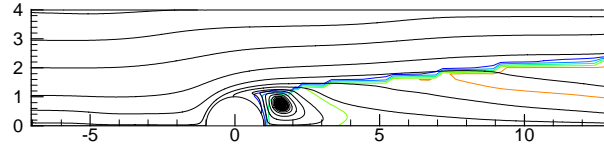
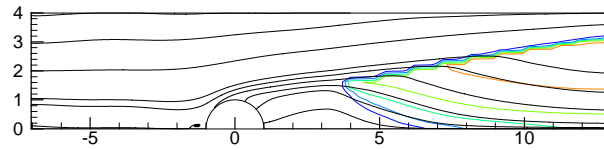


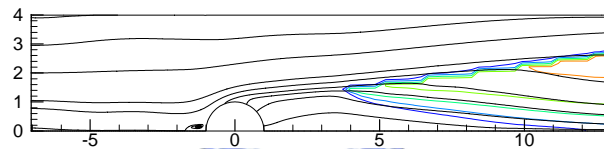
(a) $U_{in} = 0.75$ m/sec



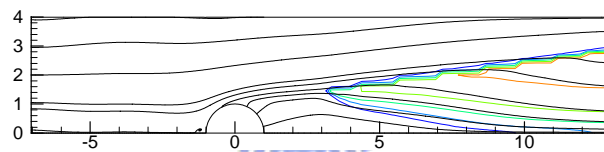
(b) $U_{in} = 1.0$ m/sec



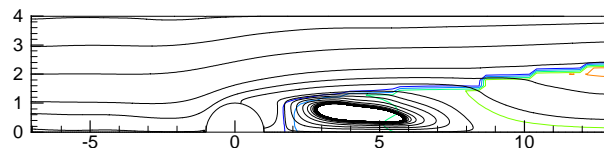
(c) $U_{in} = 1.01$ m/sec



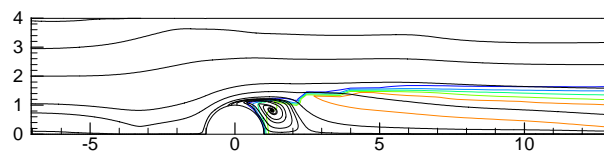
(d) $U_{in} = 1.15$ m/sec



(e) $U_{in} = 1.3$ m/sec



(f) $U_{in} = 1.39$ m/sec



(g) $U_{in} = 1.4$ m/sec

Fig. 1.1 Series of temperature contour distributions and flow streamlines for a circular cylinder by Tsa (2003)

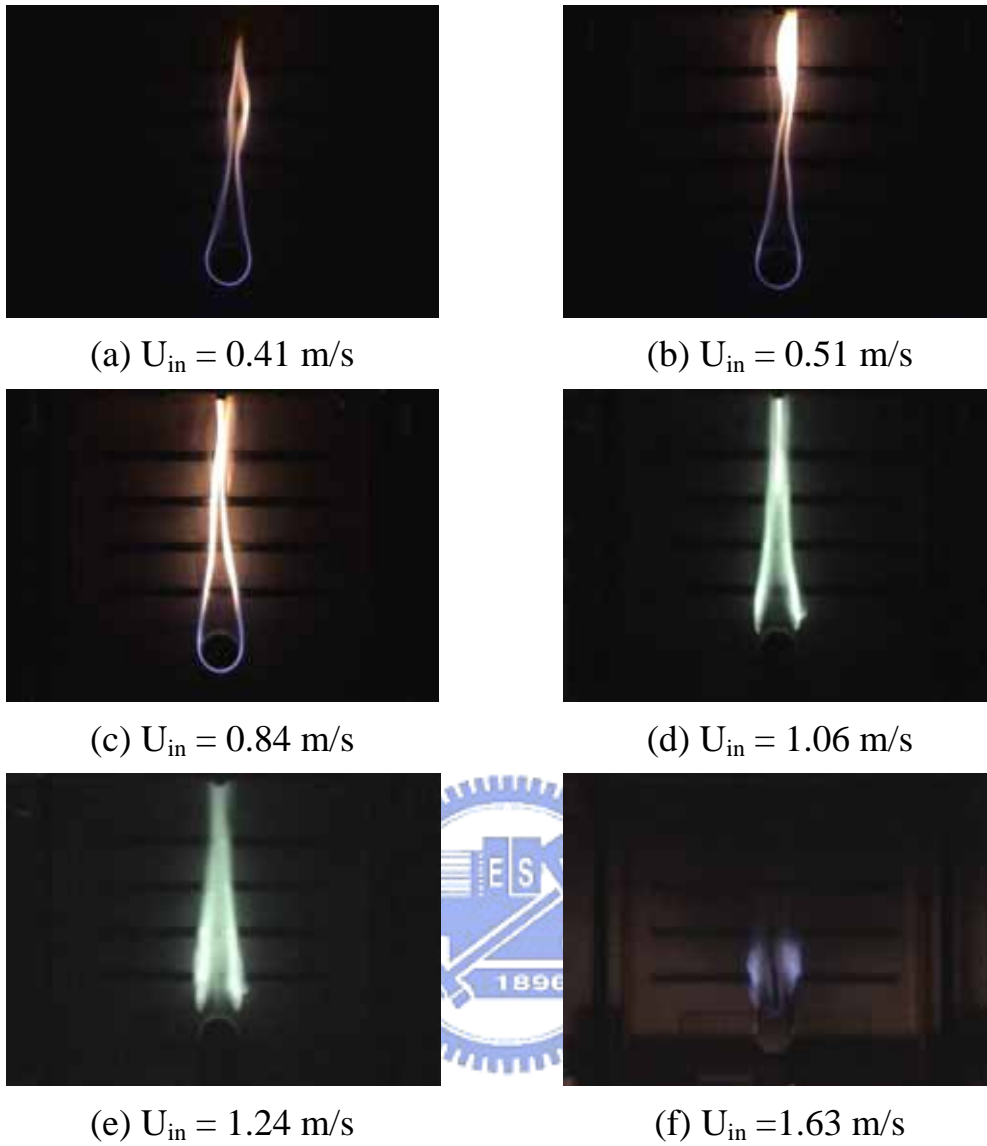


Fig. 1.2 Series of flame configurations as a function of incoming flow velocity ($V_w = 1.4 \text{ cm/s}$) by Chang (2002)

