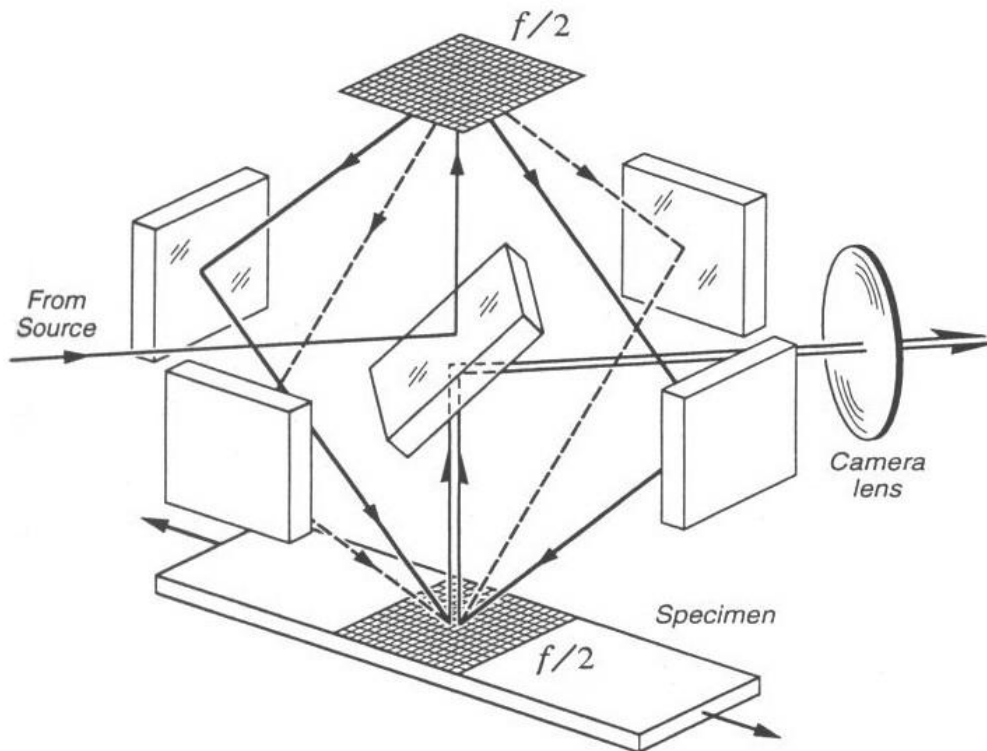


Figure 3.11 The four beams Moiré system

3.1.3 Phase shift technique

The resolution of a regular Moiré interferometry depends on the grating frequency. The most common grating frequency for Moiré interferometry is 1200 lines/mm which represents the resolution of 0.417 μm . However, such sensitivity is not enough to measure the deformation of solder bumps. As a result, a phase shifting Moiré interferometry was employed to enhance the resolution to 26 nm. Each fringe spacing is equal to a 2π phase angle difference and 0.417 μm displacement. The unknown phase angle can be extracted from four precisely phase-shifted interference patterns by phase shifting Moiré interferometry. The intensity of the four patterns can be expressed as [39]:

$$\begin{aligned} I_1(x, y) &= I_0(x, y) + I'(x, y)\cos[\phi(x, y)] \\ I_2(x, y) &= I_0(x, y) + I'(x, y)\cos\left[\phi(x, y) + \frac{\pi}{2}\right] \\ I_3(x, y) &= I_0(x, y) + I'(x, y)\cos[\phi(x, y) + \pi] \\ I_4(x, y) &= I_0(x, y) + I'(x, y)\cos\left[\phi(x, y) + \frac{3\pi}{2}\right] \end{aligned} \quad (3.12)$$

where,

$I_0(x, y)$ and $I'(x, y)$ are the background and periodically varying intensities in the interference pattern.

$\phi(x, y)$ is the unknown phase angle of the interference pattern at each pixel location. Each subsequent pattern is obtained by shifting a phase angle of exactly $\pi/2$ of the fringe period.

The different intensities of the four patterns can be used to determine the

unknown phase angle by the following equation:

$$\phi = \arctan \frac{I_4 - I_2}{I_1 - I_3} \quad (3.13)$$

After the phase angle is solved, the continuous displacement of U field can be determined as:

$$u = \frac{\phi}{4\pi f} \quad (3.14)$$

The displacement of V field is similar to Eq. (3.14).



3.2 Finite Element Analysis (FEA)

3.2.1 Introduction to FEA

The term of finite element analysis (FEA) was proposed by Clough in 1960. The conception of FEA is that an actual object can be divided into many elements and nodes which can be general called mesh. Each element should follow basic mechanical formulas which can be expressed as a matrix. Afterwards, some boundary conditions such as loads and constraints will be inputted into the matrix to obtain the displacements and stresses of the element. Thus, FEA is very useful in scientific research and engineering design fields because it has the advantages of economic and efficiency. Basically, the FEA analysis software is developed to solve the aforementioned matrix. Figure 3.12 showed a general flow chart of FEA. However, the simulation results would still have certain unavoidable errors. The error source may be attributed as the following causes [40]:

1. The difference between simulation model and actual body.
2. Un-appropriate finite element numerical analysis.
3. Mechanics concept or software operation mistakes.

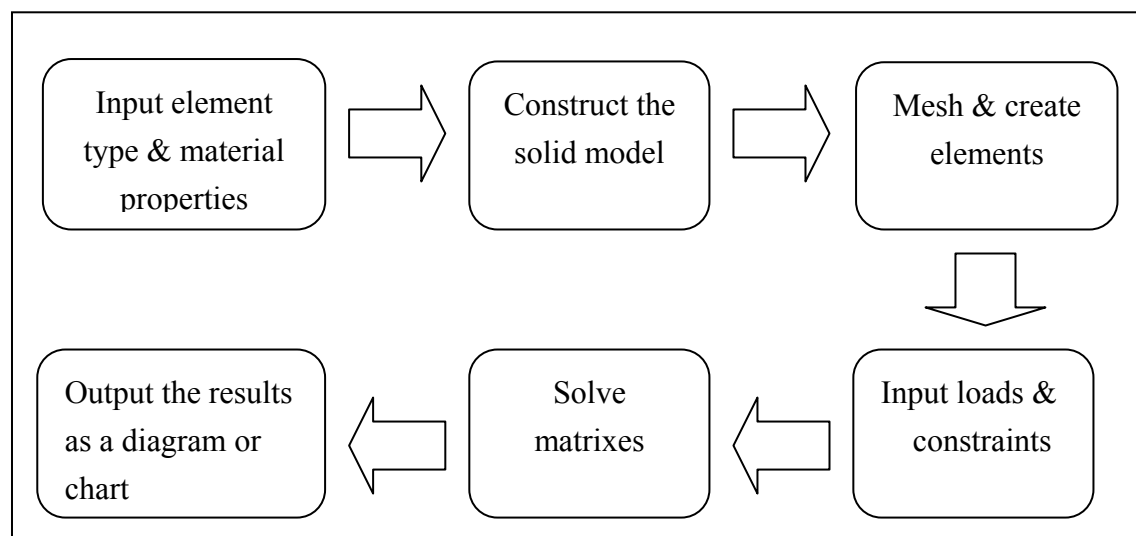


Figure 3.12 The flow chart of FEA

3.2.2 The governing equations for linear elastic material

For a three-dimension structural analysis, we chose the displacement (u), strain (ε), and stress (σ) as the unknown values. Figure 3.13 revealed the stress vectors of a 3-D solid.

$$\begin{aligned} u &= \{u_x \quad u_y \quad u_z\} \\ \varepsilon &= \{\varepsilon_x \quad \varepsilon_y \quad \varepsilon_z \quad \varepsilon_{xy} \quad \varepsilon_{yz} \quad \varepsilon_{zx}\} \\ \sigma &= \{\sigma_x \quad \sigma_y \quad \sigma_z \quad \sigma_{xy} \quad \sigma_{yz} \quad \sigma_{zx}\} \end{aligned}$$

For linear elastic material, the relationship between strain and stress should obey the Hooke's law:

$$\sigma = [E]\varepsilon_e \quad (3.15)$$

where E is elastic matrix.

If we consider a thermo-mechanical deformation:

$$\varepsilon = \varepsilon_e + \varepsilon_{th} \quad (3.16)$$

$$\varepsilon_{th} = \{\alpha_x \quad \alpha_y \quad \alpha_z \quad \alpha_{xy} \quad \alpha_{yz} \quad \alpha_{zx}\} \Delta T$$

where ε_e represents the elastic strain, ε_{th} means the thermal strain, and α is the coefficient of thermal expansion.

$$\Delta T = T_{ref} - T$$

where T_{ref} is the reference temperature, and T is the analysis temperature.

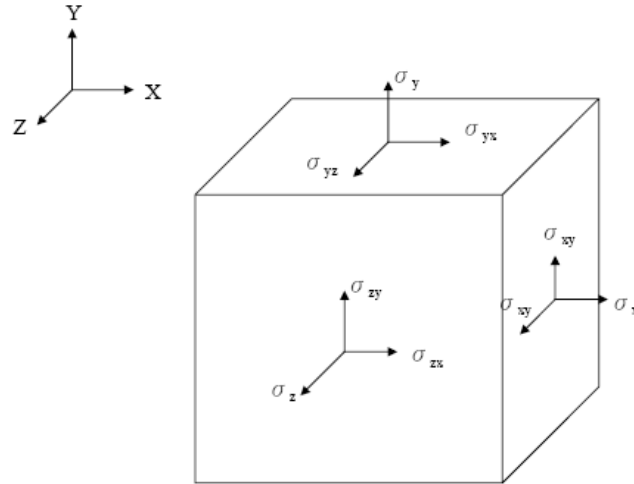


Figure 3.13 The stress vectors of a 3-D solid

From Eq. (3.15) and (3.16), we can get the following equation as:

$$\boldsymbol{\varepsilon} = \boldsymbol{\sigma}[\boldsymbol{E}]^{-1} + \boldsymbol{\varepsilon}_{th} \quad (3.17)$$

where,

$$[\boldsymbol{E}]^{-1} = \begin{bmatrix} \frac{1}{E_x} & -\nu_{xy} & -\nu_{xz} & 0 & 0 & 0 \\ \frac{-\nu_{yx}}{E_x} & \frac{1}{E_y} & -\nu_{yz} & 0 & 0 & 0 \\ \frac{-\nu_{zx}}{E_x} & \frac{-\nu_{zy}}{E_y} & \frac{1}{E_z} & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{G_{xy}} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{G_{yz}} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{G_{zx}} \end{bmatrix}$$

where, E_x, E_y, E_z represent the elastic modulus in x, y, z direction, respectively, ν is the Poisson's ratio, and G is the shear modulus.

In elastic deformation region, the relationship between E , ν , and G can be expressed as:

$$G = \frac{E}{2(1 + \nu)} \quad (3.18)$$

For isotropic materials:

$$E = E_x = E_y = E_z$$

$$G = G_{xy} = G_{yz} = G_{zx}$$

$$\nu = \nu_{xy} = \nu_{yz} = \nu_{zx} = \nu_{yx} = \nu_{zy} = \nu_{xz}$$

$$\alpha = \alpha_x = \alpha_y = \alpha_z$$

Thus, Eq. (3.17) can be re-written as:

$$\boldsymbol{\varepsilon} = [\mathbf{E}]^{-1} \boldsymbol{\sigma} + \alpha \Delta T \quad (3.19)$$

The normal and plane strains can be determined as:

$$\varepsilon_x = \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E} + \alpha \Delta T$$

$$\varepsilon_y = -\nu \frac{\sigma_x}{E} + \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E} + \alpha \Delta T$$

$$\varepsilon_z = -\nu \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} + \frac{\sigma_z}{E} + \alpha \Delta T$$

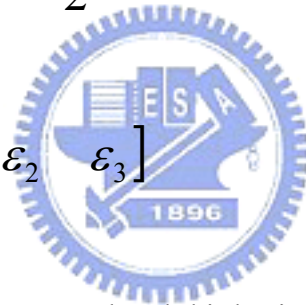
$$\varepsilon_{xy} = \frac{\sigma_{xy}}{G}$$

$$\begin{aligned}\varepsilon_{yz} &= \frac{\sigma_{yz}}{G} \\ \varepsilon_{zx} &= \frac{\sigma_{zx}}{G}\end{aligned}\tag{3.20}$$

The principal strain (ε_p) can be determined as:

$$\begin{vmatrix} (\varepsilon_x - \varepsilon_p) & \frac{1}{2}\varepsilon_{xy} & \frac{1}{2}\varepsilon_{xz} \\ \frac{1}{2}\varepsilon_{xy} & (\varepsilon_y - \varepsilon_p) & \frac{1}{2}\varepsilon_{yz} \\ \frac{1}{2}\varepsilon_{xz} & \frac{1}{2}\varepsilon_{yz} & (\varepsilon_z - \varepsilon_p) \end{vmatrix} = 0$$

$$\varepsilon_p = [\varepsilon_1 \quad \varepsilon_2 \quad \varepsilon_3]$$



where $\varepsilon_1, \varepsilon_2, \varepsilon_3$ is the first, second and third principal stress, respectively. And $\varepsilon_1 > \varepsilon_2 > \varepsilon_3$.

The von Mises strain (ε_E) can be repressed as:

$$\varepsilon_E = \frac{1}{1 + \nu_E} \left\{ \frac{1}{2} \left[(\varepsilon_1 - \varepsilon_2)^2 + (\varepsilon_2 - \varepsilon_3)^2 + (\varepsilon_3 - \varepsilon_1)^2 \right] \right\}^{\frac{1}{2}}$$

where ν_E is the effective Possion's ratio.

The principal stress (σ_p) can also be determined as:

$$\begin{vmatrix} (\sigma_x - \sigma_p) & \sigma_{xy} & \sigma_{xz} \\ \sigma_{xy} & (\sigma_y - \sigma_p) & \sigma_{yz} \\ \sigma_{xz} & \sigma_{yz} & (\sigma_z - \sigma_p) \end{vmatrix} = 0$$

$$\sigma_p = [\sigma_1 \quad \sigma_2 \quad \sigma_3]$$

The von Mises stress (σ_E) can also be repressed as:

$$\sigma_E = \left\{ \frac{1}{2} [(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2] \right\}^{\frac{1}{2}}$$



3.3 Sample preparation for Moiré interferometry

The exterior of our experimental FC-BGA packaging sample with 40 x 40 mm² dimensions was shown in Fig. 3.14(a). In this study, the thermal deformation of FC-BGA assemblies with UF-1 and UF-2 were measured and compared by Moiré interferometry. The solder bumps in these samples were Sn0.7Cu alloy with a pitch of 200 µm.

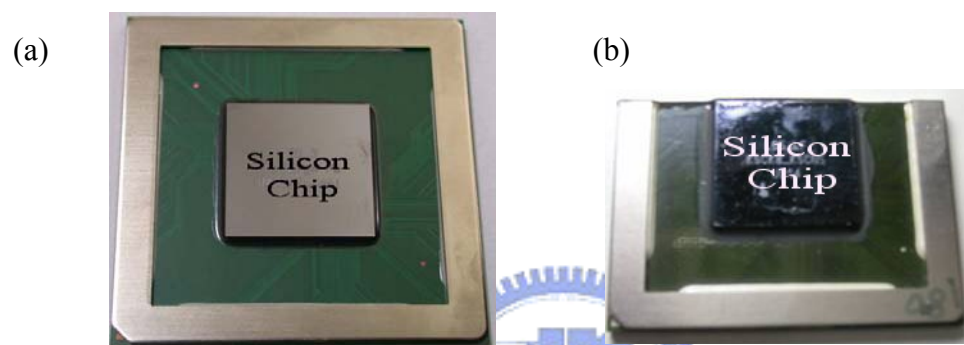


Figure 3.14 The experimental FC-BGA assembly (a) before and (b) after cutting

The procedure of sample preparation for Moiré interferometry was schematically illustrated in Fig. 3.15 and described in details as below:

1. The samples were cut by a low speed diamond saw and polished with 1200 grid abrasive papers to the cross-section of the first bump row as shown in Fig. 3.13(b).
2. The cross-section image was observed and captured by the optical microscopy for superposition onto high resolution Moiré images.
3. Both the polished assembly and grating were placed in an oven at 85 °C as the zero-displacement reference state. A TRA-BondTM F253 epoxy was chosen as the adhesive. The adhesive was spread out smoothly by optical tissues to control the thickness of adhesive. Afterwards, the cross-section

of the assembly was attached to the thin grating in the oven for an hour.

4. The assembly was pried off from the grating mold carefully after curing process.

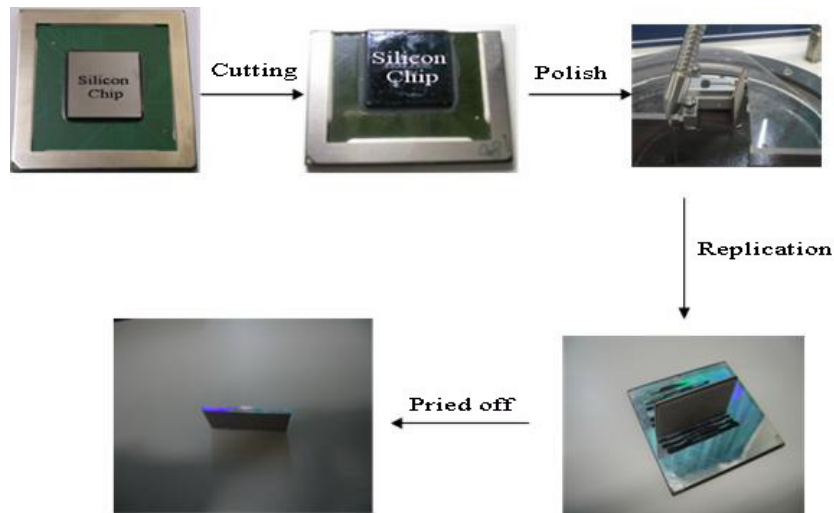


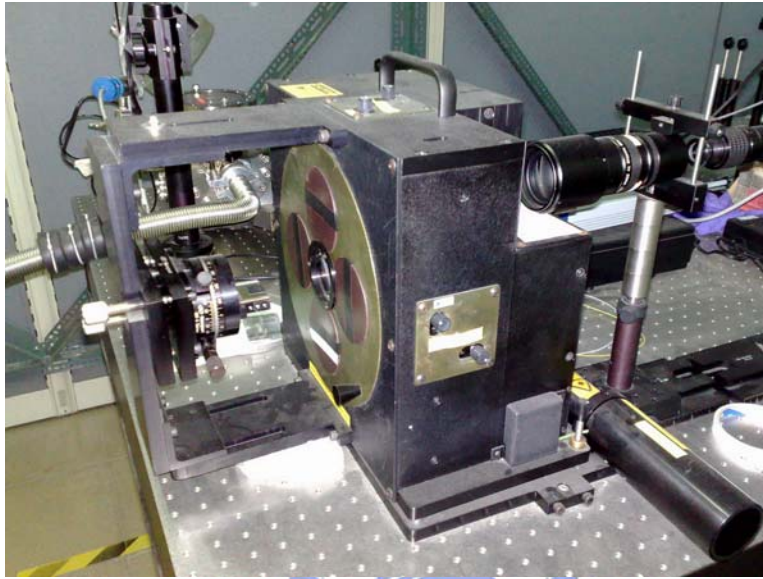
Figure 3.15 The procedure of sample preparation for Moiré interferometry

The Portable Engineering Moiré interferometer (PEMI) from IBM Corp. was employed for this study. To achieve sensitivity up to 26 nm, the PEMI system was upgraded to add a piezoelectric transducer (PZT) part to a high-resolution Moiré interferometer. The reference grating was controlled to shift 147 nm (I_2), 295 nm (I_3), and 441 nm (I_4) relative to I_1 image by the PZT. The experimental equipment was placed on an optical table to avoid the vibration issue. Figures 3.16 and 3.17 showed our high resolution Moiré system and the PZT part, respectively.

The in-plane thermal deformation of assemblies was measured by Moiré interferometry at room temperature (25 °C). This represented a -60 °C thermal loading applied in the assemblies. The reflective mirrors were tune to produce the null field by the original un-deformed grating before the experimental measurement. After the calibration, we could obtain a reference grating with 2400 lines/mm frequency.

The fringe patterns were captured by a 1.3 M pixel CCD camera with 12-bit grayscale resolution. A program developed by UT-Austin group was used to analyze the four continuous fringe patterns obtained by high resolution Moiré interferometer [39].

(a)



(b)

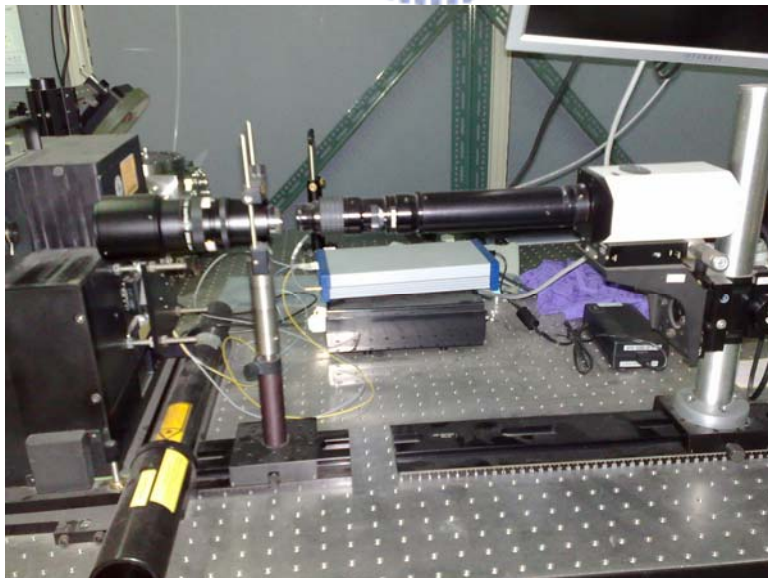


Figure 3.16 (a) The main body of PEMI (b) high-magnification lens

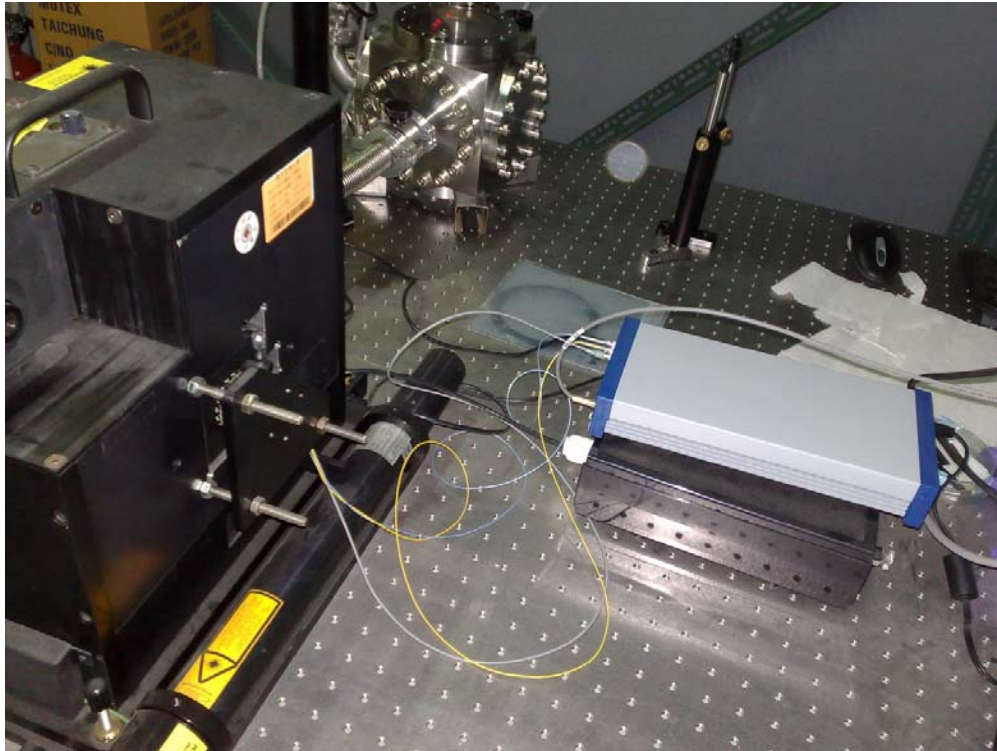


Figure 3.17 The PZT part and its controller

In addition, the thermo-mechanical reliability of six kinds of FC-BGA assemblies with different underfill materials and bump alloys were evaluated by thermal fatigue tests. These assemblies underwent the JEDEC level 3 precondition (260 °C), which described in details in Table 3.1 and TCT 1000 cycles (from -55 to 125 °C). The thermal reliability test results would compare with those by stress prediction results using FEA.

Table 3.1 JEDEC precondition level 3

JEDEC precondition level 3		
Item	Test status	Time/Cycles
Bake	125 °C	24 Hours
Moisture Soak	60 °C/60%RH	40 Hours
IR reflow (Sn37Pb)	240 °C (peak)	3 X
IR reflow (Pb-free)	260 °C (peak)	3 X

3.4 Simulation model and basic assumptions

A simulation model using ANSYSTM program was employed to predict the die warpage and stress distribution. The dimensions and material properties of key components in this study which were provided by UMC Corp. are listed in Table 3.2. In this study, the finite elements model had to be established as a 3-D shape because the square heat sink ring of the FC-BGA packages need to be considered. Since the packaging samples were symmetric assemblies, a 1/2 model was used for the cut assembly as shown in Fig. 3.18 (a). The simulation results will then compare with the measurement results by Moiré interferometry. In addition, a 3-D 1/4 model with boundary conditions was created for full assembly as shown in Fig. 3.18 (b) to predict the stress distribution for one TCT cycle. To reduce elements and calculation time, only 10 rows of bumps near the sectioned plane were established. The accuracy of 1/2 symmetric model was validated by the experimental data from Moiré. Since the accurate non-linear material properties and adhesion strength of each component were hardly obtained and a great element numbers (~ 300,000) of the 3-D finite elements model, it was difficult to solve non-linear matrices for TCT 1000 cycles. However, the thermal induced stress of one TCT cycle could be used as an index of delamination potential for qualitative analysis. Therefore, a 1/4 model was used to predict the stress distribution after one TCT cycle from 125 to -55 °C. The maximum stress value in the outmost bump and lower layer would be compared among various samples. The element type solid 185 of ANSYSTM was used in this study. It was defined by eight nodes, and each node has three degrees of freedom (UX, UY and UZ).

In addition, the following assumptions were applied:

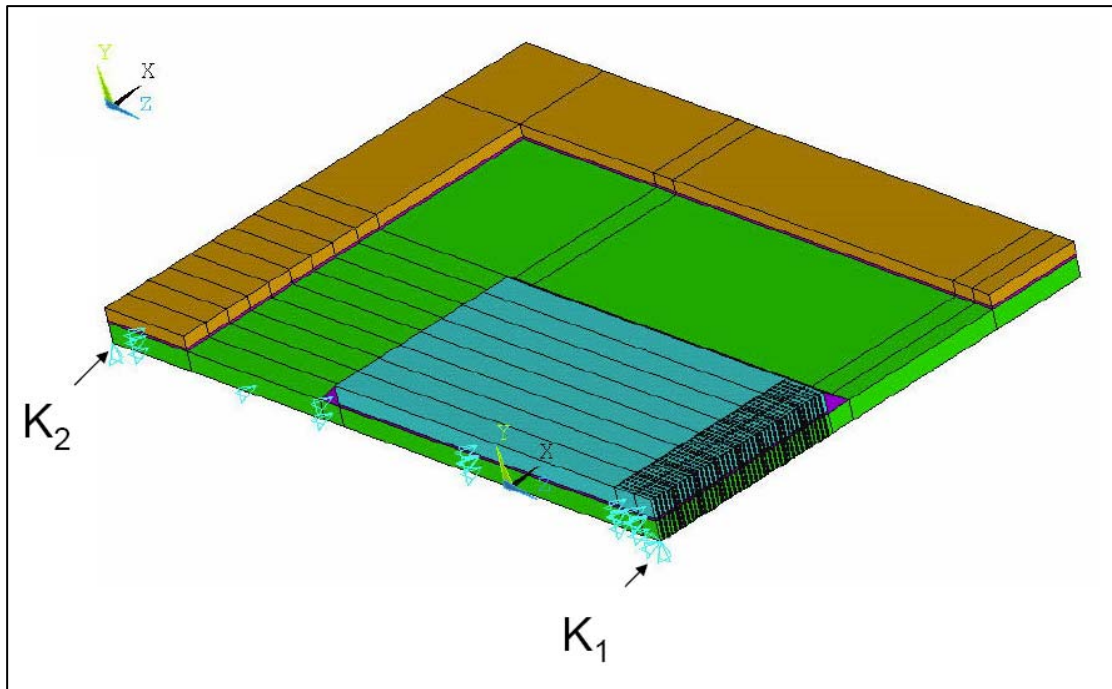
1. All components were homogeneous and linear elastic materials.

2. All interfaces have perfect adhesion to each other.
3. For 1/2 model, the planes at $X=0$ was fixed in X direction, keypoint K_1 was fixed in all direction and keypoint K_2 was fixed in Y direction.
4. For 1/4 model, the original point K_0 was fixed in all direction, the planes at $X=0$, and $Z=0$ were set as symmetric boundary.
5. For the 1/4 model, the reference temperature was set as $125\text{ }^\circ\text{C}$, and analysis temperature was from $125\text{ }^\circ\text{C}$ cooled down to $-55\text{ }^\circ\text{C}$.

Table 3.2 The dimensions and material properties of the FCBGA assemblies (sources: UMC)

Material	Properties				
	Thickness (mm)	Young's Modulus (kg/mm ²)	CTE(ppm/ $^\circ\text{C}$)	ν	Tg($^\circ\text{C}$)
Die (16.35*16.35 mm)	0.75	16000	2.8	0.3	-
Underfill 1	0.1	E1=826.5, E2=30.6	CTE1=26 CTE2=91	0.35	125
Underfill 2		E1=969, E2=11	CTE1=27 CTE2=92	0.35	100
Underfill 3		E1=800, E2=4.7	CTE1=32 CTE2=102	0.35	80
Underfill 4		E1=700, E2=4.7	CTE1=32 CTE2=110	0.35	70
Sn0.7Cu		2600	22	0.35	-
Sn37Pb		2730	23.5	0.35	-
Sn95Pb		2388	29.1	0.35	-
BT Core	0.8	2451	X=Y=14, Z=58	0.28	-
Heat Sink, Cu	Width=4, thick=0.5	12100	16.3	0.3	-
Glue	0.1	700	CTE1=48 CTE2=99	0.35	75

(a)



(b)

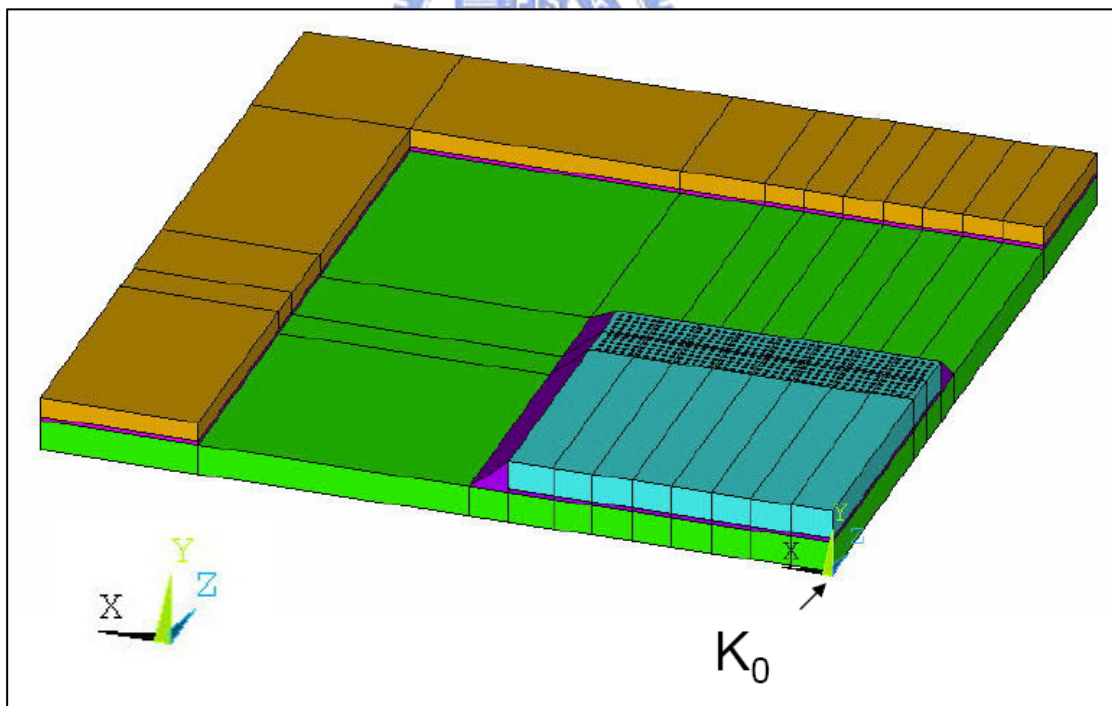


Figure 3.18 The 3D finite element models. (a) 1/2 symmetry (b) 1/4 symmetry

Chapter 4 Results and Discussion

4.1 Measurement results by regular Moiré interferometry

Figures 4.1 (a)-(d) showed the interference images of the packages with UF-1 and UF-2 obtained by regular Moiré interferometry. The grating was replicated onto the cross-section of assemblies at 85 °C, and measured its deformation at 25 °C ($\Delta T = -60$ °C). Each fringe spacing was 0.417 μm . The neutral line was set as a zero deformation reference due to the symmetry of packages. We can readily calculate the relative displacement by counting the number of fringes from the neutral line.