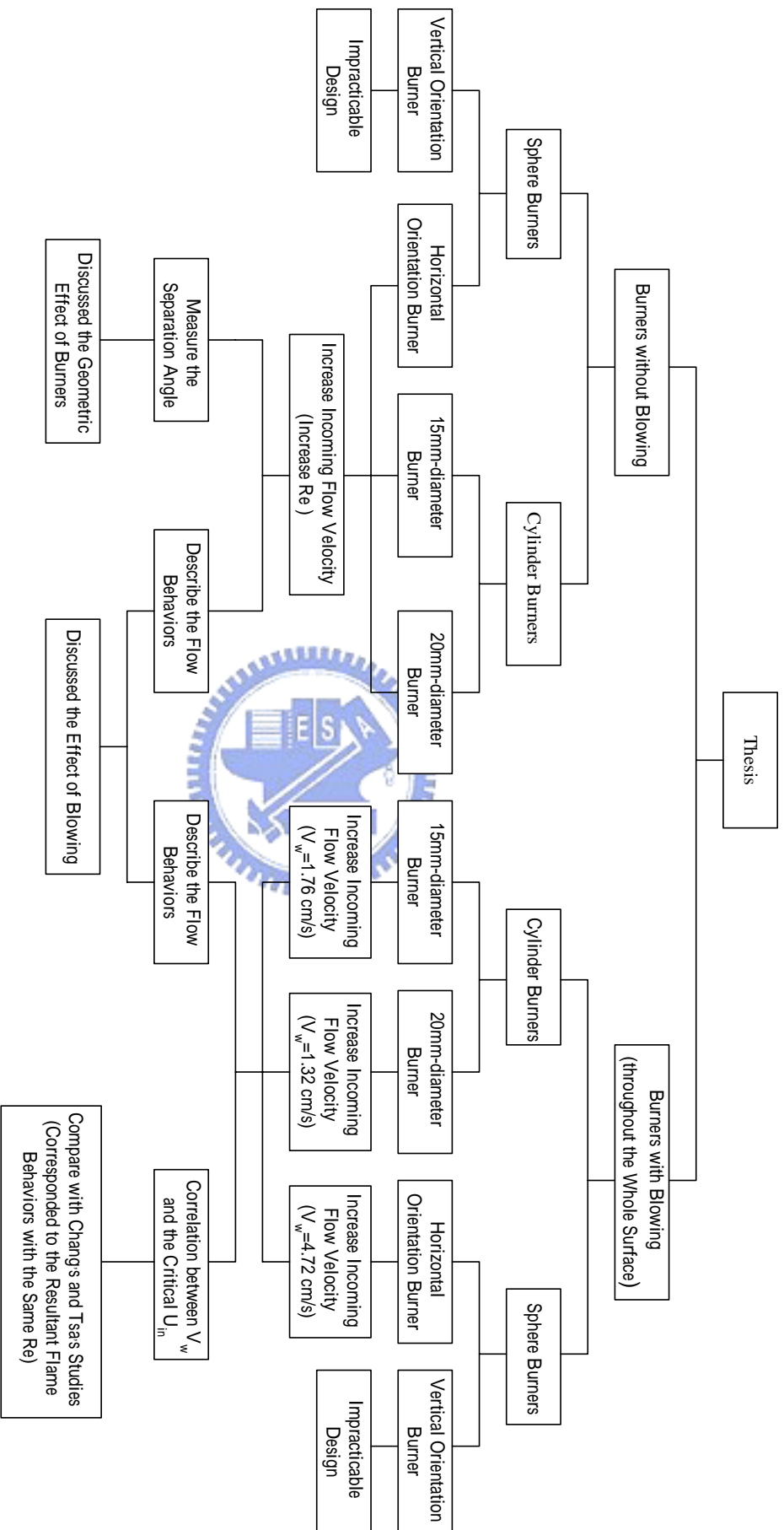


Fig. 1.3 Scheme diagram of the thesis

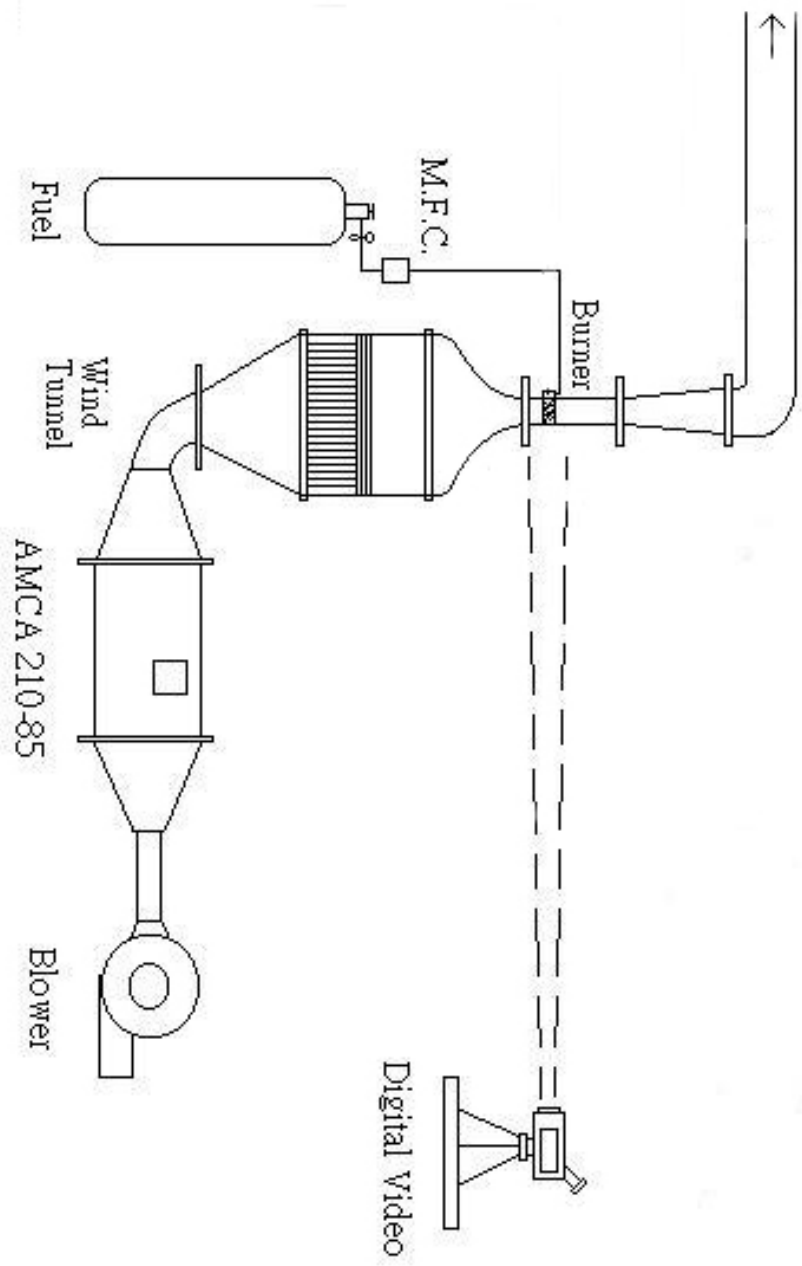


Fig. 2.1 Schematic drawing of overall experimental system

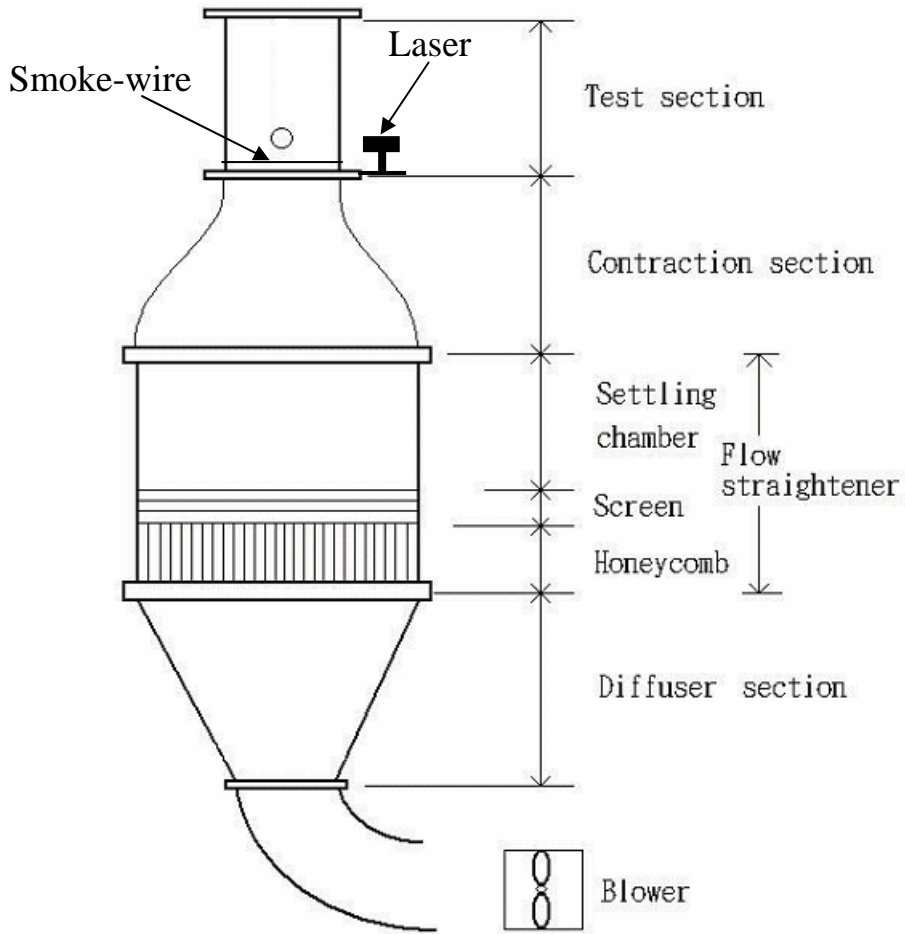


Fig. 2.2 Schema of the wind tunnel

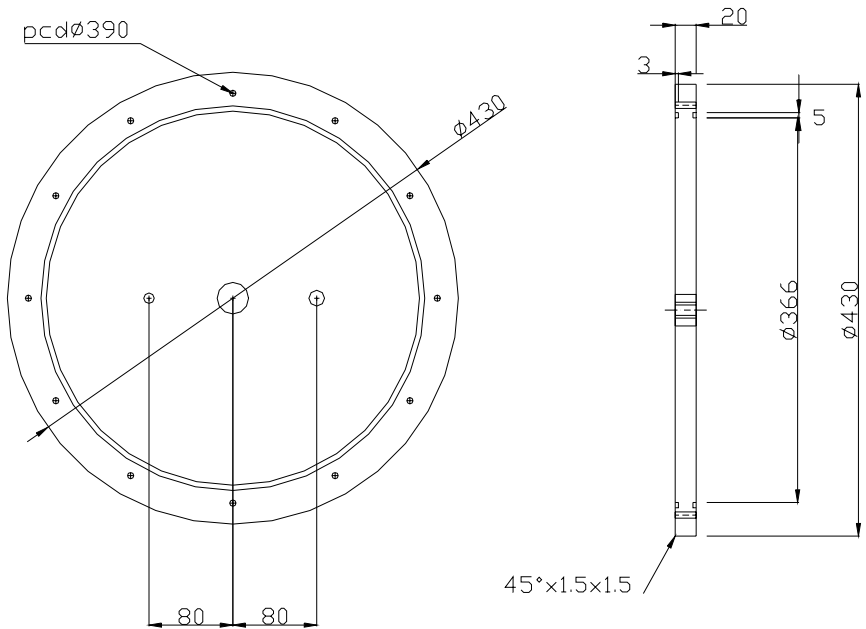
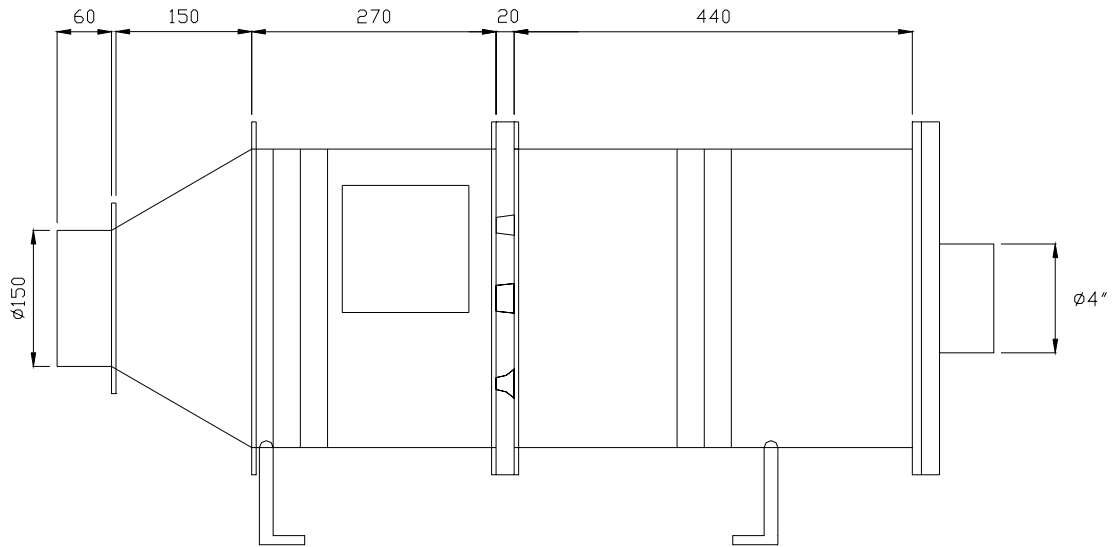


Fig. 2.3 The design of AMCA 210-85 standard

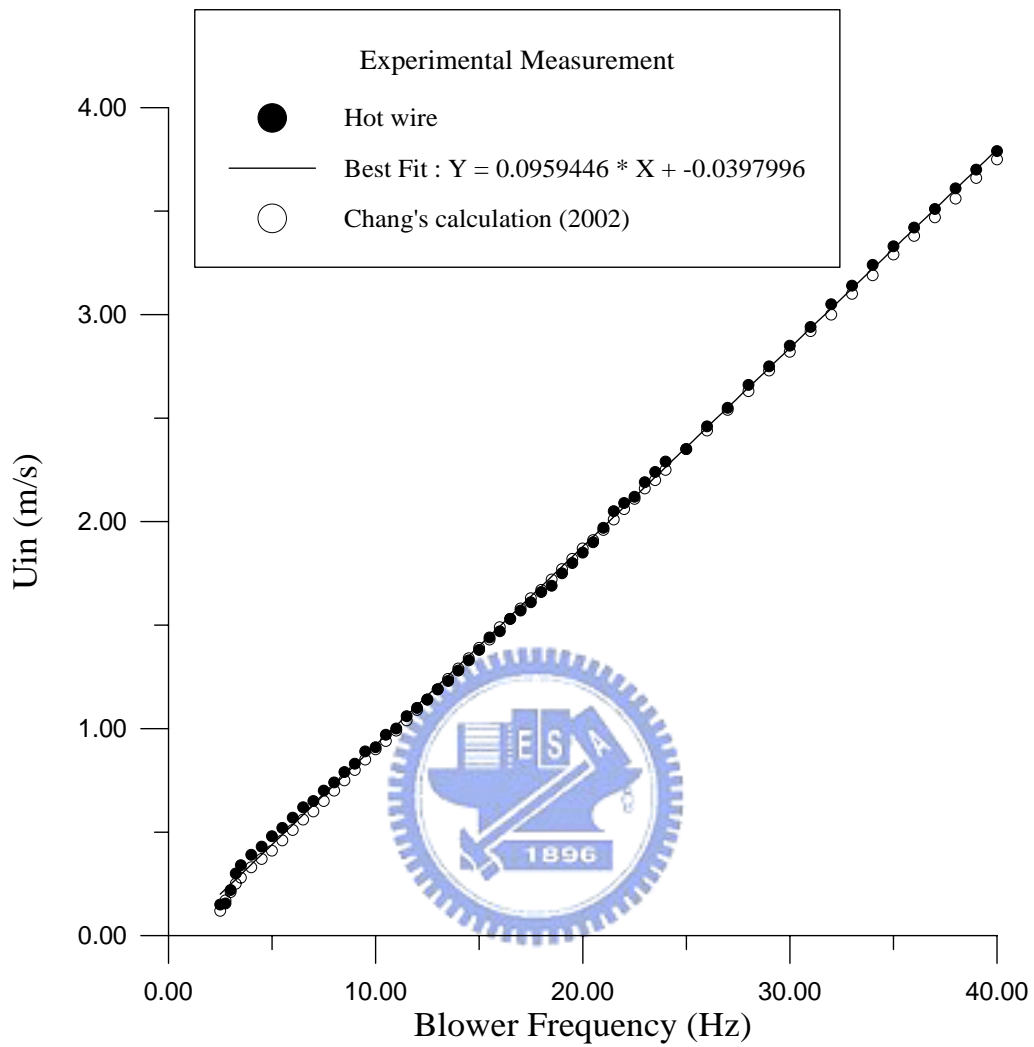


Fig. 2.4 The relation figure of blower frequency and airflow velocity



Fig. 2.5 The bases of wind tunnel and blower are separated by a flexible plastic ductwork