The fringe patterns of U field showed larger compressive strains at the bottom of package and almost zero strain at the top edge of silicon chip. The V field fringe patterns also showed much higher strain gradient at print circuit board than that at silicon chip. It could be attributed to the higher CTE and lower elastic modulus of substrate than silicon chip. It also revealed that the package was under bending after a cooling process. We could observe some zigzag fringes occurrence in substrate region. These zigzag fringes were not caused by the optical noise or operation mistakes during grating replication step, but by the multiple glass epoxy composite PCB.

Based on the Eq. (3.11), *i.e.* $\gamma_{xy} = \frac{1}{f} \left(\frac{\partial N_x}{\partial y} + \frac{\partial N_y}{\partial x} \right)$, the term of $\frac{\partial N_y}{\partial x}$ implied

that the more fringes of Y direction in U field pattern, the larger shear strain was. So was the term of $\frac{\partial N_x}{\partial y}$. From these fringe images, we observed that the fringe gradient increased gradually from the center to the edge of the chip. This represented the maximum shear would occur at the die edge region.

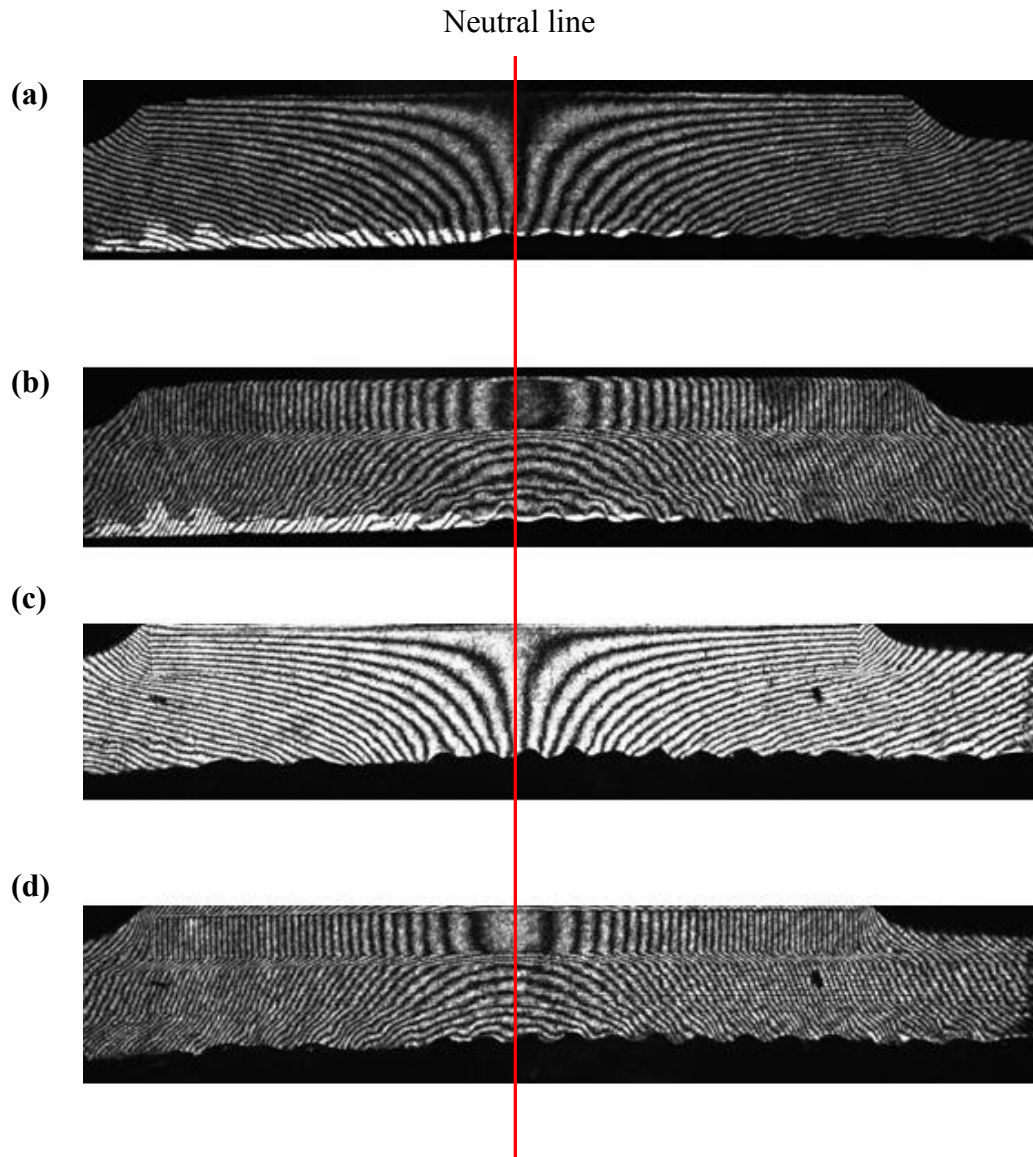


Figure 4.1 Regular Moiré patterns of

(a) U field for UF-1 (b) V field for UF-1 underfill material

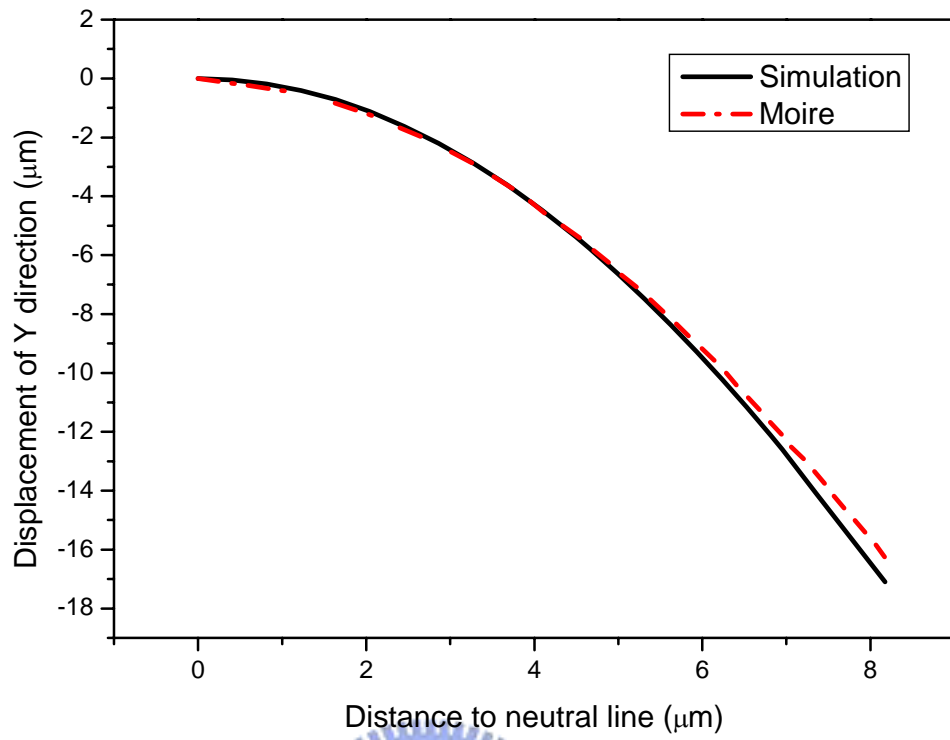
(c) U field for UF-2 (d) U field for UF-2 underfill material

4.2 Comparison between regular Moiré interferometry and simulation

In this study, the Moiré interferometry was employed to verify the accuracy of simulation results. The Y direction displacement from the center to the edge of chip, namely die warpage, was shown in Fig. 4.2. The overall packages were bending downward by the negative value of the displacement of Y direction. The assembly with UF-1 showed smaller die warpage ($\sim 0.2 \mu\text{m}$) than the assembly with UF-2 for a $-60 \text{ }^\circ\text{C}$ thermal loading. It could be attributed to lower elastic modulus of UF-1 underfill material. The compliant underfill material absorbed the contraction force induced by the CTE mismatch between silicon chip and plastic substrate. It indicated that the underfill material with higher elastic modulus would induce larger die warpage.

Table 4.1 summarized the difference between regular Moiré interferometry and simulation on the maximum axial displacements for the assemblies with two different underfill materials. The measurement results showed good agreement with simulation data. The error rates of displacement of V field for the assemblies with UF-1 and UF-2 were 4.97% and 4.75%, respectively. For U field, the error rates were less than 3% for the both assemblies.

(a)



(b)

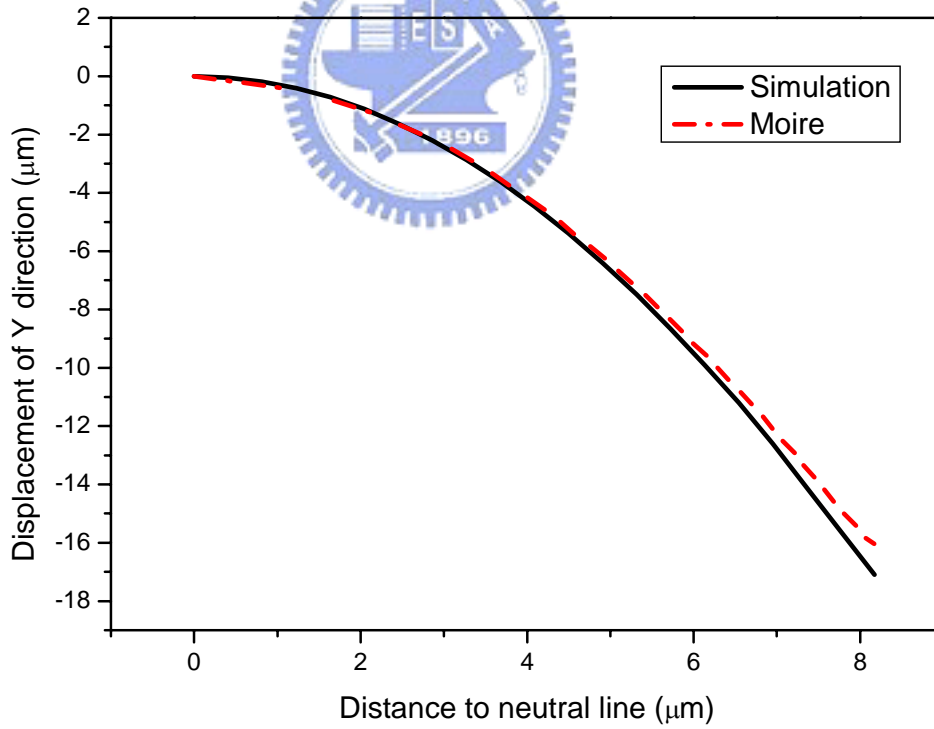


Figure 4.2 The distribution of die warpage for assembly with

(a) UF-1 (b) UF-2 underfill materials

Table 4.1 The difference between regular Moiré interferometry and simulation on the maximum axial displacement

Assembly with	Fringe counts		Displacement (μm)		different rate
			Moiré	Simulation	
UF-1	U field	6	2.50	2.43	2.88%
	V field	38.5	16.06	16.90	4.97%
UF-2	U field	6	2.50	2.43	2.88%
	V field	39	16.26	17.07	4.75%



4.3 Measurement results of high resolution Moiré interferometry

In U field fringe patterns by regular Moiré interferometry, there were only 1-2 fringes in the bump/underfill layer. We could not observe the thermo-mechanical deformation of bumps in detail. The V field fringe patterns by regular Moiré interferometry were similar. Thus, the resolution of regular Moiré interferometry was not enough to measure the thermo-mechanical deformation of solder bumps with 110 μm diameter because the displacement was too small to be resolved. Therefore, a high resolution Moiré interferometry was used. Its spatial resolution was enhanced to 26 nm by phase shifting technology [39]. Such sensitivity would be adequate for resolving the displacement of solder bumps.

Since delaminations or cracks often occurred near die edge, we employed high resolution Moiré interferometry to observe the thermo-mechanical deformation behaviors of the critical region. Figures 4.3 and 4.4 showed the continuous displacement images of U field and V field for the UF-1 package, respectively. The fringes were shifted precisely by the PZT controller. A program developed by Prof. Paul S. Ho's group at the University of Texas at Austin was employed to calculate and analysis the displacement and strains of the two assemblies [39]. The U and V field images would be transformed to a phase contour map by this program. The cross-sectional SEM image of the assembly was superimposed onto the phase contour maps with the help of the obvious turns at die/underfill interface of the contour maps for relating with their relative positions. Figures 4.5 (a)-(b) showed the phase contour maps for U field and V field, respectively. Each fringe space was equal to 208 nm for the contour maps. The contour resolution in Fig. 4.6 was enhanced to 52 nm to help us to study the distribution of thermal induced strain in bump/underfill layer. Figures

4.7 – 4.10 were relative images of the UF-2 assembly.