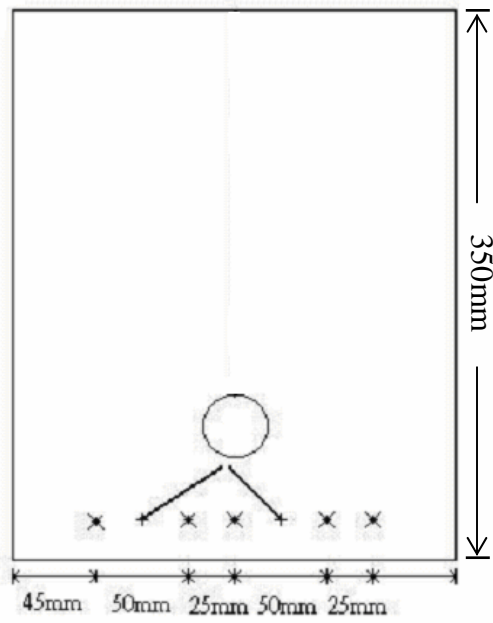


Fig. 2.6 The positions of hot wire sensors



Fig. 2.7 Porous cylindrical burner and cylindrical brass rod

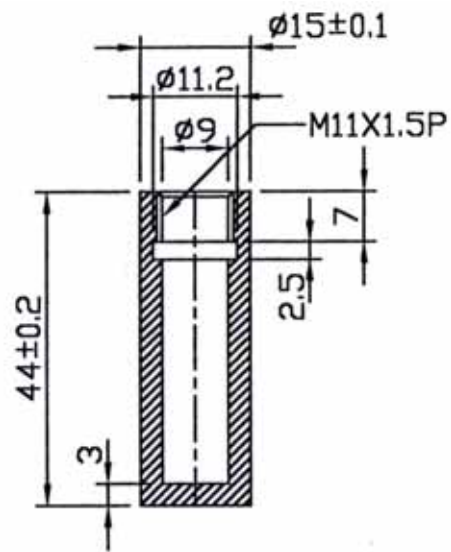


Fig. 2.8 Cylinder of 15mm

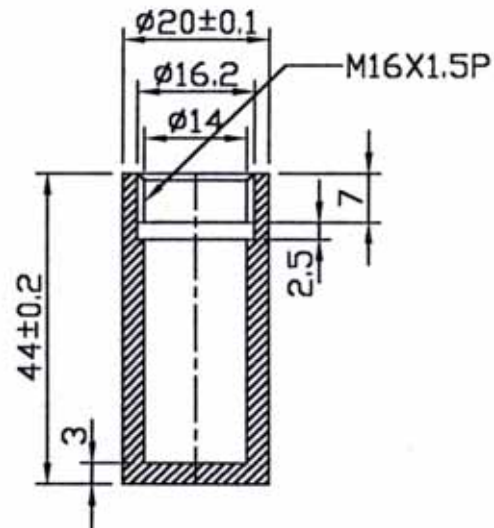
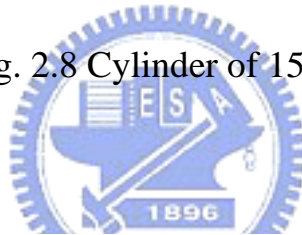
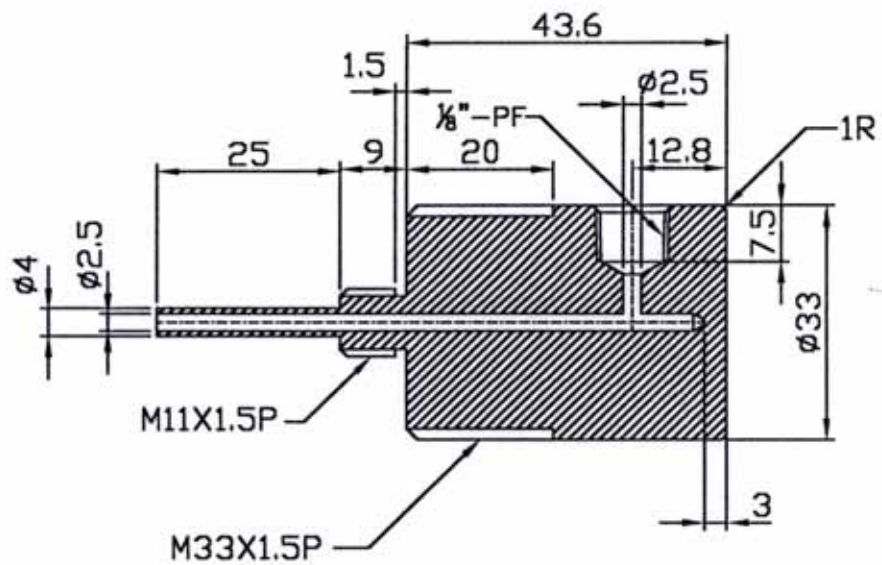
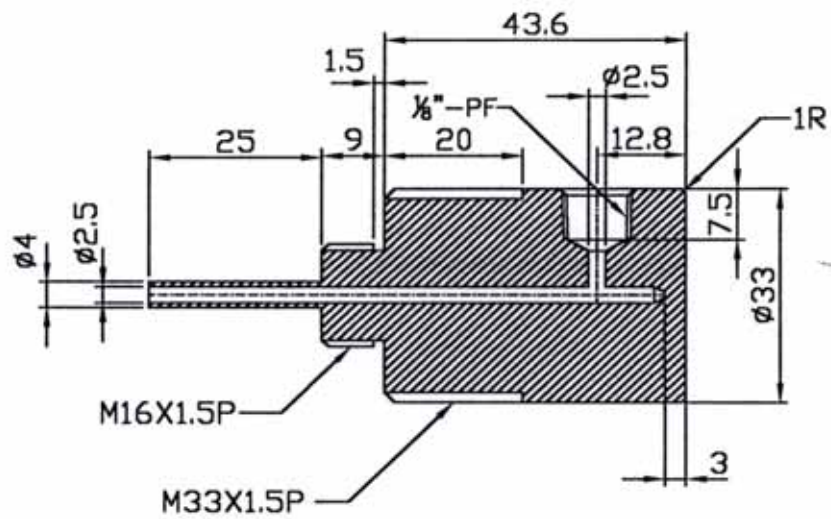


Fig. 2.9 Cylinder of 20mm



(a)



(b)