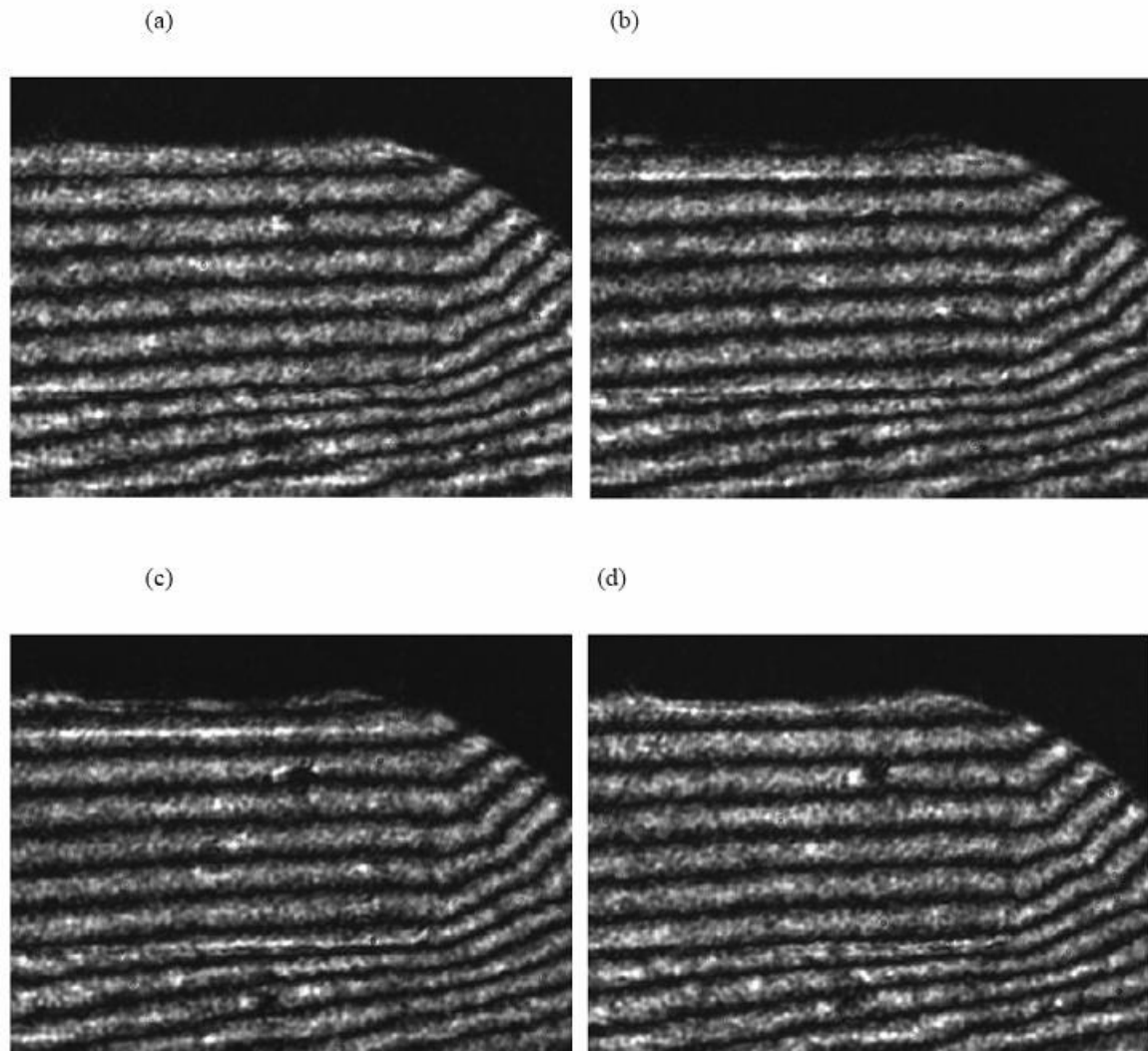


Figure 4.3 The U field continuous displacement images of the package with underfill-1

(a) I_{x1} (b) I_{x2} (c) I_{x3} (d) I_{x4}

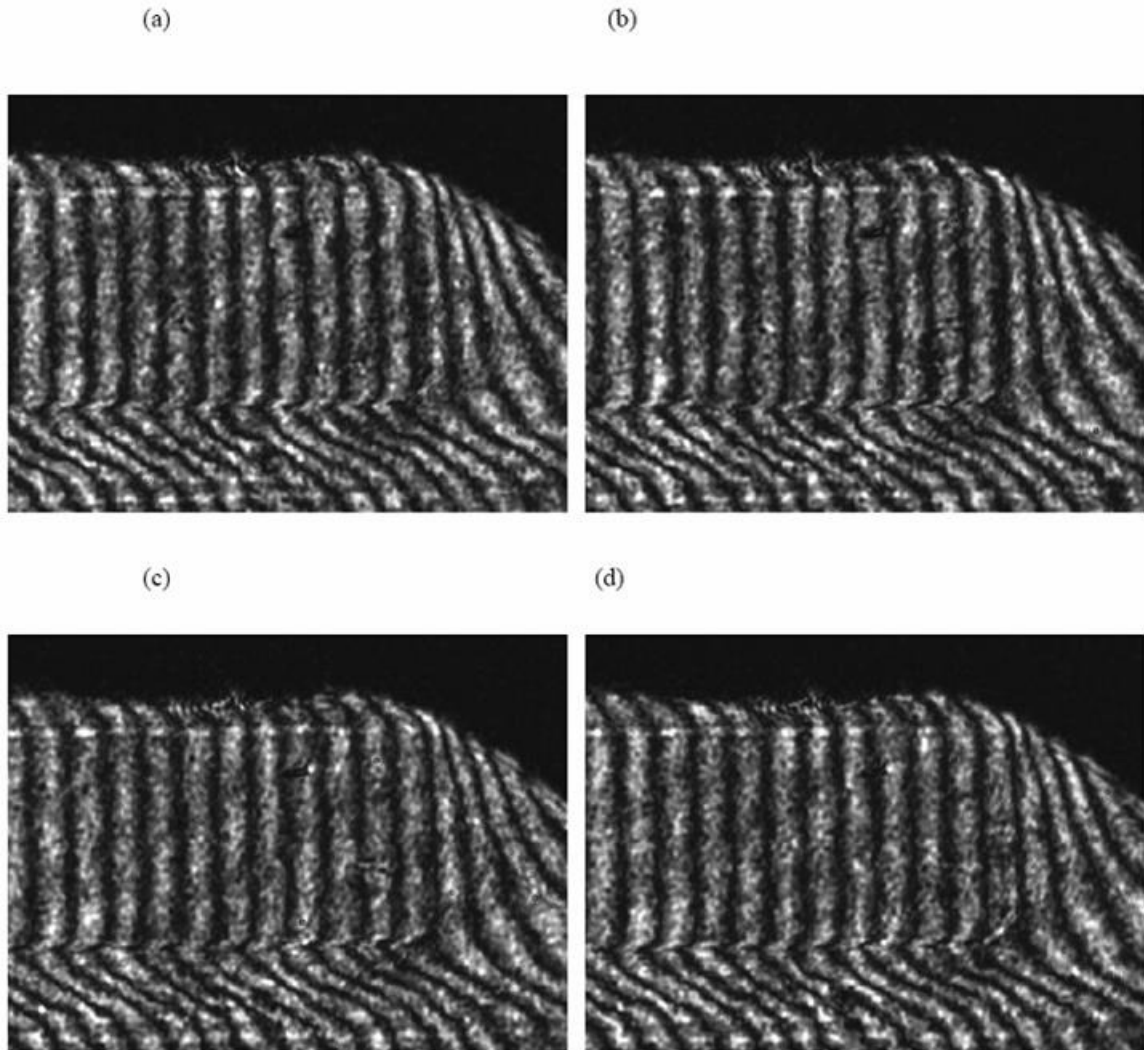
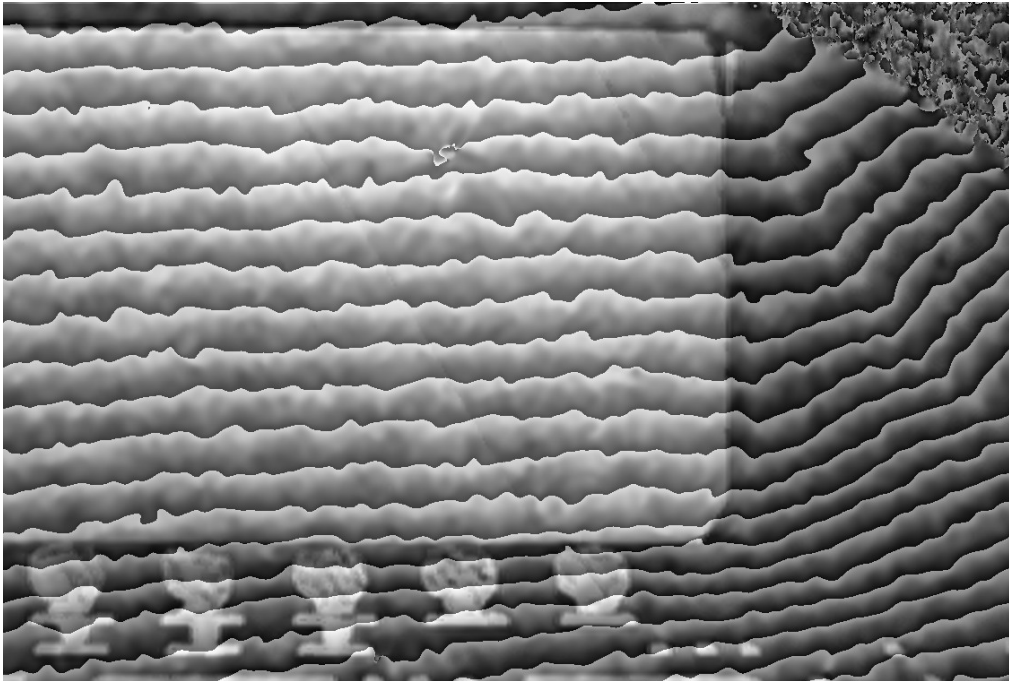


Figure 4.4 The V field continuous displacement images of the package with underfill-1

(a) I_{y1} (b) I_{y2} (c) I_{y3} (d) I_{y4}

(a)



(b)

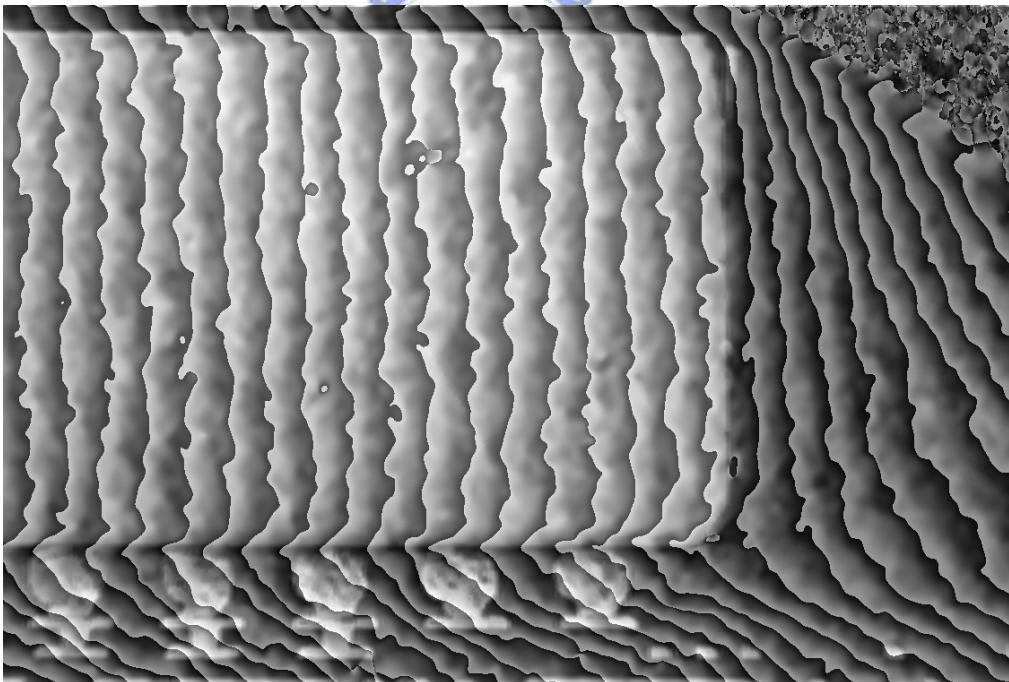
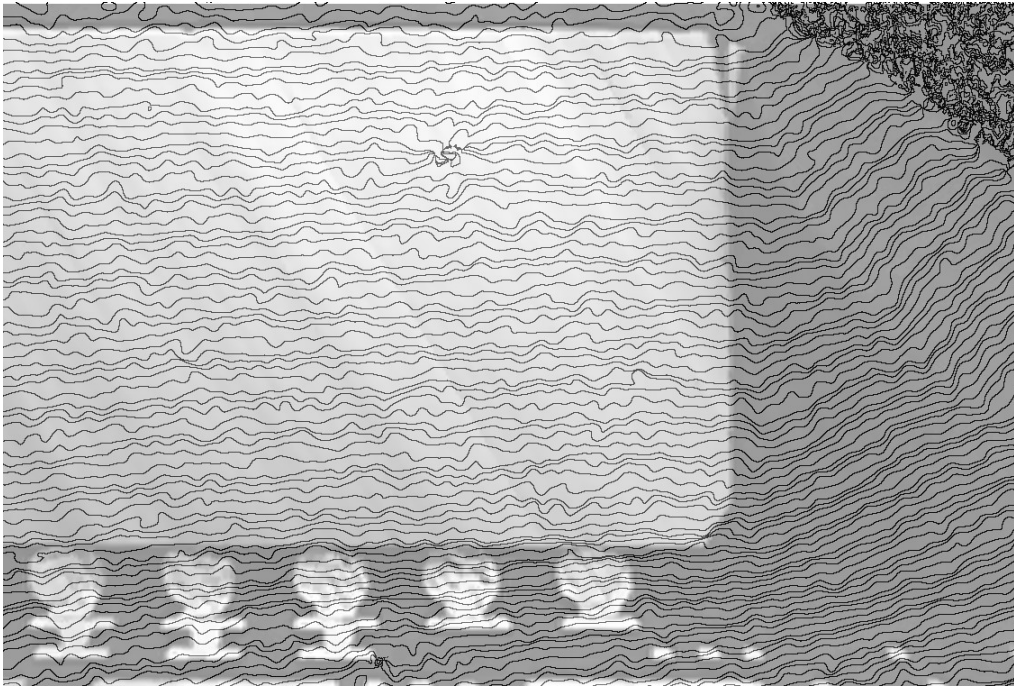


Figure 4.5 The phase contour maps of the package with UF-1 (each fringe spacing = 208nm)

(a) U field (b) V field

(a)



(b)

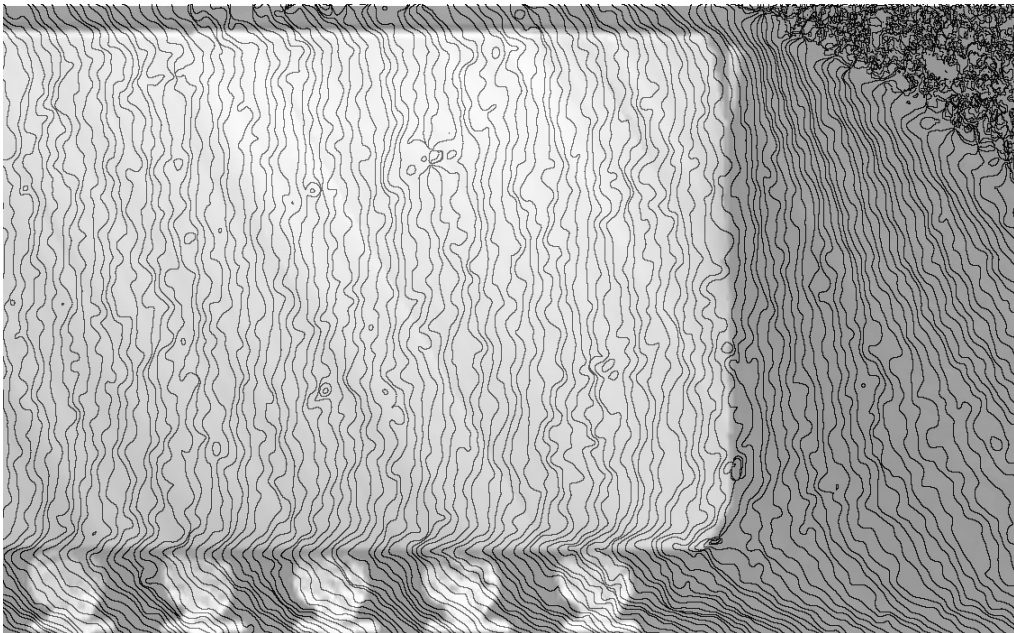


Figure 4.6 The displacement contour maps of the package with underfill-1

(each contour spacing = 52 nm)