Fig. 2.10 Cylindrical brass rod for (a) 15mm and (b) 20mm cylinders

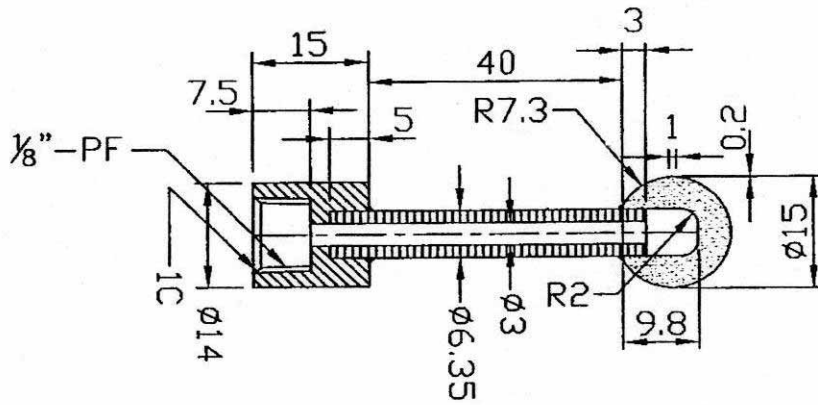


Fig. 2.13 Horizontal orientation burner



Fig. 2.14 Hot wire



Fig. 2.15 IFA 100 Diagnostics

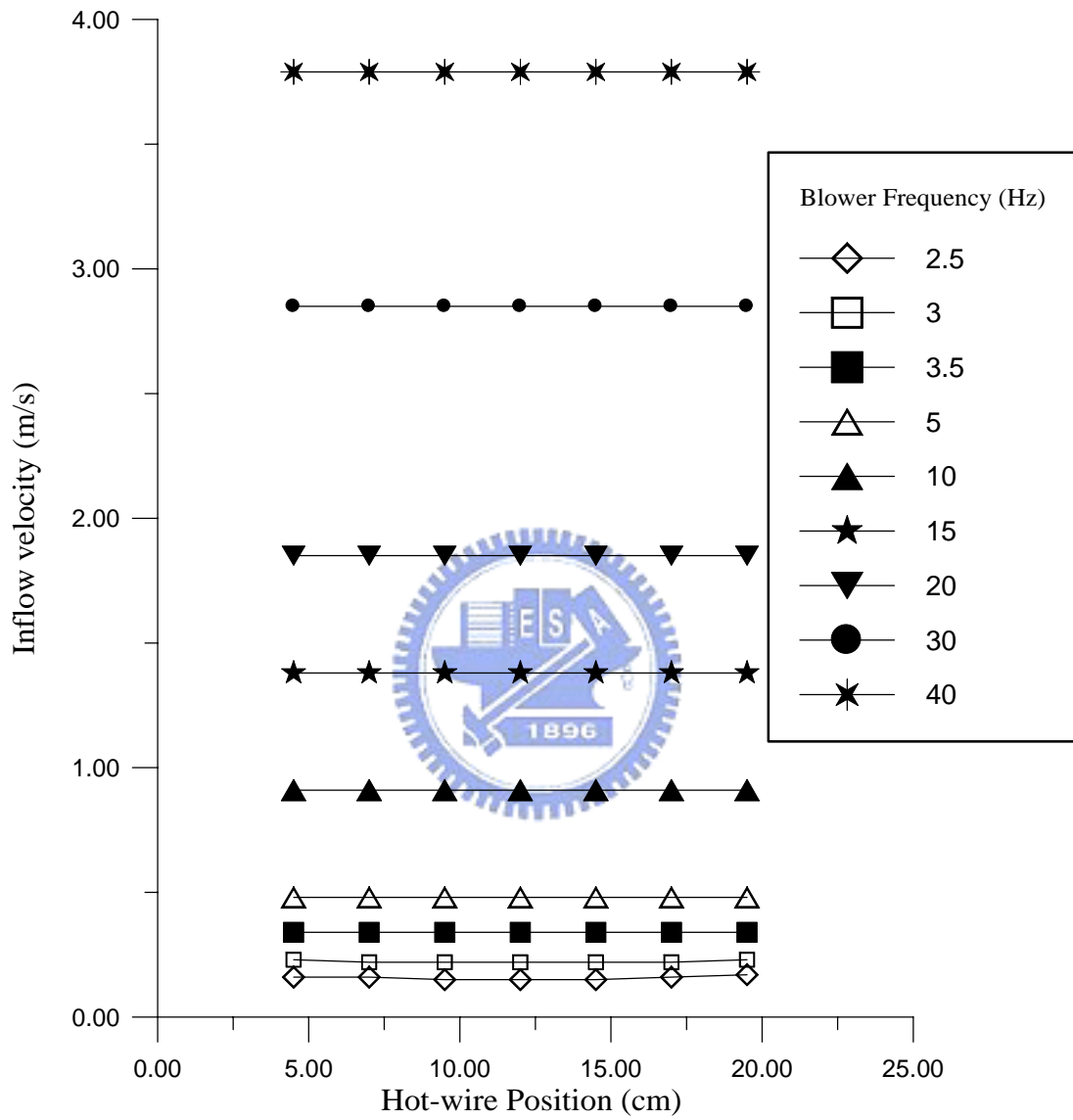


Fig. 2.16 Inflow velocity at each position in the test section

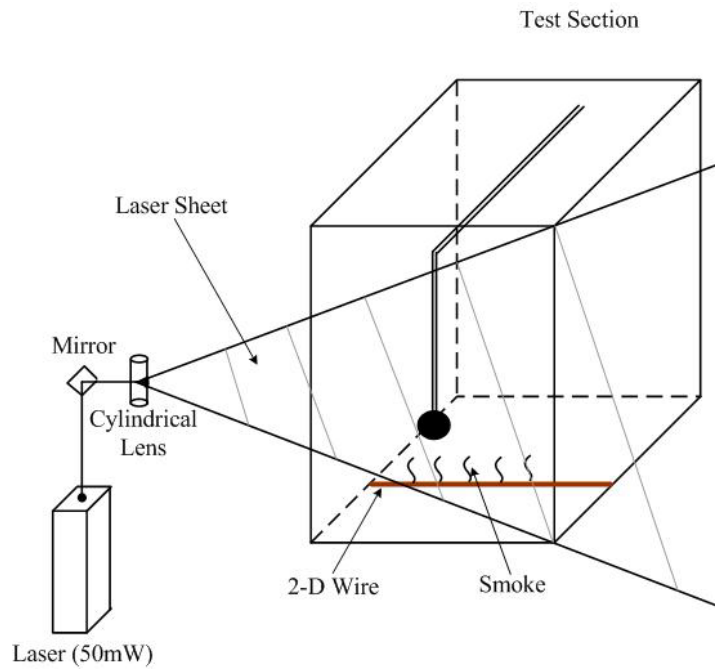


Fig. 2.17 The schematic configuration of smoke wire



Fig. 2.18 The structural configuration of smoke generation

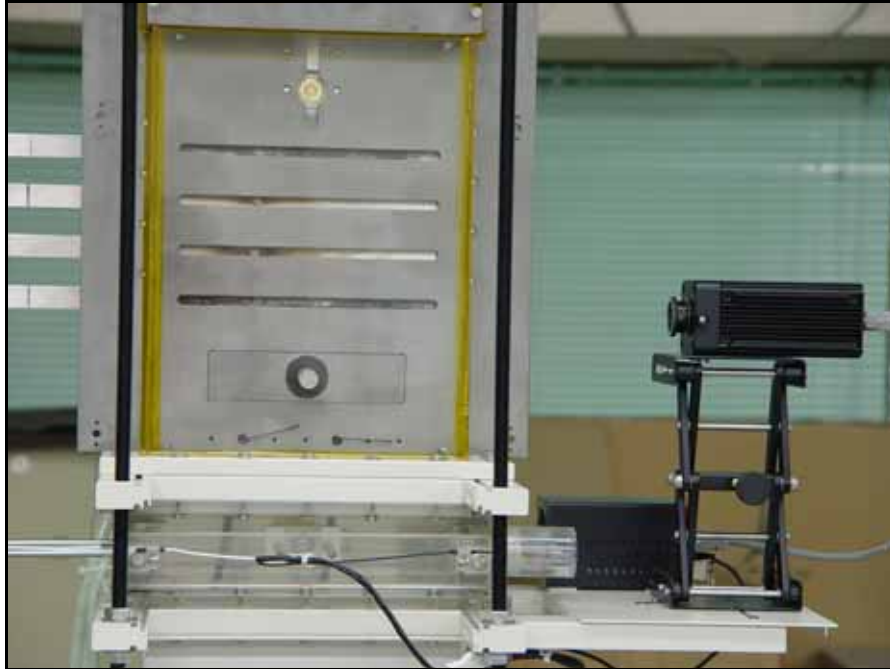


Fig. 2.19 Arrangement of Laser

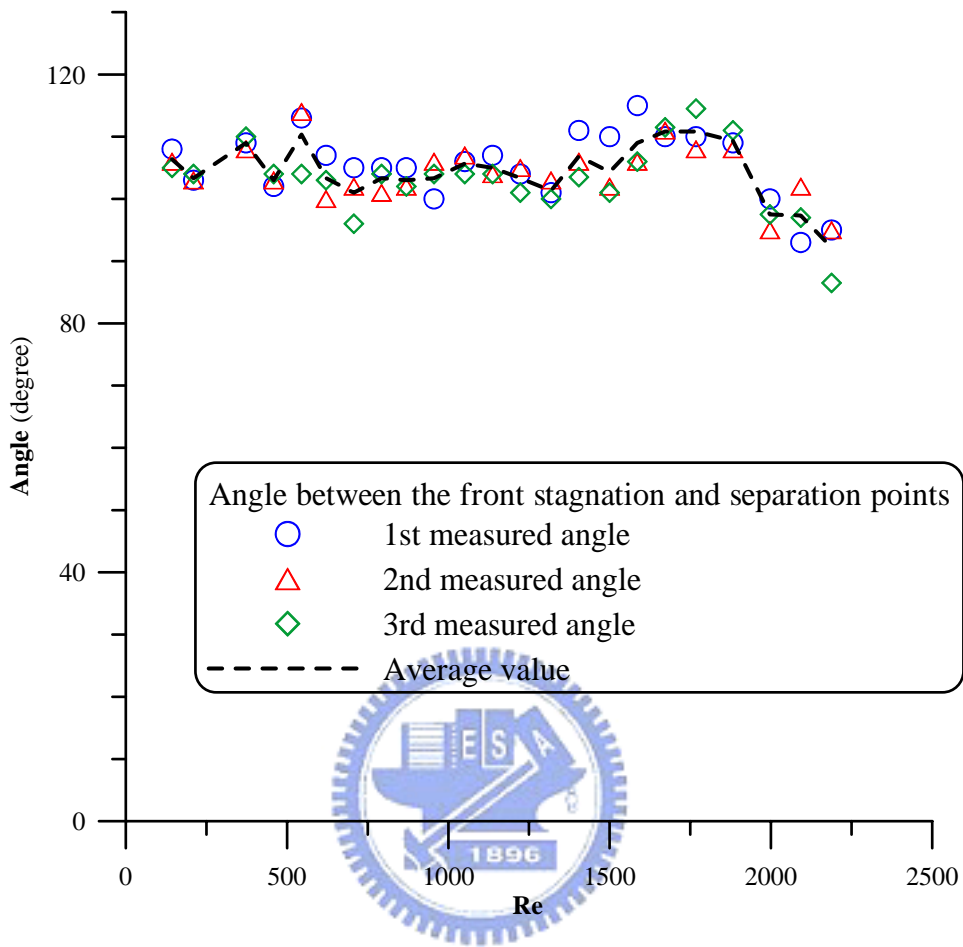


Fig. 3.1 The diagram of experimental repeatability

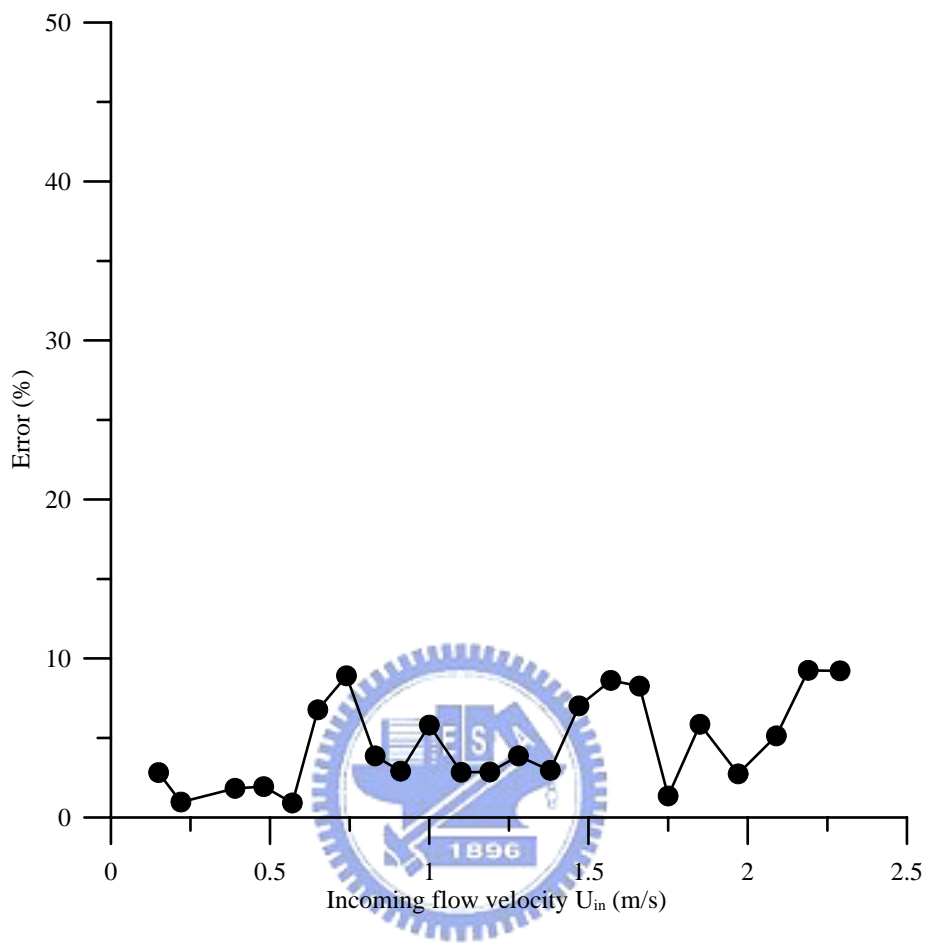
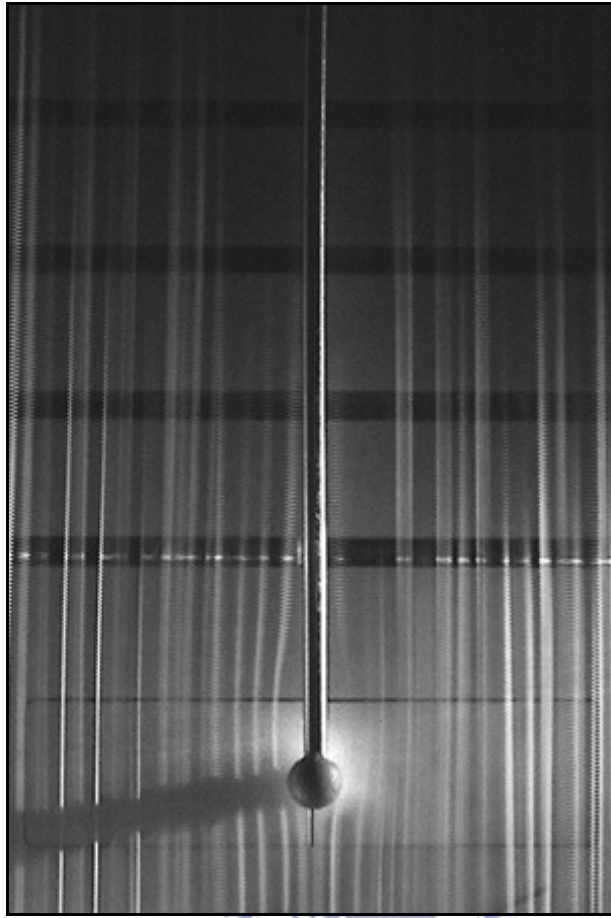
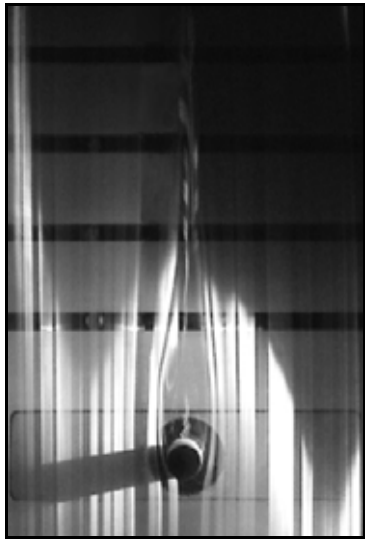


Fig. 3.2 The errors of experimental repeatability

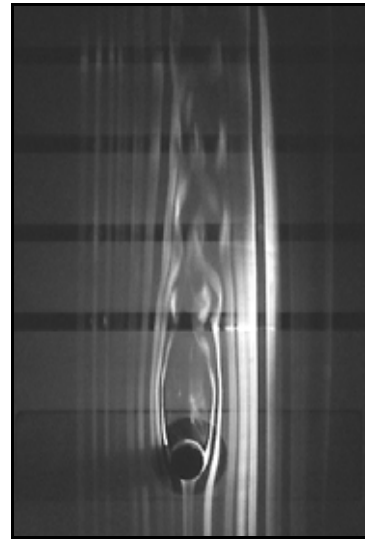


Re = 2,092

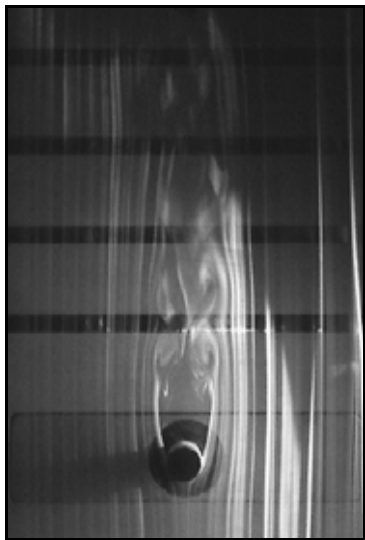
Fig. 4.1 Flow past the burner in vertical orientation parallel to the flow stream