(a) U field (b) V field

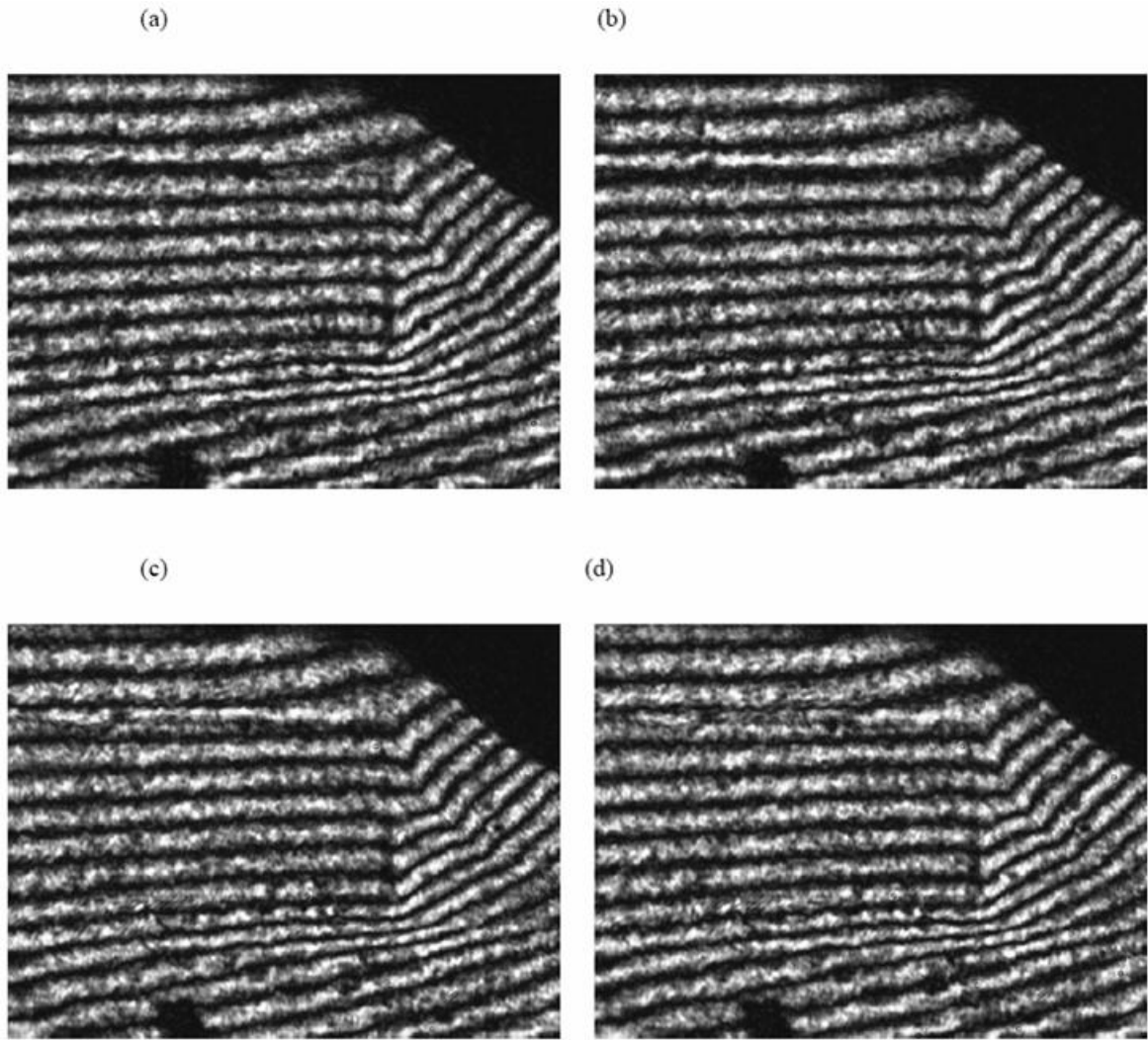


Figure 4.7 The U field continuous displacement images of the package with UF-2 underfill material

(a) I_{x1} (b) I_{x2} (c) I_{x3} (d) I_{x4}

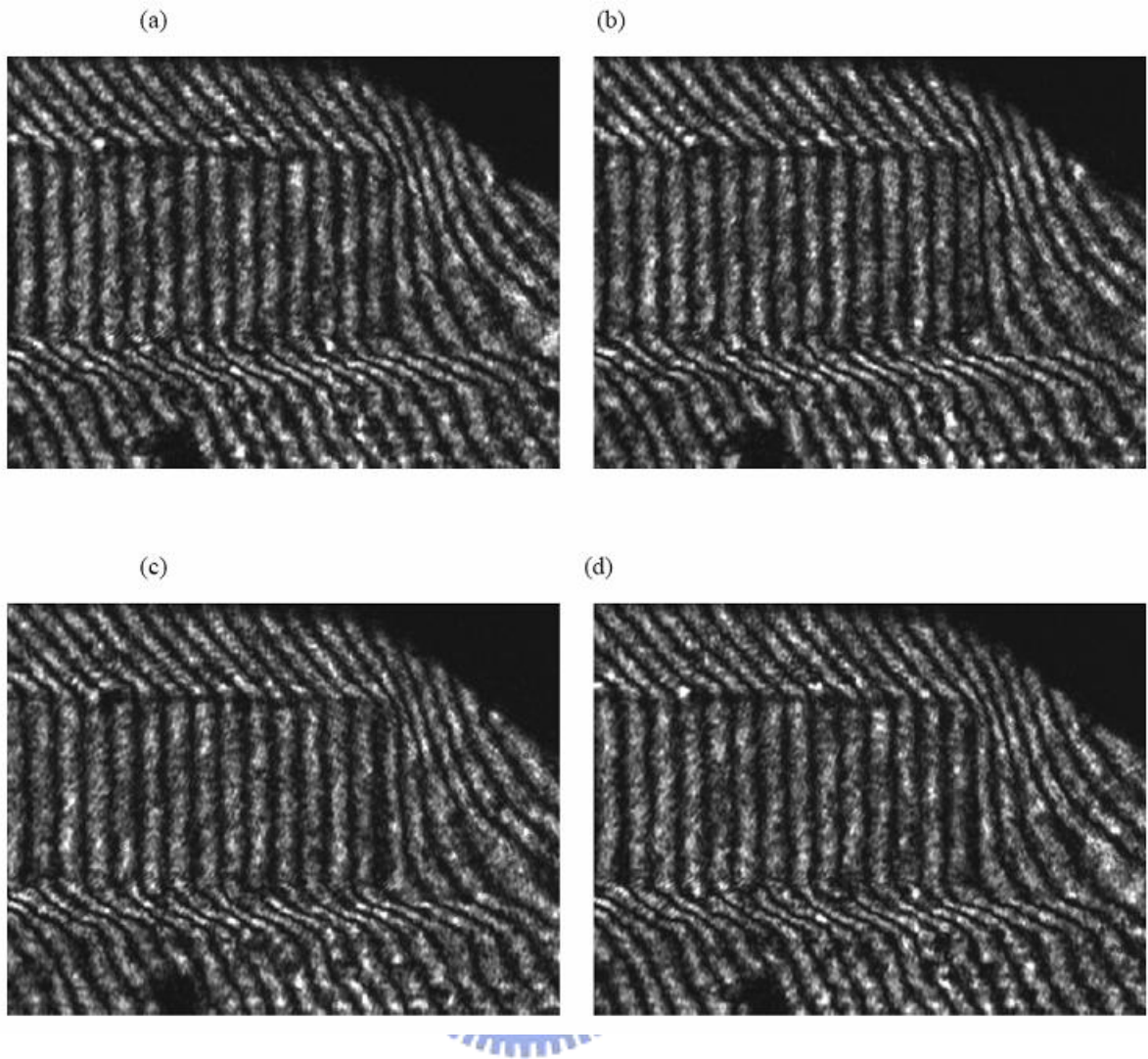
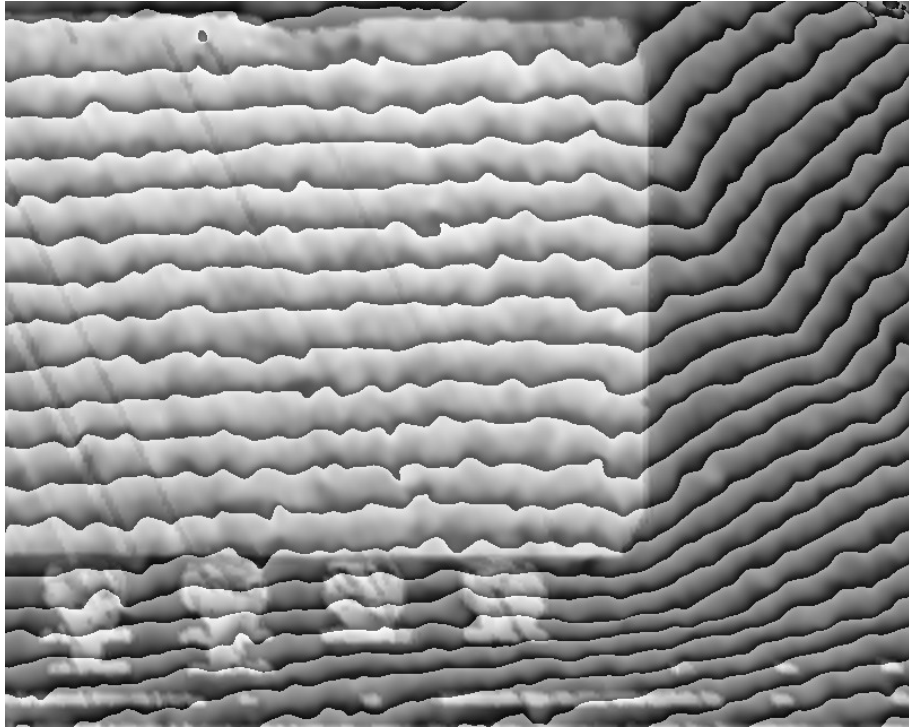


Figure 4.8 The U field continuous displacement images of the package with UF-2 underfill material

(a) I_{y1} (b) I_{y2} (c) I_{y3} (d) I_{y4}

(a)



(b)

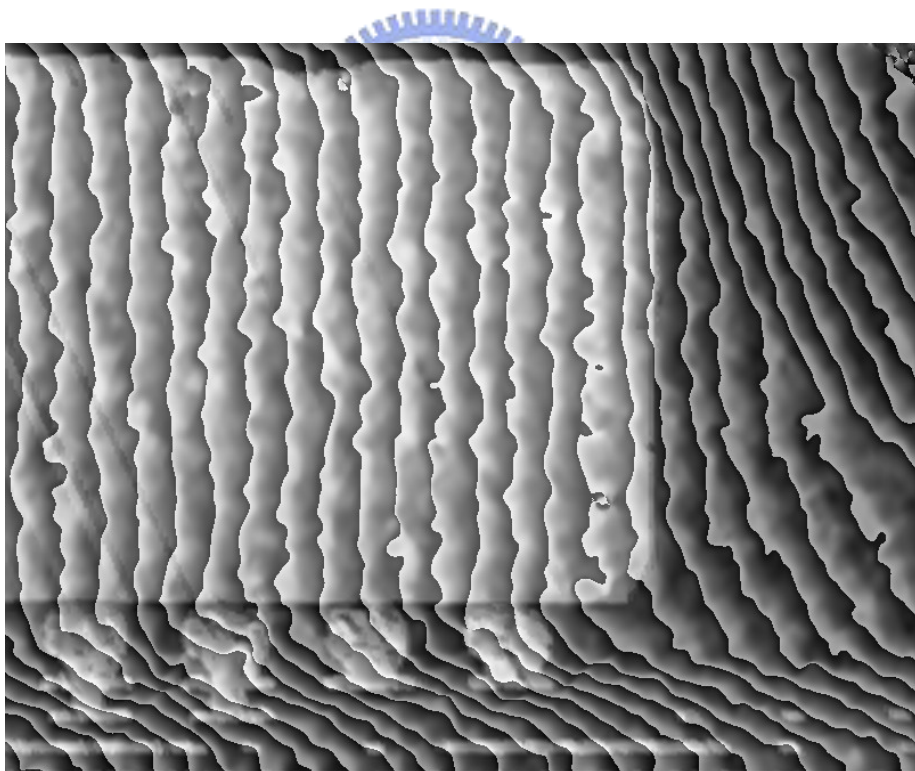
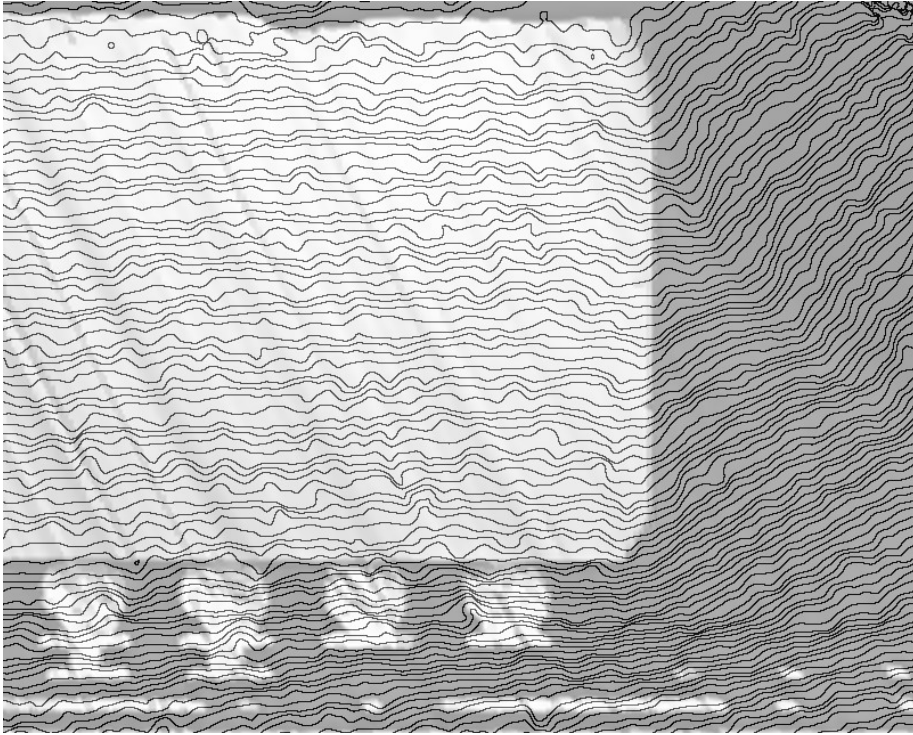


Figure 4.9 The phase contour maps of the package with UF-2 underfill materials

(each fringe spacing = 208nm)

(a) U field (b) V field

(a)



(b)

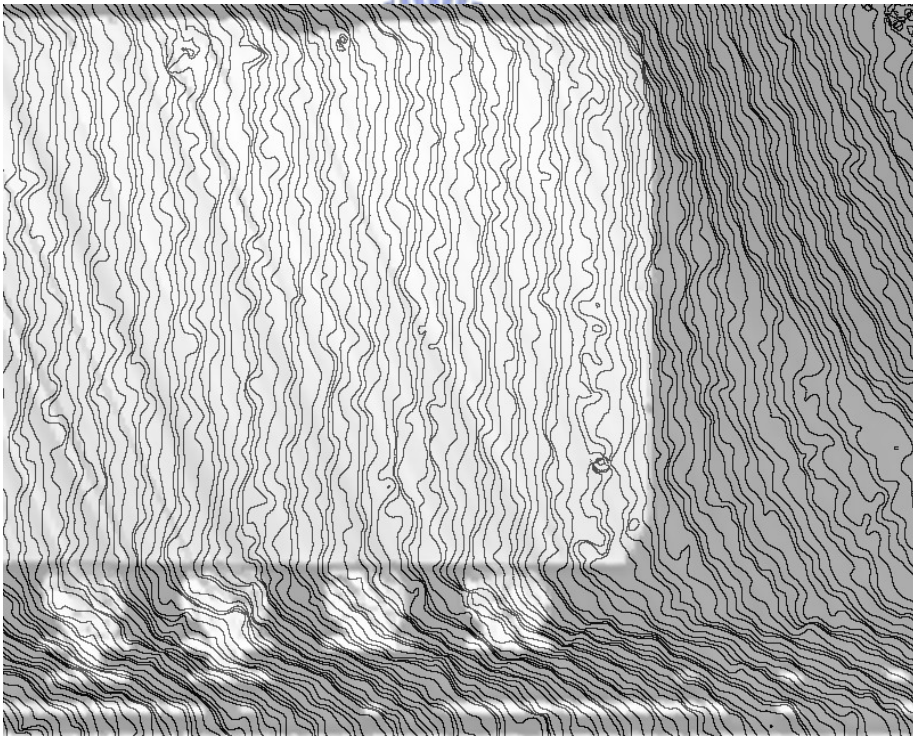
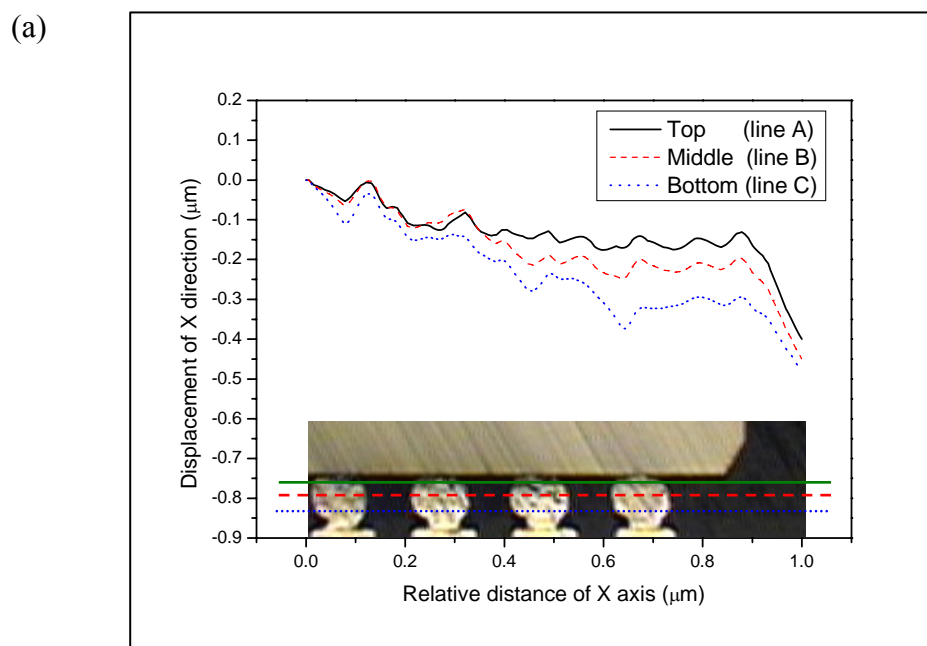


Figure 4.10 The phase contour maps of the package with underfill-2

(each contour spacing = 52nm)

(a) U field (b) V field

From the U field phase contour map, the undulate contour lines could be observed in underfill layer due to different mechanical properties between bumps and underfill material. These contour maps could transform to the axial displacement data for the two assemblies by the Moiré analysis program. In order to study the thermal deformation at different position of bumps, we extracted displacement data along 3 lines as: Top (line A), middle (line B) and bottom (line C) of bumps. Figures 4.11 (a) and (b) showed the X and Y direction displacement difference of UF-2 assembly between the 3 lines, respectively. From Fig. 4.11(a), the X displacement along line A was observed as smallest. Since the silicon die ($\alpha=2.6 \text{ ppm}/^\circ\text{C}$) had smaller CTE than Sn0.7Cu bumps ($\alpha=22 \text{ ppm}/^\circ\text{C}$) and UF-2 material ($\alpha=27 \text{ ppm}/^\circ\text{C}$), the X direction displacement along line A in bump/underfill layer would be constrained by silicon chip. From Fig. 4.11(b), the Y displacement along line A had largest Y direction displacement. Since the lateral multi-structure FC-BGA assembly was under bending, the underfill layer and BT substrate would deform easily due to their lower elastic modulus. Therefore, the Y displacement along line A would be slightly larger than which along the other two lines.



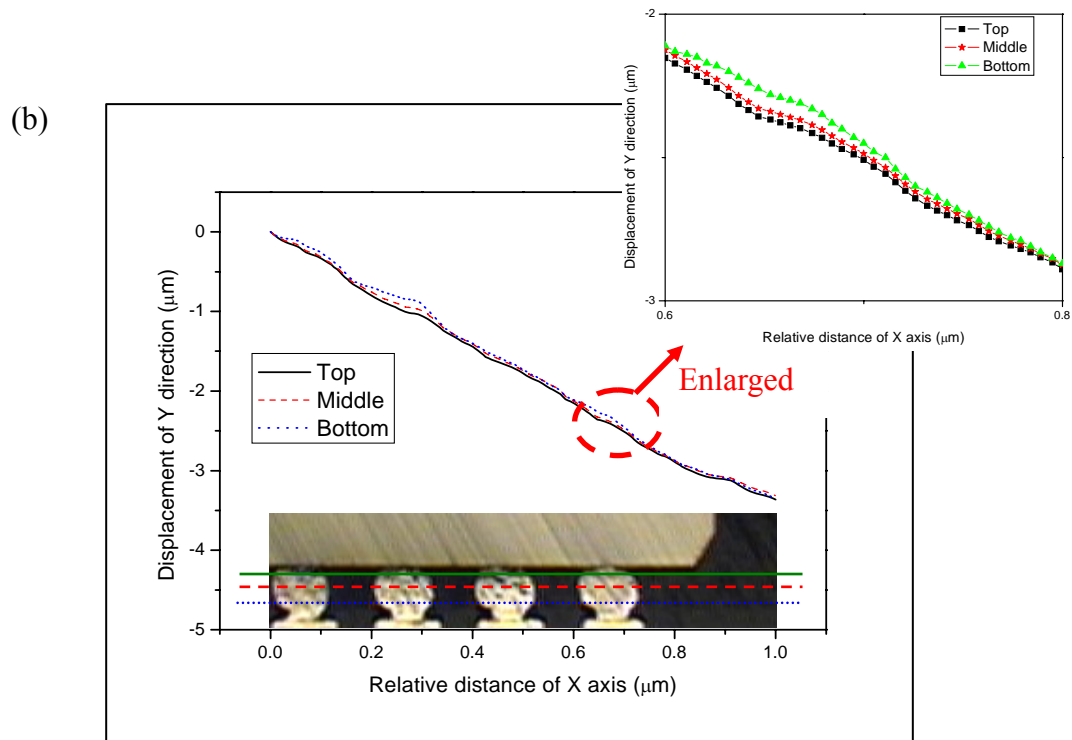


Figure 4.11 (a) X direction (b) Y direction displacement along the three lines in bump/underfill layer

Figure 4.12 revealed the XY plane shear strain along line B was higher than with along line A and C. The obtained XY plane shear strain data was solved by the Eq.

$$(3.11), \gamma_{xy} = \frac{1}{f} \left(\frac{\partial N_x}{\partial y} + \frac{\partial N_y}{\partial x} \right).$$

Since the underfill/bump layer had higher CTE than

the substrate and silicon chip, the upper and lower side of bump would be confined by chip and substrate. Therefore, the XY plane shear strain along line B (middle of bump) would produce a larger thermal induced strain while a thermal loading applied in the package.

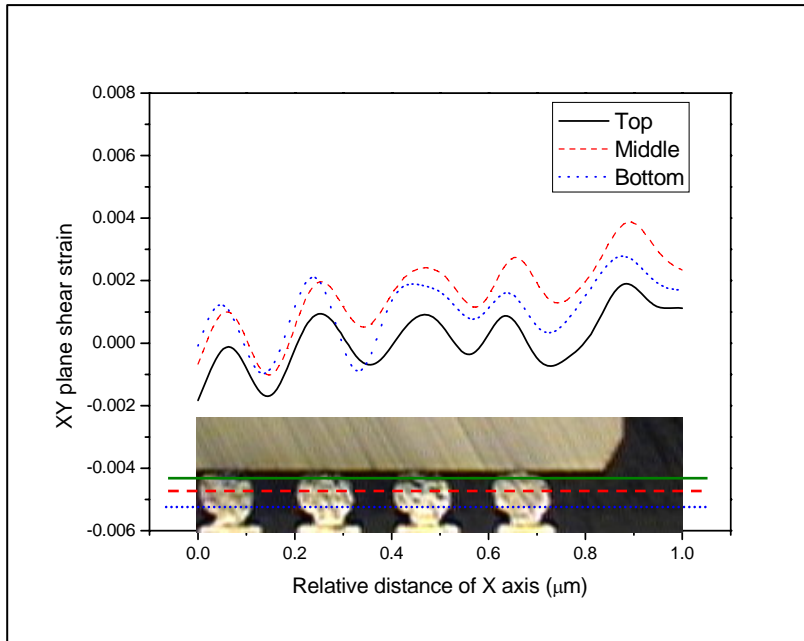
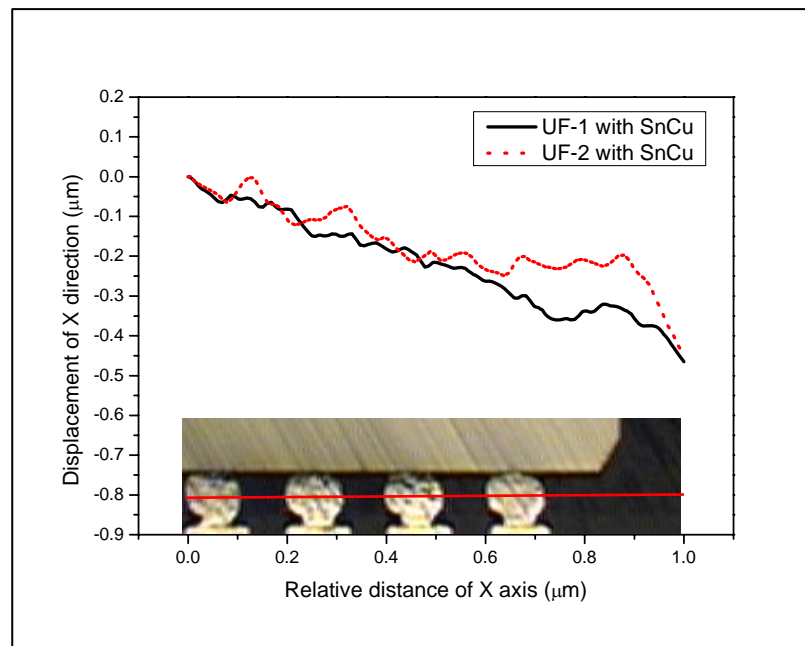


Figure 4.12 The XY shear strain along the three lines in bump/underfill layer

Figures 4.13(a) and 4.13(b) depicted the X and Y displacement along the center of bumps for UF-1 and UF-2 assemblies, respectively. The displacement of V field was larger than U field for both assemblies. This implied the assemblies were under bending deformation. From the U field, the displacement curves dropped sharply in the region which the underfill layer did not exist under the silicon chip because underfill materials had larger CTE than silicon chip. When the region was no longer confined by the silicon chip, the underfill would produce a larger thermal displacement in the region. The X direction displacement of UF-1 assembly was less than that of UF-2 assembly, while Y direction displacement showed the opposite trend. This could be ascribed to UF-2 had higher modulus (969 kg/mm^2) than UF-1 (826.5 kg/mm^2).

(a)



(b)

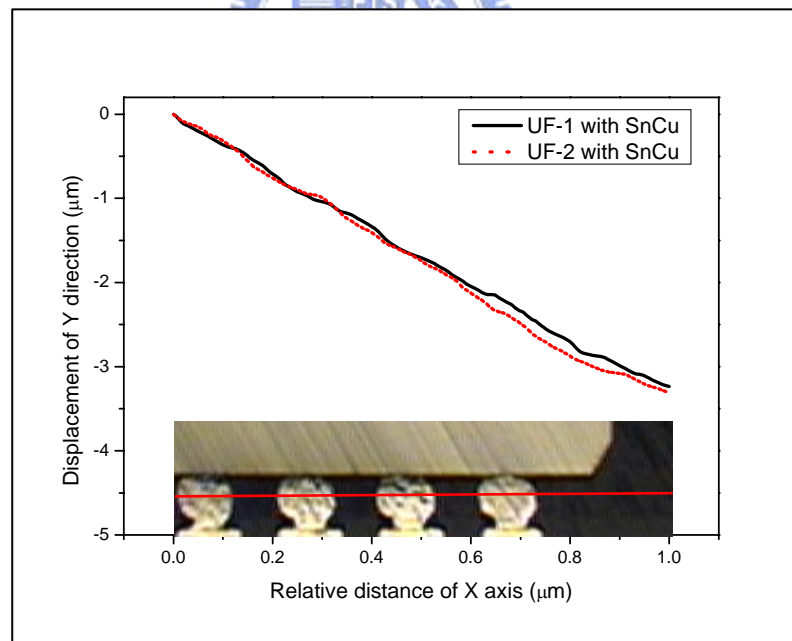


Figure 4.13 The displacement measurement by high resolution Moiré interferometry (a) X direction (b) Y direction

Figure 4.14 showed the shear strain data along the plane of bump center near die edge for the two different underfill materials based on high resolution Moiré measurement. The shear strain obtained from Moiré interferometry was the total strain, *i.e.*, the sum of stress induced strain and thermal strain. The shear strain distributions revealed that the highest shear strain located near the bottom of die edge. This confirmed our previous inferences. The difference between solder bumps and underfill was readily observed. The shear strain increased in solder bump region and decreased in underfill region. This implicated the compliant underfill material mitigated the thermal induced shear stress. Moreover, the outmost solder bump had the highest shear strain compared to other ones. It implied that the outmost solder bump has highest risk of crack or delamination. Since the UF-1 possessed lower modulus than UF-2 material, the bump/underfill would sustain more thermal induced stress. From Fig. 4.14, the larger shear strain of UF-1 was readily observed. This may raise the bump crack potential.

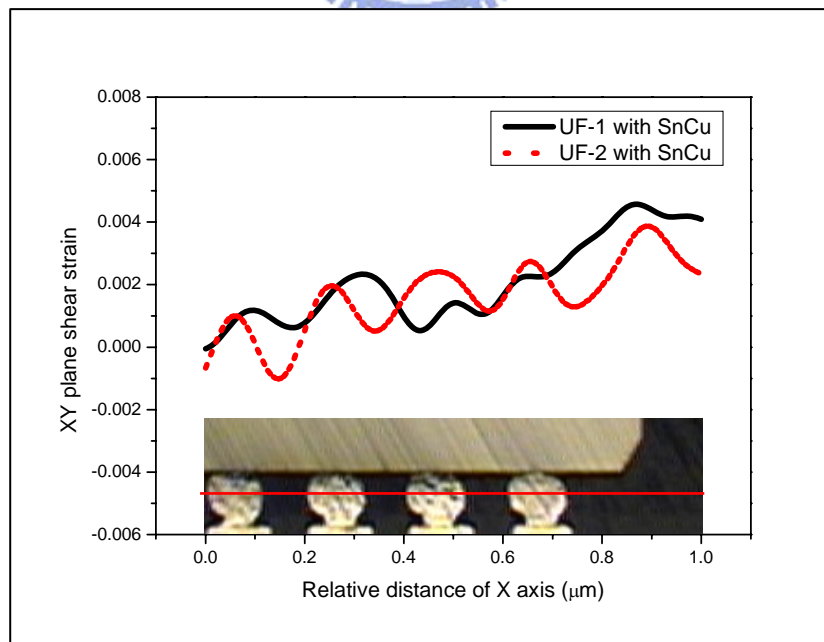


Figure 4.14 The XY plane shear strain results by high resolution Moiré interferometry

4.4 Comparison between TCT results and simulation data

The accuracy of the simulation model had been validated by Moiré interferometry measurement results. However, previous simulation model was not a complete FC-BGA package because the assembly had been cut at plane about 12 mm from the edge of frame. This did not correspond with the actual assemblies. Therefore, a quarter symmetric model was created to predict the thermal induced stresses for different packaging samples, and compared simulation results with actual reliability tests.

Temperature cycling test (TCT) under 125 to -55 °C for 1000 cycles was carried out for the assemblies with UF-1, UF-2 and Sn0.7Cu solder. For comparative study, two more underfill materials (UF-3 and UF-4) with higher CTE but lower modulus and lower Tg temperature were introduced along with eutectic Sn37Pb, high-lead (Sn95Pb) and Sn0.7Cu solders. In total, 6 FC-PGA package assemblies with different underfill and solder bumps underwent the same TCT process. Table 4.2 summarized the temperature cycling test results for these various samples.