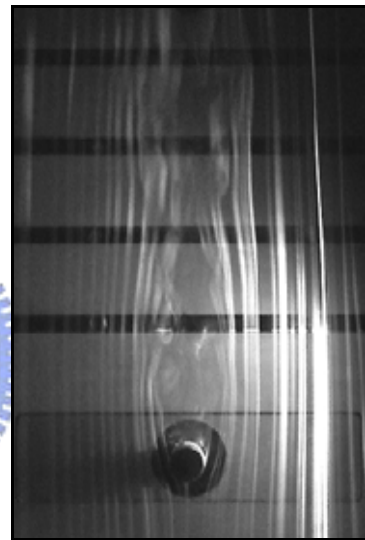
(a) $Re = 143$



(b) $Re = 210$



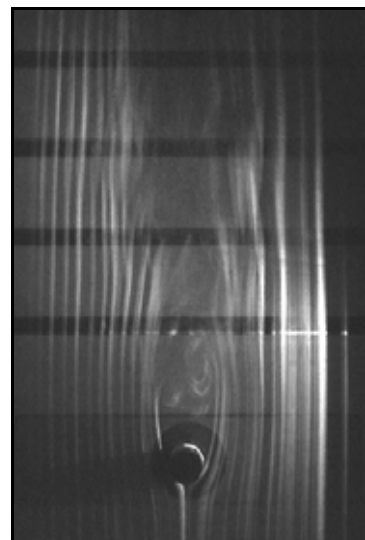
(c) $Re = 372$



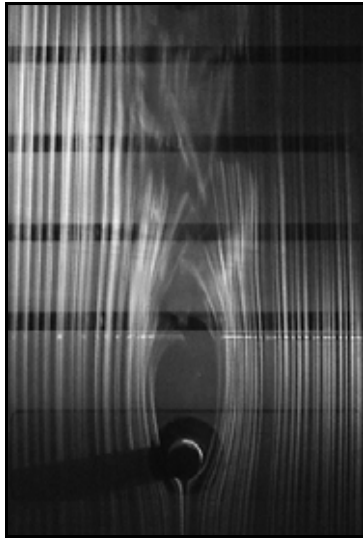
(d) $Re = 544$



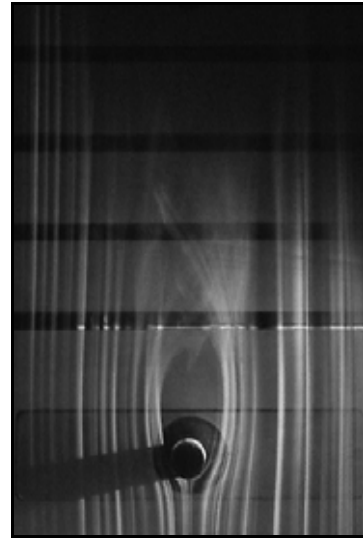
(e) $Re = 707$



(f) $Re = 1,050$



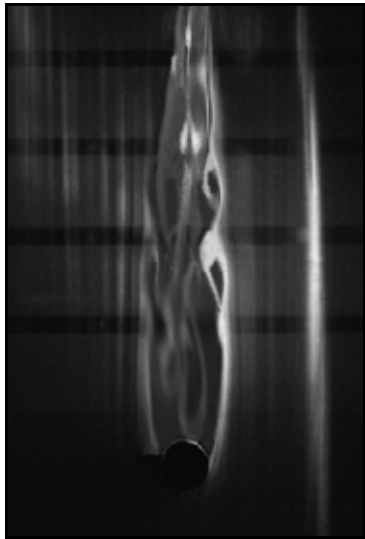
(g) $Re = 1,500$



(h) $Re = 2,092$

Fig. 4.2 Series of flow behaviors as a function of Reynolds number (15mm-diameter cylinder, $V_w = 0$ cm/s)





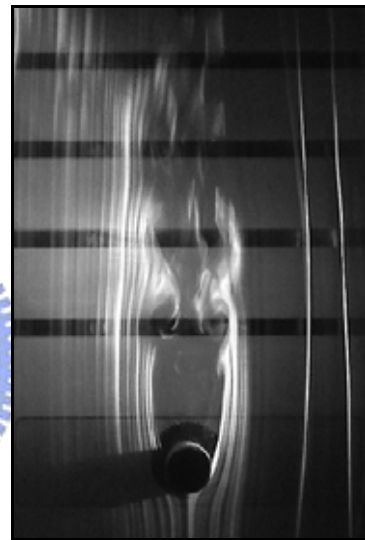
(a) $Re = 191$



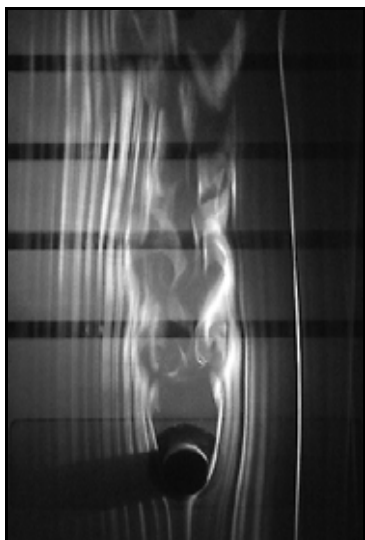
(b) $Re = 280$



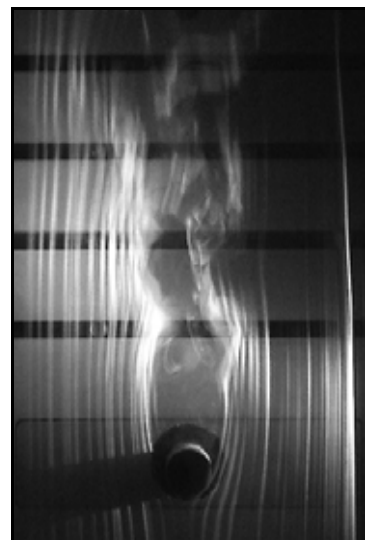
(c) $Re = 496$



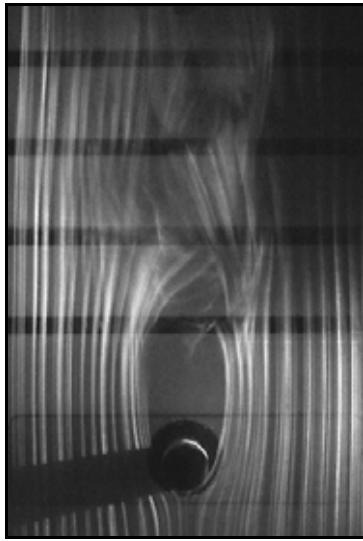
(d) $Re = 611$



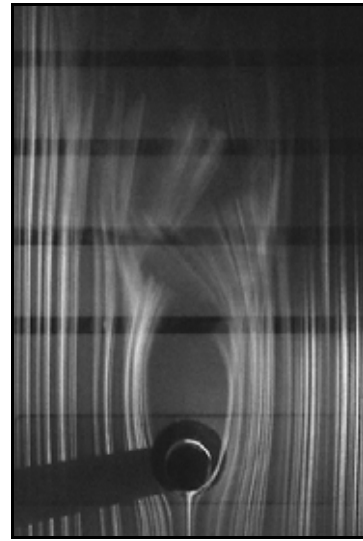
(e) $Re = 726$



(f) $Re = 1,057$



(g) $Re = 2,000$



(h) $Re = 2,356$

Fig. 4.3 Series of flow behaviors as a function of Reynolds number (20mm-diameter cylinder, $V_w = 0$ cm/s)



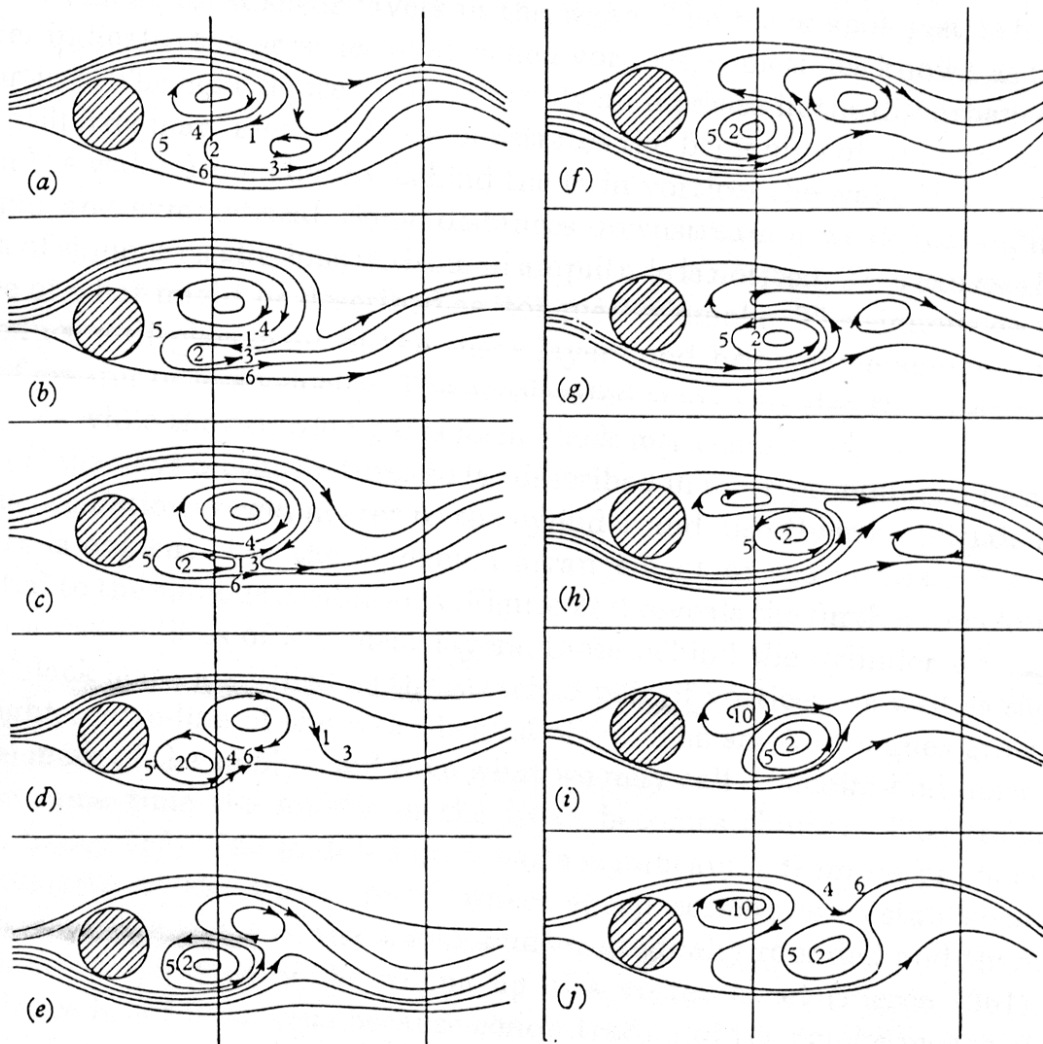


Fig. 4.4 The development of vortices by Nayler and Frazer (1917)

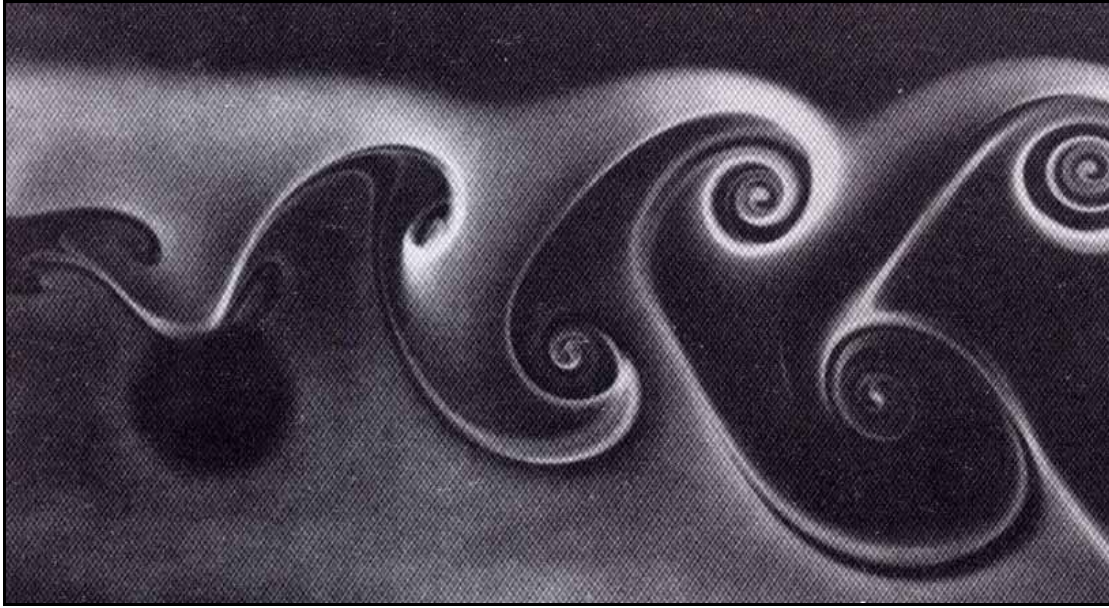


Fig. 4.5 Kármán vortex street behind a circular cylinder at $Re = 200$.
(Photograph by Gary Koopmann)

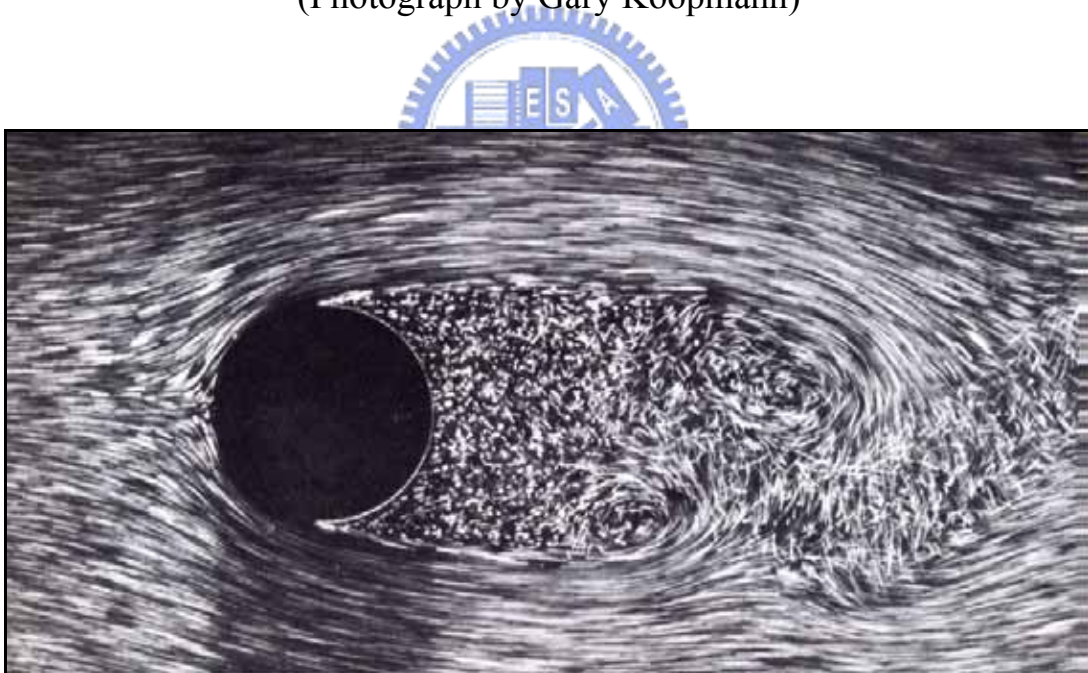
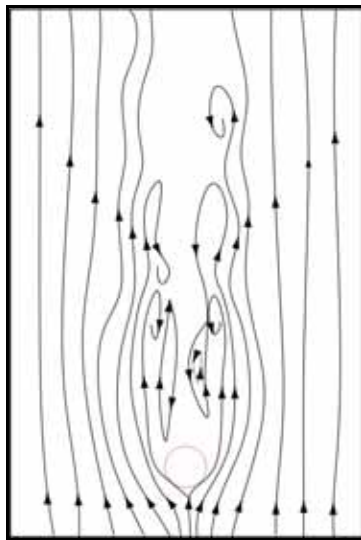
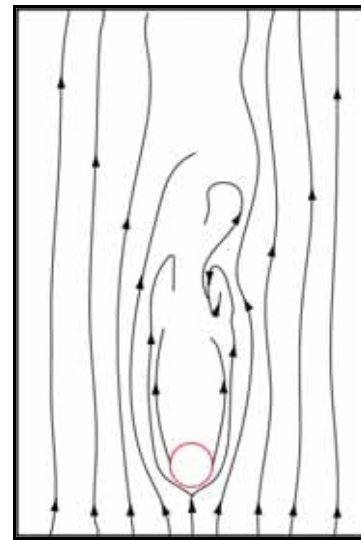


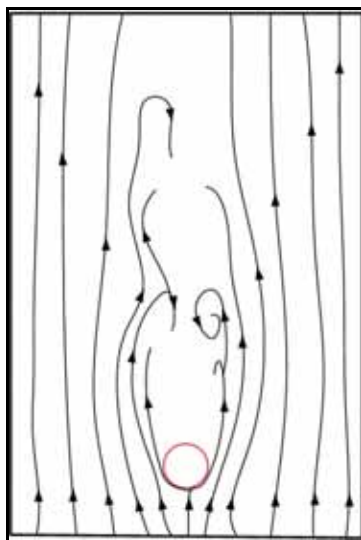
Fig. 4.6 Flow pattern for circular cylinder at $Re = 2,000$.
(Photograph by Werlé & Gallon 1972)



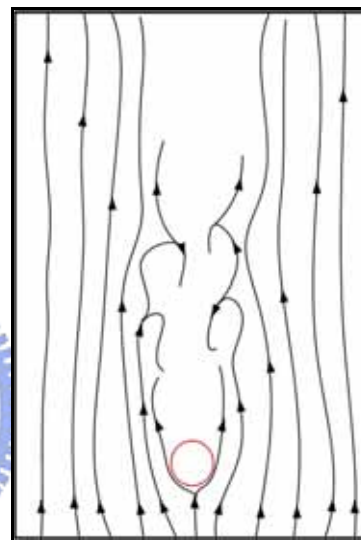
(a) $Re = 280$



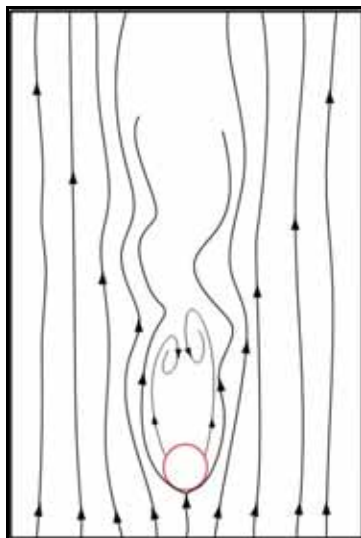
(b) $Re = 496$



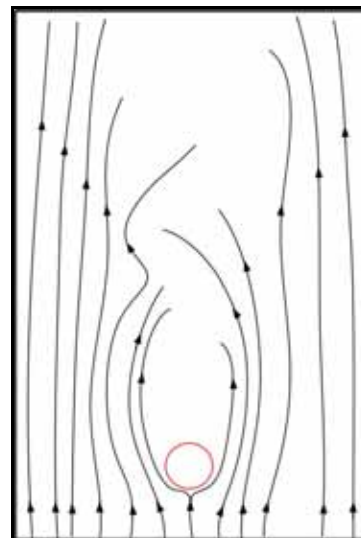
(c) $Re = 611$



(d) $Re = 726$



(e) $Re = 1,057$

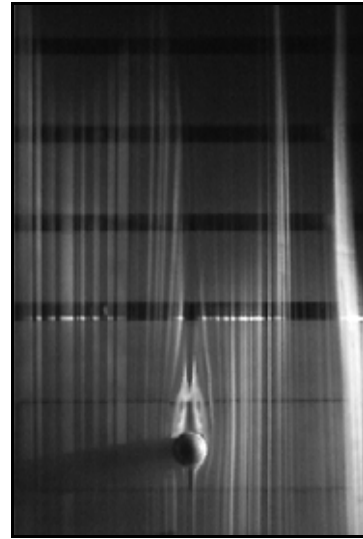


(f) $Re = 2,356$

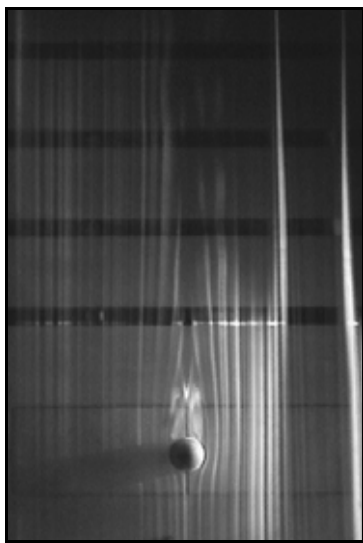
Fig. 4.7 Series of schematic sketches corresponded to Fig. 4.3



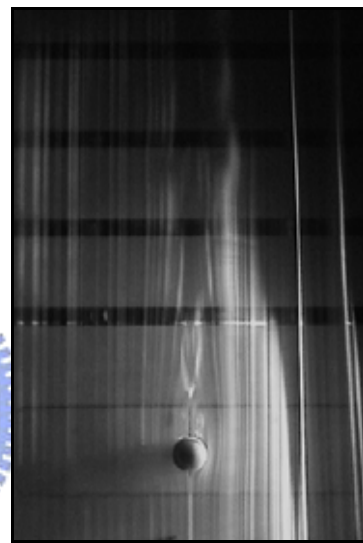
(a) $Re = 143$



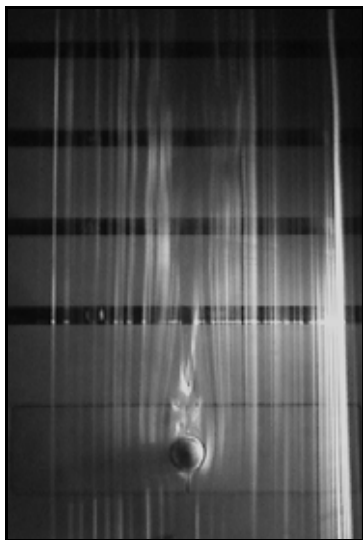
(b) $Re = 210$



(c) $Re = 372$



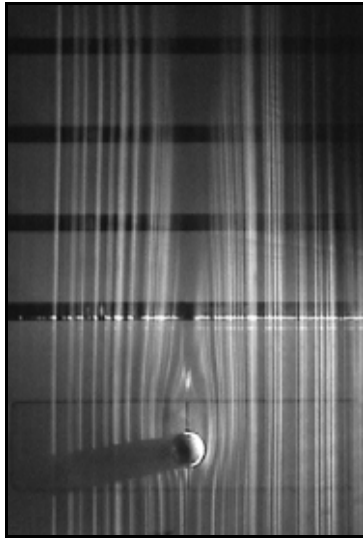
(d) $Re = 458$



(e) $Re = 621$



(f) $Re = 1,050$



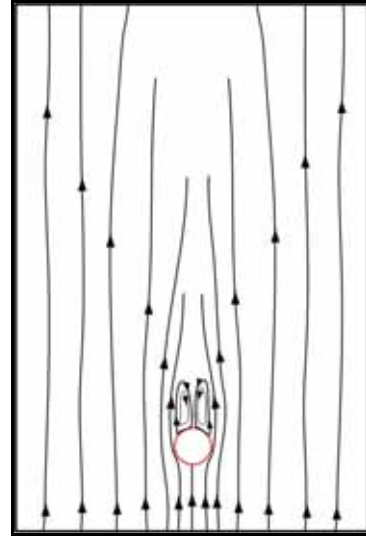
(g) $Re = 1,500$



(h) $Re = 2,092$

Fig. 4.8 Series of flow behaviors as a function of Reynolds number (15mm-diameter sphere, $V_w = 0$ cm/s)