Table 4.2 TCT1000 results for various packaging samples with different underfill materials and solder bumps

Item	underfill	solder bump	bump failure	low-K failure
A	UF-4	Sn 95Pb	failed	passed
B	UF-3	Sn 95Pb	passed	
C	UF-3	Sn 37Pb	passed	
D	UF-3	Sn0.7Cu	failed	
E	UF-2	Sn0.7Cu	passed	
F	UF-1	Sn0.7Cu	passed	

Since the bump fracture was caused by shear force mode based on failure analysis, we investigated the failure potential by the XY plane shear stress, σ_{xy} , and von Mises stress, σ_e , at the outmost solder bump. Figure 4.15 showed the stresses distribution of the outmost solder bump in package D. The maximum stresses located at right bottom corner of the out most solder bump and the die edge corner of lay-k layer. This corresponded excellently with the actual crack position shown by SEM viewgraph in Fig. 4.16, kindly provided by UMC corp.

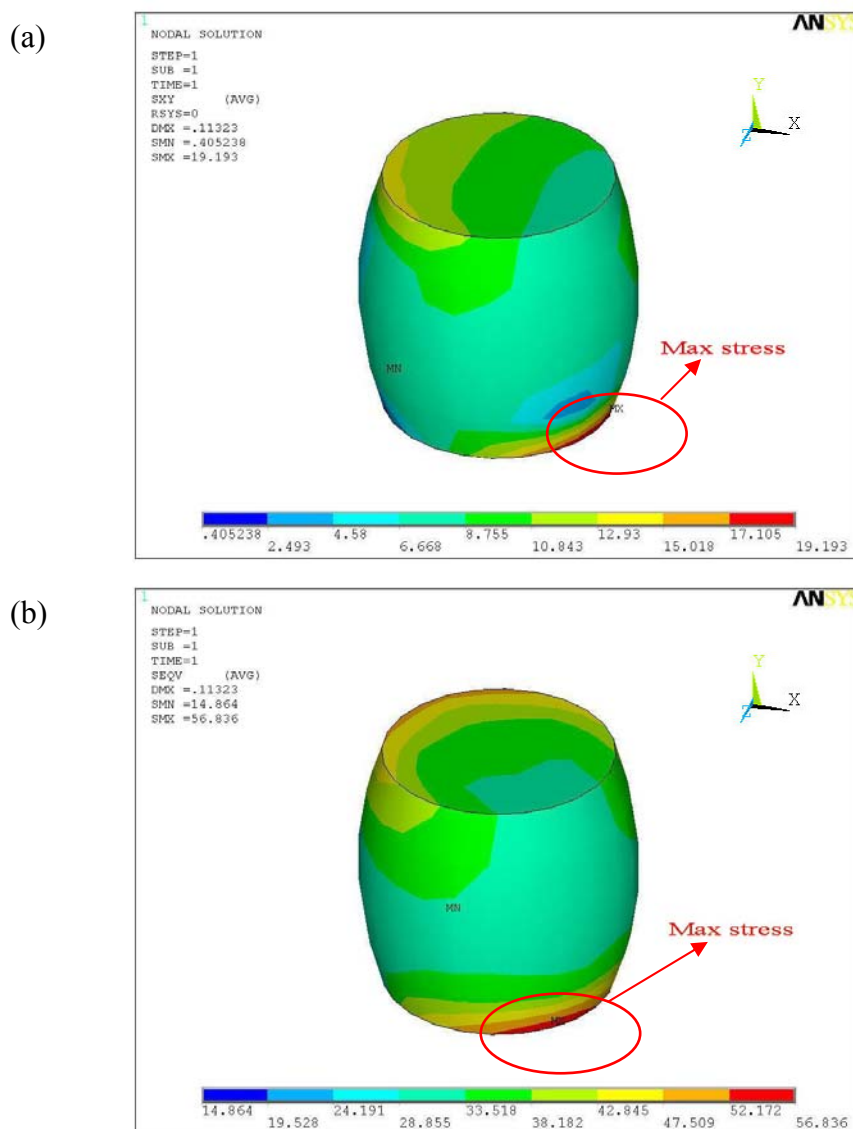


Figure 4.15 The outmost bump stress distribution for sample D

(a) XY plane shear stress (b) von Mises stress

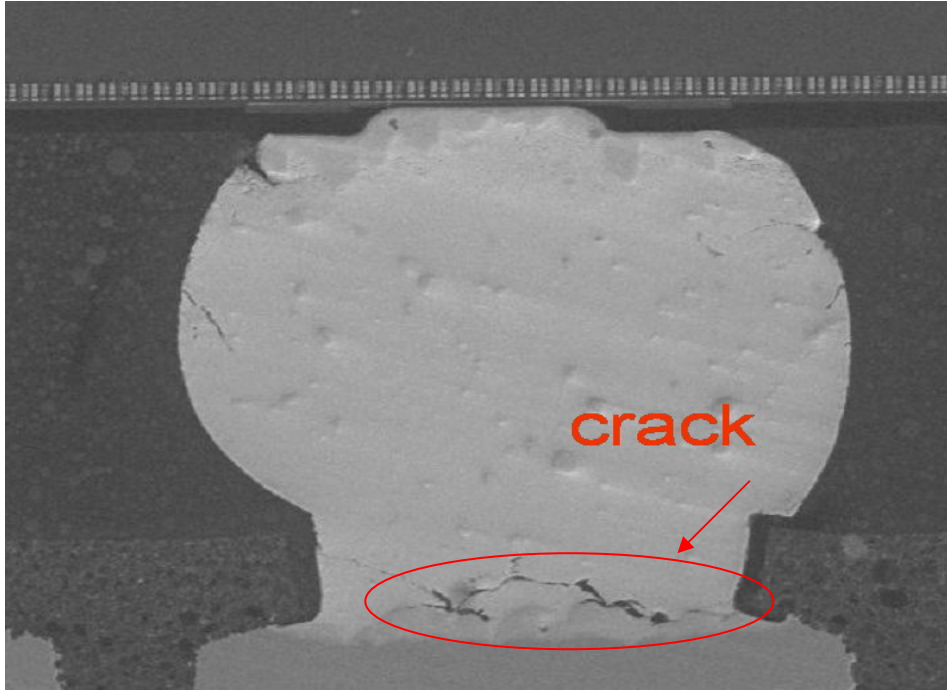


Figure 4.16 The SEM picture of sample D

Figures 4.17 and 4.18 showed the maximum XY plane shear stress and von Mises stress of the outmost solder bump for the six packaging samples from simulation results, respectively. Good correlation between simulation and TCT reliability test was found for bumping cracking in sample A (UF-4) with high-lead Sn95Pb solder. The high-lead alloy (Sn95Pb) had lower elastic modulus ($E=2388 \text{ kg/mm}^2$) than conventional Sn37Pb eutectic alloy ($E=2600 \text{ kg/mm}^2$). Thus, high-lead solder bumps may deform easily during reflow process or TCT. Therefore, high-lead packaging assembly needed more rigid underfill material to provide enough protection for solder bumps. Since UF-4 material had lowest Tg temperature and elastic modulus than the other three underfill materials, solder bumps would experience higher stress than the other underfill materials. Based on FEA results, the von Mises stress of the outmost bump in sample A (60.3 kg/mm^2) was 23.8% higher than sample B (48.7 kg/mm^2). Thus, bump crack may occur easily in sample A resulting from the higher stress induced during thermal fatigue test.

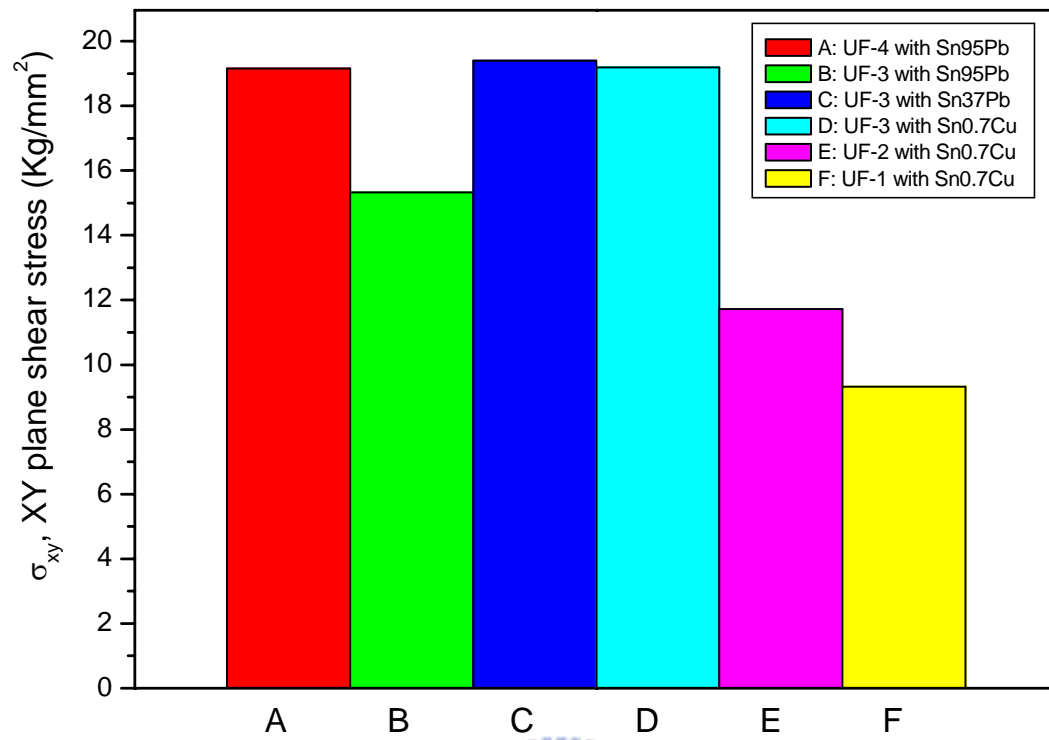


Figure 4.17 The max shear stress of the outmost bump

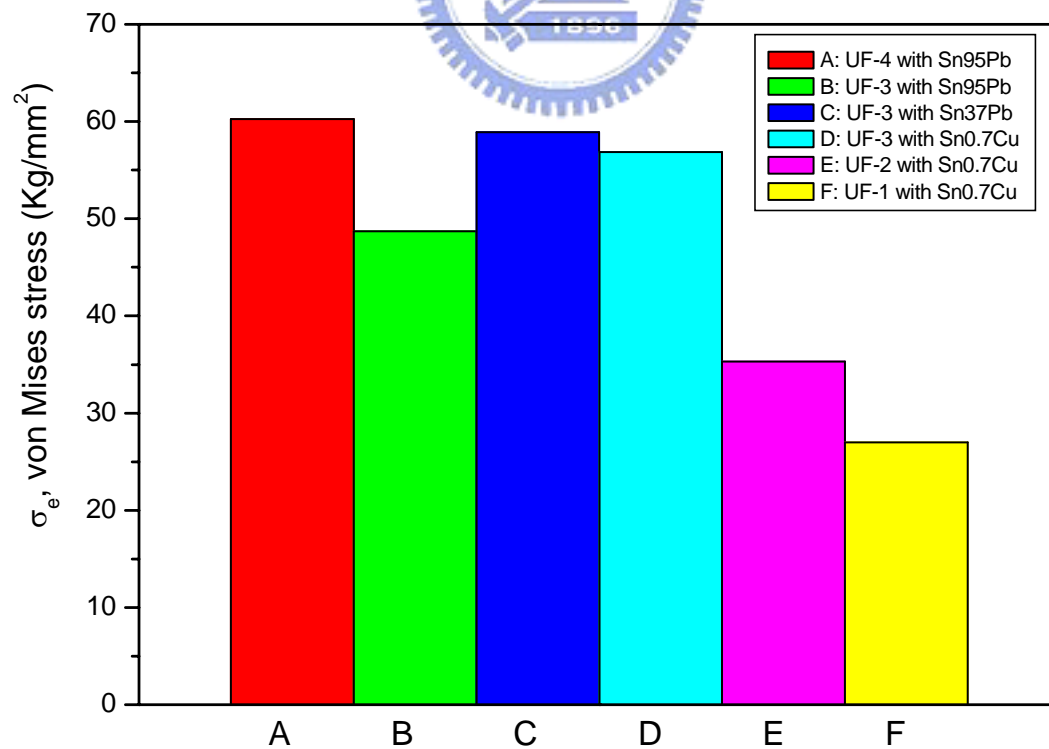


Figure 4.18 The max von Mises stress of the outmost bump

Among three kinds of underfill material (UF-1 to UF-3) under evaluation for lead-free solder bump, only the package with UF-3 underfill material showed failure with bump crack in Sample D. The von Mises stress of the outmost bump in the assembly with UF-3 underfill materials showed the highest stress (56.8 kg/mm^2) than the other two underfill materials, because UF-3 material provided more compliant mechanical properties (low T_g and low modulus). However, the UF-3 material was also used in the packaging sample with eutectic solder (Sample C) and high-lead bumps (Sample D). The von Mises stress in the package with eutectic solder was slightly higher than lead-free Sn0.7Cu assembly, but it did not cause any bump failure. This implied that weaker solder joints strength existed in the lead-free Sn0.7Cu solder bumps in the package even though the modulus of Sn0.7Cu was higher than lead-tin solder. Since lead-free solder introduced new fluxes in the process and high reflow temperature, it was suspected that the residues and byproducts may degrade the adhesion between the die and the underfill material [41]. Furthermore, Sn0.7Cu alloy may form a brittle Cu-Sn inter-metallic compound (IMC) layer at the solder/Cu pads interface. Moreover, the thickness of IMC layer was proportional to reflow temperature and time [42]. The sustained growth of Cu-Sn layer during reflow or reliability temperature cycling test may decrease the interface strength of solder joints. If there was a crack occurring in this region, it would propagate easily along the IMC layer [43]. Therefore, if lead-free alloy was adopted as the solder bump, we must pay more attention to the bump cracking issue.

The UF-3 material had the lower T_g temperature and lower elastic modulus than UF-1 and UF-2 underfill materials. Thus, UF-3 material could not provide enough protection to solder bumps, especially when the environmental temperature exceeded its T_g temperature. Therefore, underfill material with higher elastic modulus and higher T_g temperature was highly desired for bump protection.

Although there was no low-K delamination observed after TCT 1000 cycles, low-K delamination risks in these 6 samples was still assessed by FEA. For the low-K delamination, σ_{xy} and σ_e did not play an import role. The low-K delamination potential index was changed to be the first principal stress, σ_I [44]. Figure 4.19 showed the maximum σ_I occurred at the corner of low-K layer. Figure 4.20 showed the maximum σ_I in low-K layer for samples A through F. Clearly, sample F (UF-1 with Sn0.7Cu) had the lowest stress than other samples in low-K layer. The σ_I value appeared to increase with decreasing Tg or increasing CTE among samples D, E and F. The effect of underfill materials' properties such as Tg, CTE and E on the die/package reliability warranted our further examination in the following section.