$Re = 143$



Fig. 4.9 Schematic sketch for 15mm-diameter sphere ($V_w = 0$ cm/s)

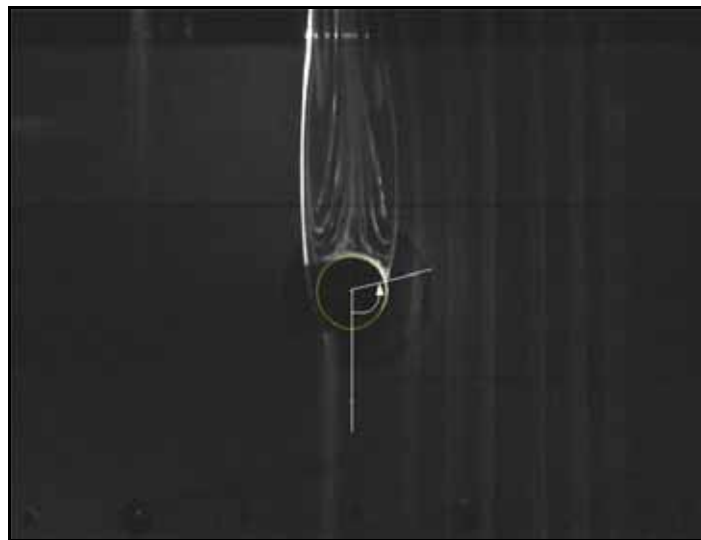


Fig. 4.10 Definition of separation angle θ

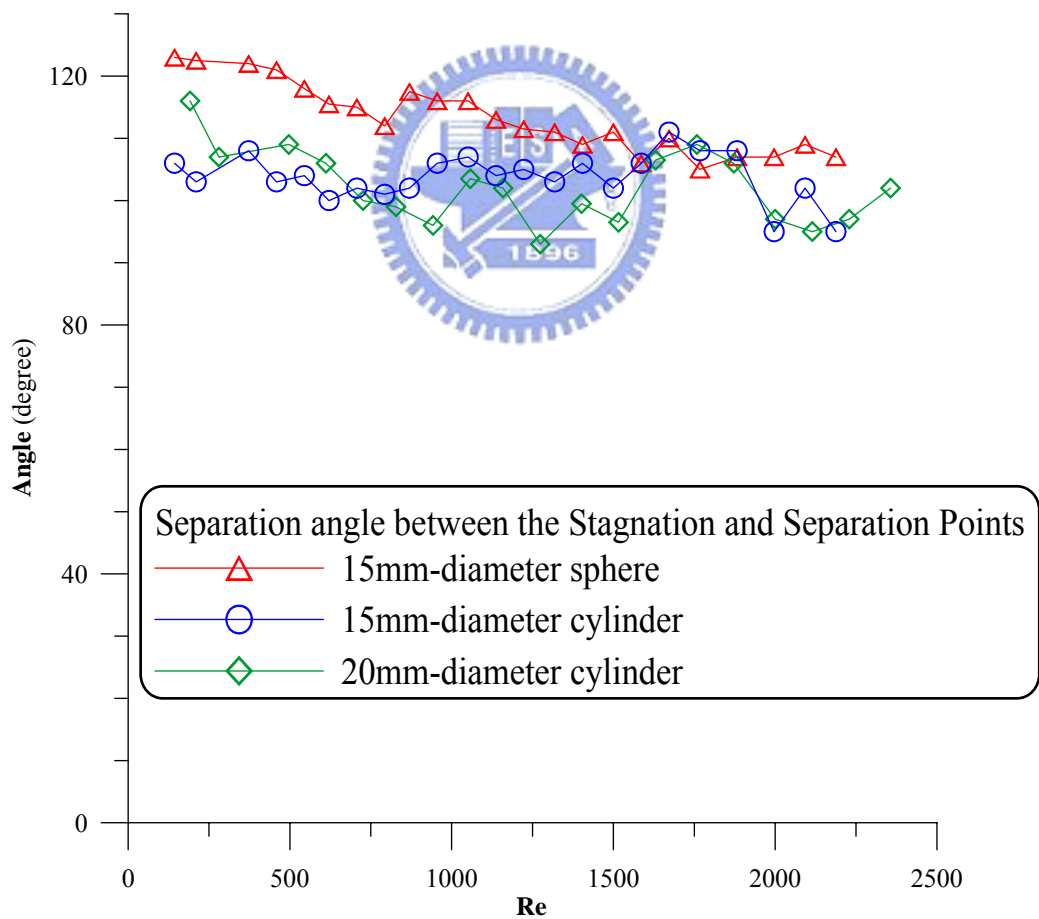
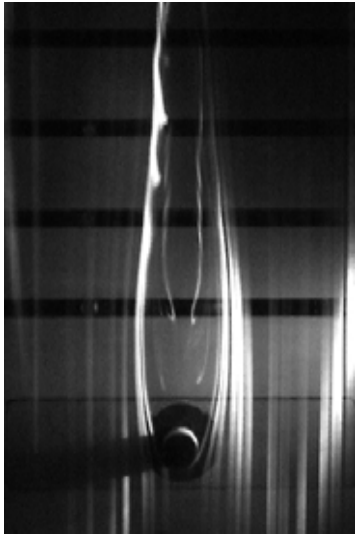
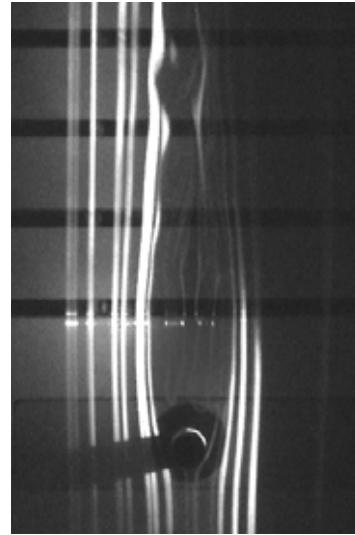


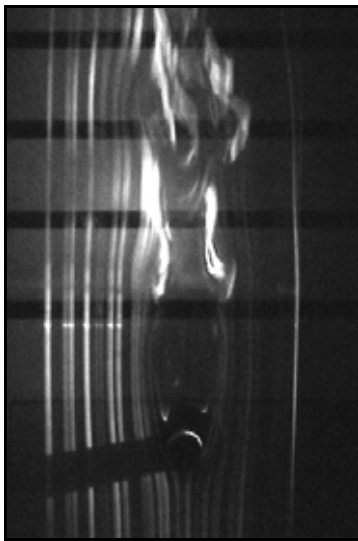
Fig. 4.11 Separation angle θ as function of Re



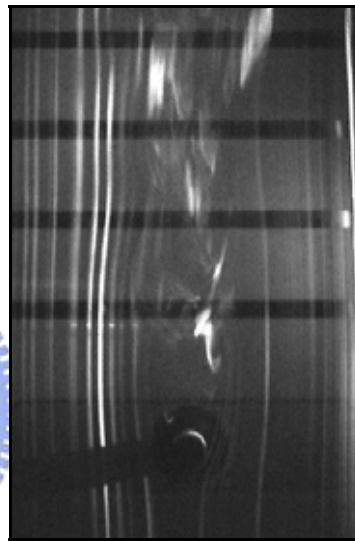
(a) $Re = 143$



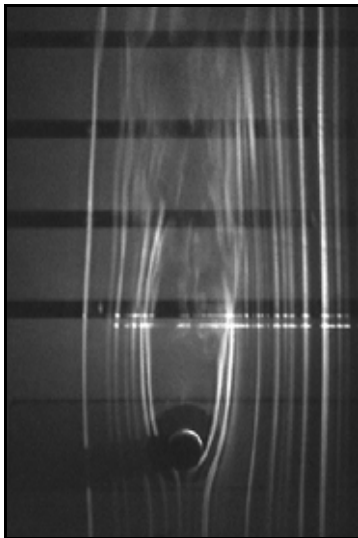
(b) $Re = 210$



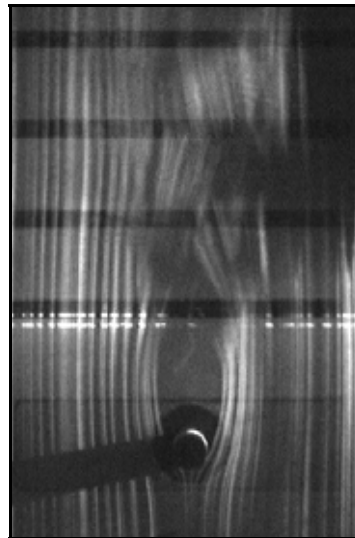
(c) $Re = 372$



(d) $Re = 707$

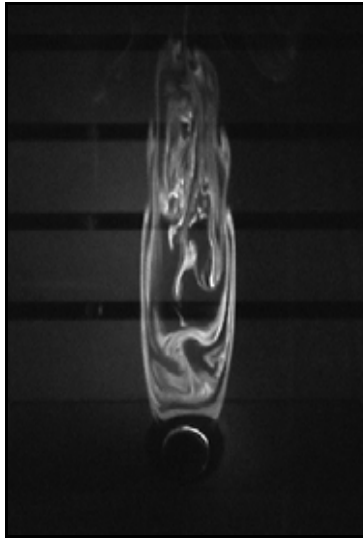


(e) $Re = 907$



(f) $Re = 1,050$

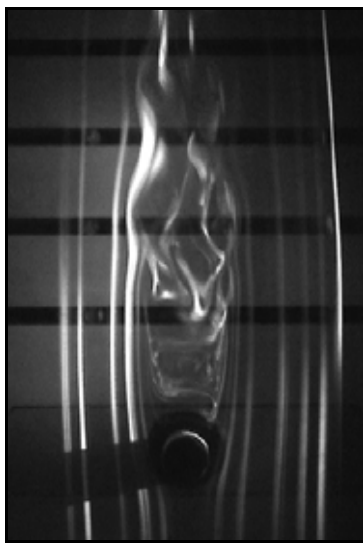
Fig. 4.12 Series of flow behaviors as a function of Reynolds number
(15mm-diameter cylinder, $V_w = 1.76$ cm/s)



(a) $Re = 191$



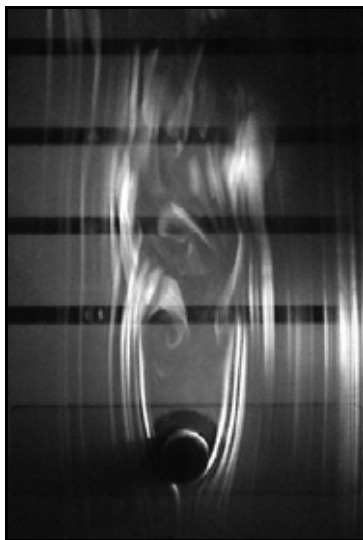
(b) $Re = 280$



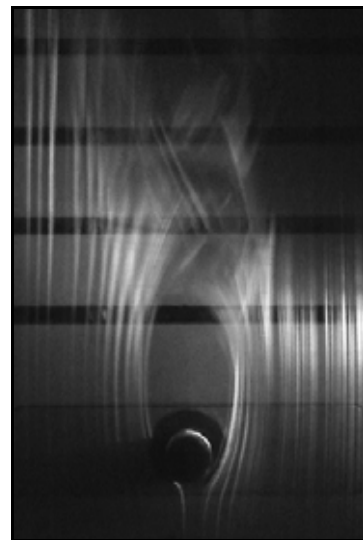
(c) $Re = 496$



(d) $Re = 726$

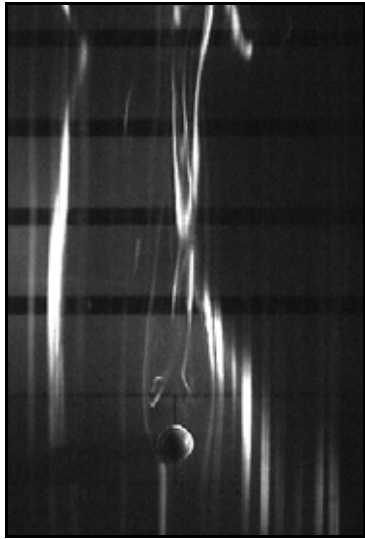


(e) $Re = 1,057$

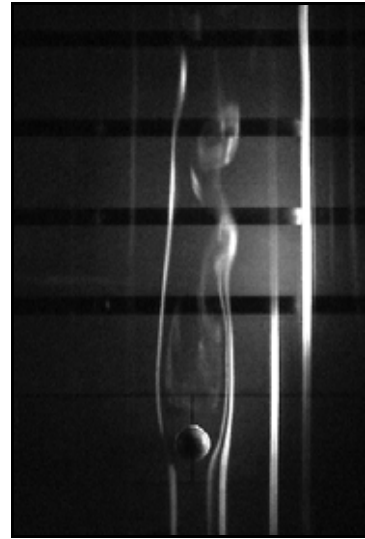


(f) $Re = 2,000$

Fig. 4.13 Series of flow behaviors as a function of Reynolds number
(20mm-diameter cylinder, $V_w = 1.32$ cm/s)



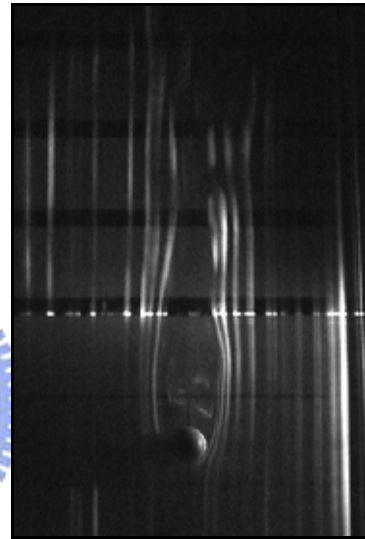
(a) $Re = 143$



(b) $Re = 210$



(c) $Re = 372$



(d) $Re = 458$



(e) $Re = 1,050$



(f) $Re = 1,720$

Fig. 4.14 Series of flow behaviors as a function of Reynolds number (15mm-diameter sphere, $V_w = 4.72$ cm/s)

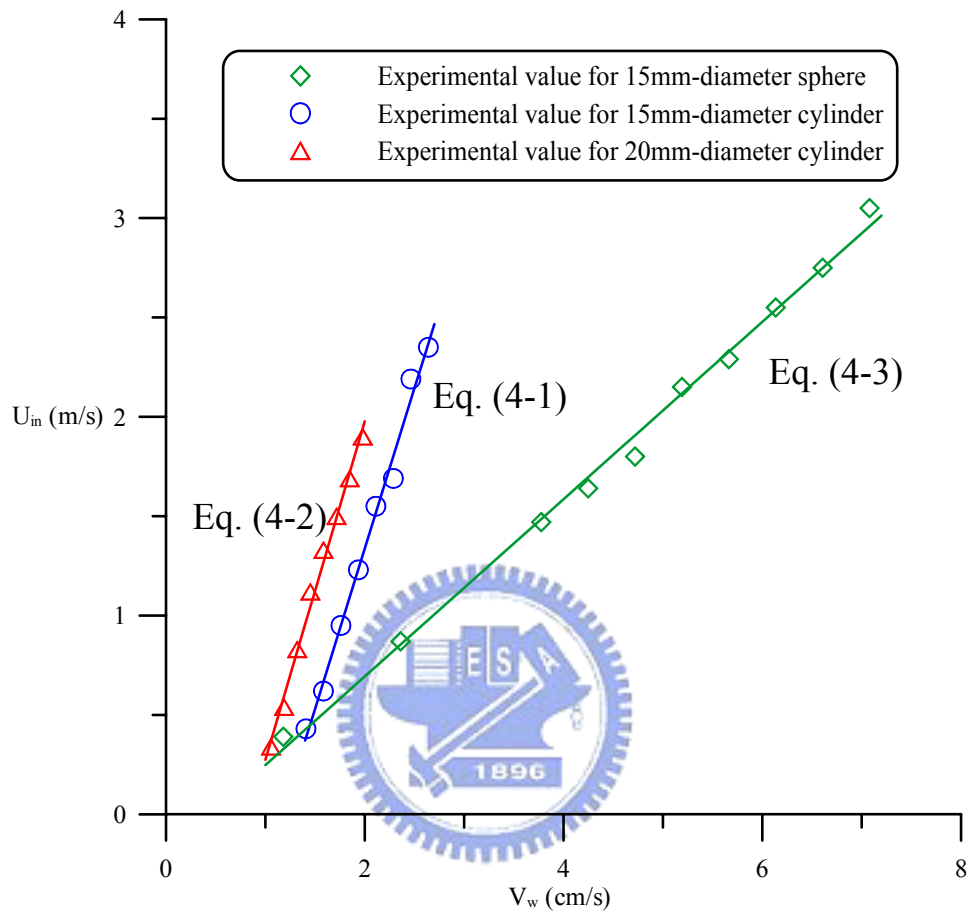


Fig. 4.15 Critical U_{in} as function of V_w

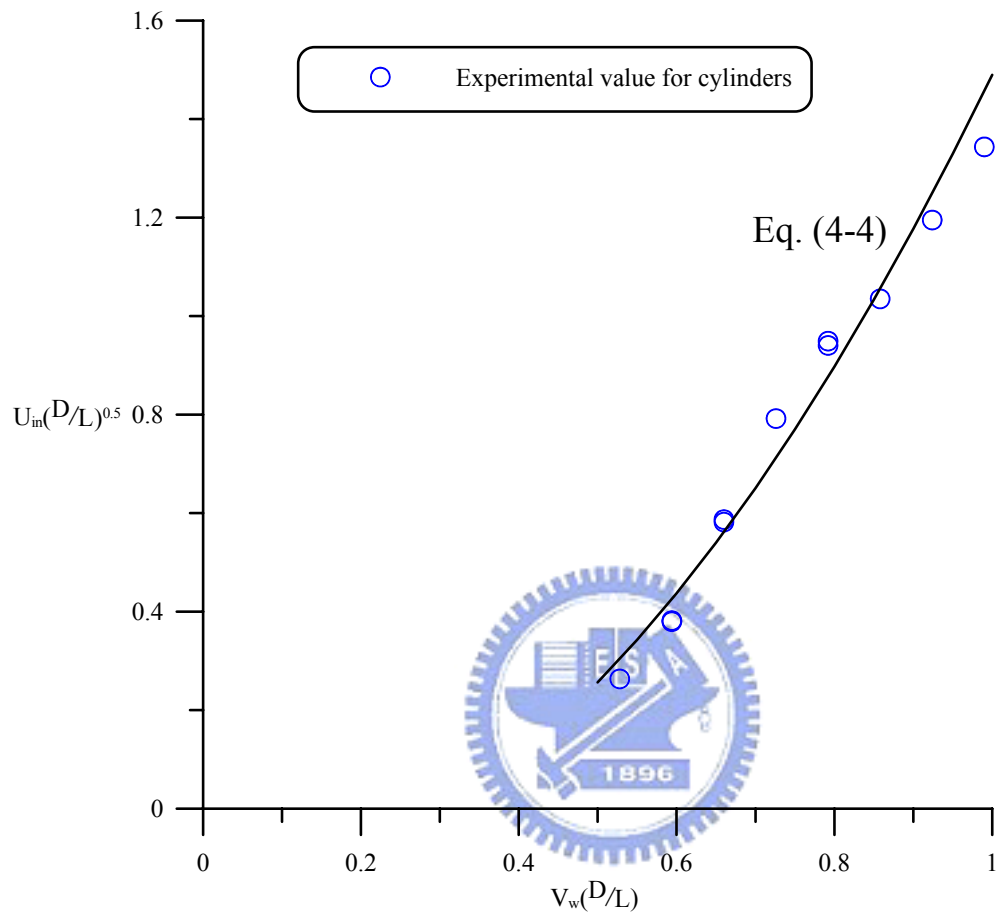


Fig. 4.16 A region map for the wake between blow-off from and return to the rear surface of burner independent of burner's diameter

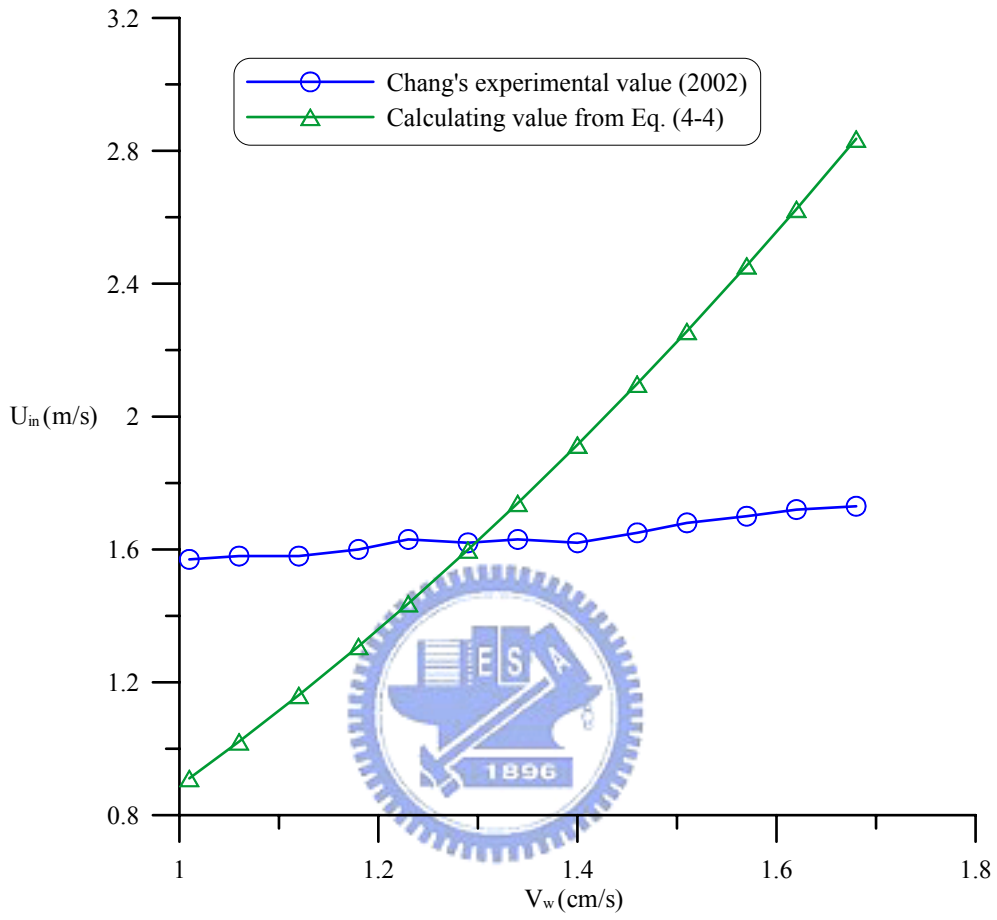


Fig. 4.17 Comparison with Chang's experiment (2002)