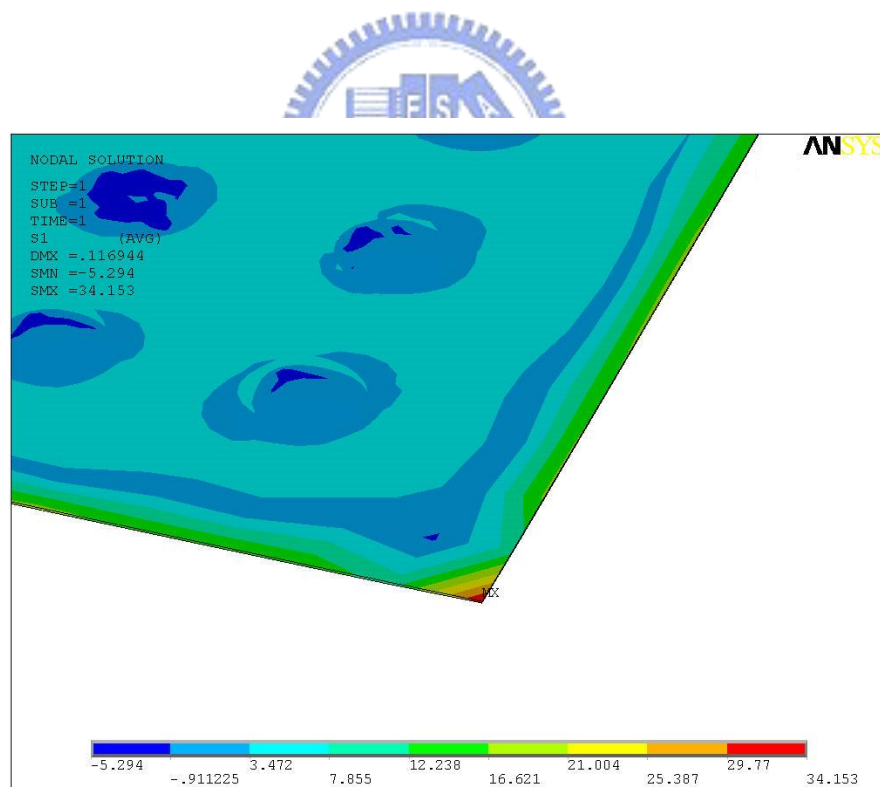


Figure 4.19 The σ_I distribution of low-K layer for sample D

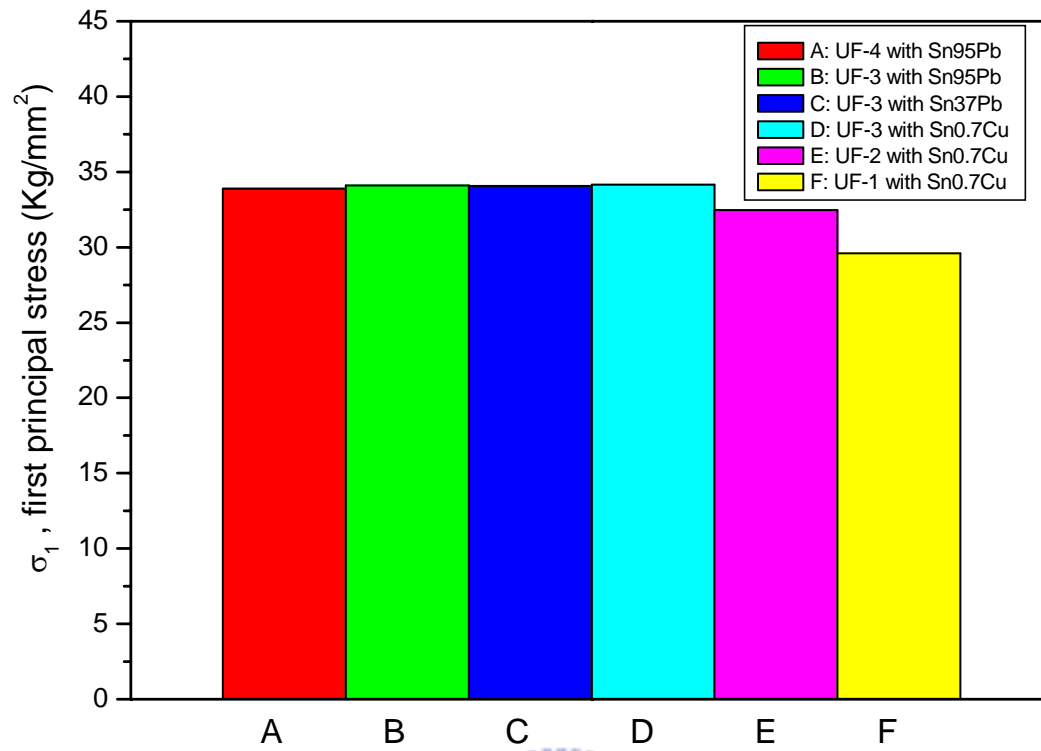


Figure 4.20 The max first principal stress of low-K layer



4.5 The effects of different mechanical properties of underfill

In the aforementioned study, stresses of bump and low-K layer were calculated by finite element method. Moreover, the simulation results correlated well with the thermal reliability test results (bump cracking) for high-lead solder with weaker strength. The higher bump crack risk in the lead-free (Sn0.7Cu) process was believed to be caused by the formation of IMC and poor adhesion due to flux residue. Nevertheless, it is of significant importance to further study how to modify the properties of underfill materials to protect both bump and fragile low-K.

The glass transition temperature (T_g), the coefficient of thermal expansion (CTE), and the elastic modulus (E) were the major thermo-mechanical properties of underfill material which directly influenced on thermal reliability. Most of the underfill materials were composed by epoxy resin and silica filler. Silica filler possessed low CTE and high modulus properties which were opposite to epoxy resin. The quantity and shape of filler determined the CTE and modulus of underfill materials. Therefore, the CTE and modulus were inversely related in the traditional formulation of underfill materials.

However, these two properties can be treated as independent parameters to examine the optimal properties of underfill materials for protecting both the lead-free bumps and low-k fragile layer. The UF-1 and UF-2 materials were a case in point, although the T_g temperatures were not the same. We chose Sample E as a reference to understand how CTE and E may affect the stress distribution in FC-BGA packages by using finite-element analysis. We modified the input data of UF-2 mechanical properties (E1 or CTE1) to study the stress variation of solder bumps and low-K layer.

Figures 4.21 and 4.22 showed the impacts of solder bump and low-K layer stresses for different modulus and CTE of underfill materials, respectively. The modulus of

underfill material showed opposite trend for solder bump and low-K layer protection. The modulus of underfill material with high modulus reduced bump stress but increased stress for fragile low-K layer. The CTE of underfill material should be as low as possible for reducing stresses at critical region. However, the minimum CTE of underfill was about 20 ppm/°C due to more than 70% filler content would cause capillary flow issue [45].

The Tg temperature of underfill material was also a critical parameter for TCT reliability because CTE and modulus changed acutely for temperature above Tg. An optimal underfill material should provide enough protection to solder bumps at higher temperature region and mitigate the stress to fragile low-K layer at lower temperature during thermal cycling test. Based on FEA results, UF-1 with high Tg, moderate E and lower CTE, can simultaneously provide the lowest stress in both bump and low-K layer.

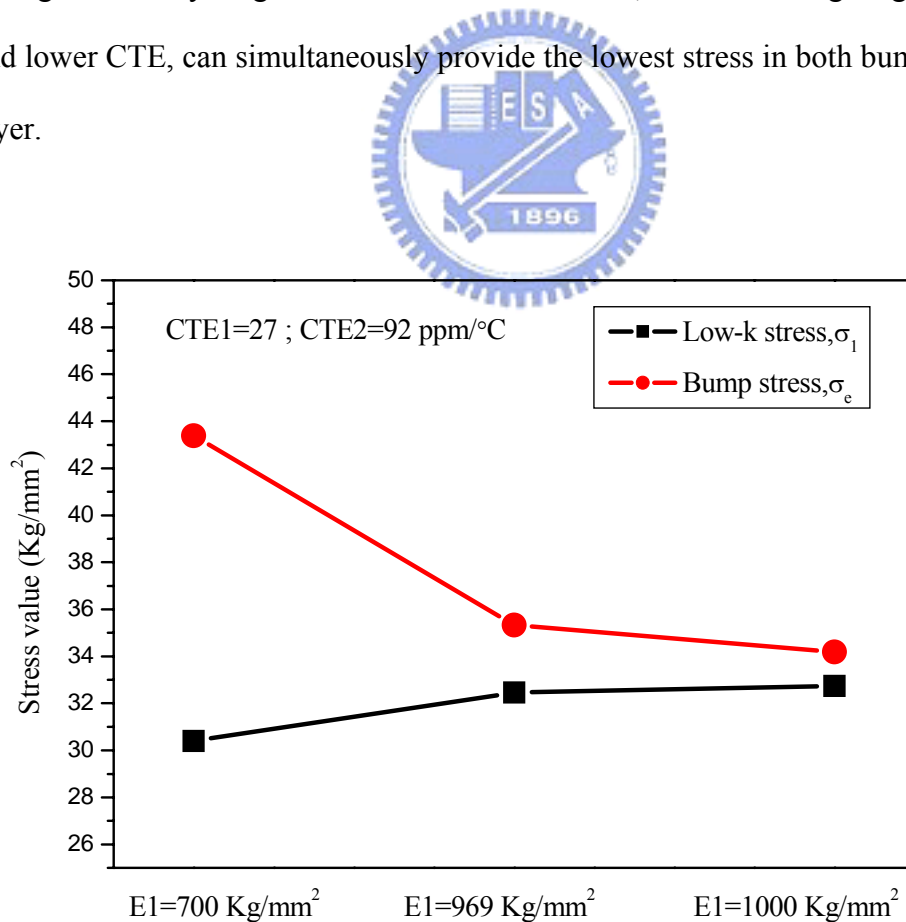


Figure 4.21 The max stress comparison of different E1 of underfill material for sample E

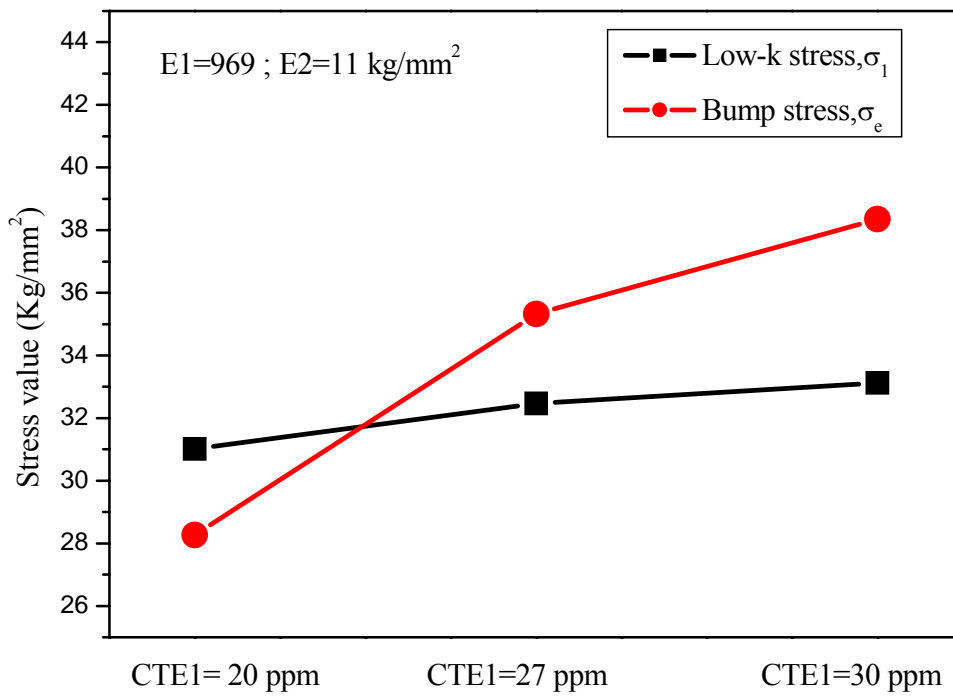


Figure 4.22 The max stress comparison of different CTE1 of underfill material for sample E



4.6 Mechanical properties of underfill material modification

The underfill material can be seen as a two-phase composite. It is often composed by epoxy matrix and silica fillers as shown in Fig. 4.23. J. Qu proposed the more filler loading and smaller filler particle size would result in smaller CTE [46]. In addition, more filler loading also leded higher effective elastic modulus of underfill material. Therefore, the effective E and CTE of underfill could be modified by adjusting the ratio of filler to epoxy resin.

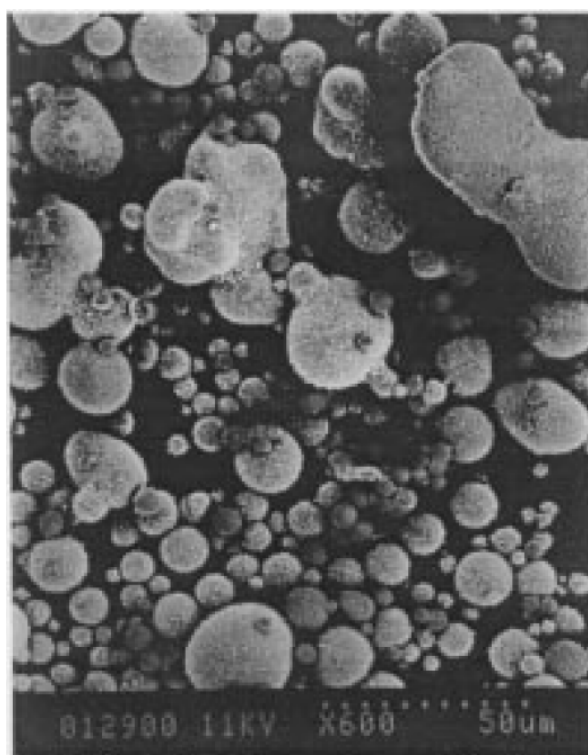


Figure 4.23 Morphology of conventional underfill system

However, the filler loading has no influence on Tg temperature. Therefore, the chemical structure of epoxy resin needs modification in order to change the Tg temperature of underfill material. Figure 4.24 showed the main chain of epoxy resin chemical structure. The more phenyl groups and double bond attached in the main chain will raise Tg temperature due to their rigidity. Crosslink of epoxy resin can be

also enhanced by addition of curing agent to the epoxy system. Typically, the aromatic and aliphatic curing agent system could be employed to increase degree of crosslink in epoxy system. E. Crawford proposed the Tg temperature will increase with higher crosslink functionality of curing agent and lower molecular weight between cross-links [47]. Therefore, we can create desired mechanical properties of underfill material by changing filler loading and epoxy system modification.

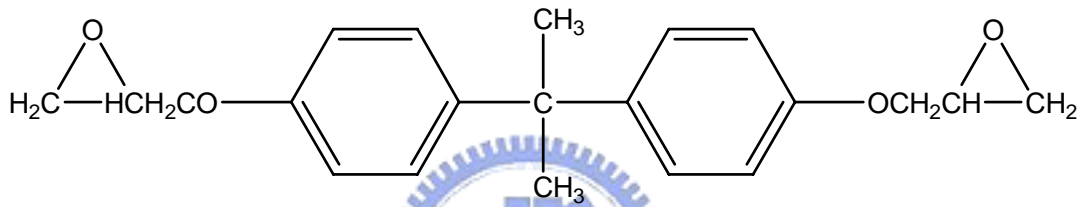


Figure 4.24 The chemical structure of epoxy resin

Chapter 5 Conclusions

As low-K dielectric material and lead-free solder alloy introduced in FC-BGA packages, the underfill material needs to re-design its mechanical properties in order to protect both low-K layer and solder bumps from delamination and crack issues. In this study, the thermo-mechanical deformation of two representative underfill materials were measured and compared by high resolution Moiré interferometry. Based on the experimental results, underfill material with higher elastic modulus may induce larger die warpage and higher strain of solder bumps.

In addition, a simplified three-dimension model for finite element analysis by ANSYSTM was established. The difference of die warpage between FEA prediction and Moiré interferometry measurement were less than 5%. Moreover, thermal reliability of six kinds of FC-BGA samples by TCT 1000 cycles were studied and compared. The reliability test results illustrated that lead-free and high-lead solder bump had higher crack potential due to the higher reflow temperature and weaker mechanical strength of Sn95Pb alloy, respectively. The thermal reliability test results showed good correlation with simulation data. This finite element model can be used to predict the failure potential of new packages without lots of money and time.

Furthermore, the optimal mechanical properties (E, CTE, and T_g) of underfill material for low-K FC-BGA packages by FEA was evaluated. Based on the simulation results, the lower CTE of underfill can mitigate the thermal induced stress for both bumps and low-K layer. However, it showed opposite elastic modulus requests of underfill for bump and low-K layer protection. The underfill with higher modulus could provide more support to solder bumps, but induced higher stress to low-K layer. The T_g temperature played an critical role in underfill material property due to both the CTE and E would possess a abrupt change while the environmental

temperature exceeded the T_g point. Thus, higher T_g temperature of underfill could result in lower CTE and higher E to protect both low-K layer and bumps at high temperature. Therefore, the mechanical properties of novel underfill material for low-K FC-BGA packages should be moderate E, low CTE and high T_g temperature.



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